Innovative aerodynamic grain separation system for plant harvesting in sloped areas: problems, research and optimization of parameters

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Abstract

This article presents the main problems associated with cereal harvesting in sloping areas. The presented innovative aerodynamic system supporting the separating unit of combine harvester can be one of the ways to counteract the negative effects of harvesting machines work on slopes. The Monte Carlo numerical method, presented in this article, was applied in the optimization of an aerodynamic sieve separation process on an inclined terrain. The given variables are the transverse slope of separator α (of the sieve), longitudinal slope β and the output of the main and side fans. The Monte Carlo method makes it possible to determine an optimized set of parameters ($\alpha = 10^\circ$, $\beta = 2.8^\circ$, $\delta = 9^\circ$), the output of the main fan ($0.67 \text{ m}^3 \text{ s}^{-1}$) and the output of the side fan ($1.86 \text{ m}^3 \text{ s}^{-1}$), allowing to obtain the best indicator values of 2.1% grain loss and 97.5% grain purity.

Keywords Cereal harvesting · Combine harvester · Sloping agricultural land · Aerodynamic separation system · System optimization · Grain loss and grain purity

1 Introduction

Growing cereal crops requires the use of combine harvesters with higher throughput capacity. Given the size limitations of a combine harvester, it is necessary to improve the elements influencing throughput capacity, i.e. for instance sieve and air flow systems [1]. From the exploitation perspective, the throughput parameter is closely connected with the level of general losses and the cleanliness of the obtained grain [2].

The pneumatic system, being one of the cleaning system elements, plays an important role in the grain cleansing process. Grain cleanliness, in turn, affects the economic effects in agriculture (reduction in costs resulting from initial seed cleaning). An appropriately used air stream results in mesh

extraction: short straw, chaff, weed seeds and other light contaminations).

The process of aerodynamic sieve separation is influenced by both the kinematic and construction parameters of a sieve, as well as the aerodynamic properties of the air stream generated by a fan. During the cleaning process of a mixture on a sieve, the thickness of the layer of material is sometimes a few times thicker than the thickness of individual grains [3–5]. Then, an essential condition of mesh pass is mixture stratification, which, however, requires a significant increase in the kinematic indicator of a sieve and the air flow speed. The intensity of sifting is the highest at the initial section of the sieve because fine grain is sifted first as they quickly reach the bottom of a layer. According to numerous works [6, 7], the height of the distribution of cereal layers on the surface of sieve cereal is not equal.

The analysis of literature [8–15] showed that the currently used constructions of cleaning systems in the majority of combine harvesters are equipped with a conventional aerodynamic sieve separation system with a centrifugal fan. According to the research on combine harvesters in mountainous and foothill areas a large majority of losses and worse quality of grain result from the incorrect operation of the cleaning system.

During the operation of a combine harvester on a slope, some difficulties occur in its system functioning due to the

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transverse and longitudinal lean (pitch and roll) of the whole machine. The circular motion of a combine harvester in a mountainous area leads to the situation in which a machine can take four basic positions (Fig. 1: travelling uphill, on a flat area, downhill or across a slope). This results in the speed changes of grain mass on the surface of a sieve and changes in load's mass (Fig. 2).

A very important problem occurring during harvests is the fact that classical aerodynamic sieve systems used in combine harvesters operate properly on a horizontal ground, while in the case of inclinations exceeding 5° their efficiency is significantly decreased.

The exploitation of a combine harvester on inclined areas leads to disadvantageous changes related to the movements

of cereal mass on the surface of sieves with respect to the direction of the resultant gravitational force. In consequence, conventional aerodynamic systems are not able to ensure sufficient mass looseness and blowing away light impurities because of the accumulation of grain, in such cases only the constant conditions of fan operation are maintained. This, among others, results in the decrease in the screening ability of the separation and cleaning system and losses of plump grain.

Due to the above, authors propose to improve the efficiency and effectiveness of grain cleansing in a combine harvester operating in varied terrain configuration by equipping the central airflow with an additional side airflow system, which would direct the airstream to the top louvered sieve. Although

Fig. 1 Operation of a combine harvester on a slope; machine can take four basic positions: travelling uphill, on a flat area, downhill (upper figure) or across a slope (bottom figure). Source: developed based on [6, 16]

Fig. 2 Distribution of grain mass on the sieve surface during operation of the combine harvester in the field: **a** flat, **b** laterally inclined, **c** longitudinally inclined “downhill” ($+\beta$), **d** longitudinally inclined “uphill” ($-\beta$). Where: α —transverse inclination angle of the main sieve plane [$^\circ$], β —longitudinal inclination angle of the main sieve plane [$^\circ$]. Source: developed based on [16]

the solutions currently used in modern combine harvesters improve the screening ability of sieves, simultaneously they significantly increase the cost of such a machine. With regard to this economic aspect, improvements in the construction of combine harvesters must lead to the use of relatively simple and inexpensive solutions. As a rule, the price of combine harvesters grows with the efficiency of the cleaning system in a large range of slopes. Hence, research should be focused on inexpensive construction solutions for cleaning systems allowing to reduce losses during the operation of combine harvesters in the needed range of slopes in each direction.

The new constructional solution of the cleaning system requires the specification of its work parameters. Therefore, the purpose of this article is to determine optimum parameters of the screening system in combine harvester (angle of lateral incline α , the angle of longitudinal incline of terrain β , the angle set for the main fan controls δ , and the outputs of the main fan Q_g and side fan Q_b) in relation to two parameters: purity and losses of grain. Authors propose two approaches to the two-criterion optimization:

- determining the acceptable level of purity of the grain (based on the previously determined regression model) and building the objective function for losses $S(\%) = f(\alpha, \beta, \delta, Q_g, Q_b) \rightarrow \min$ or
- determining the acceptable level of grain losses (based on the previously determined regression model) and building the objective function for grain purity $C(\%) = f(\alpha, \beta, \delta, Q_g, Q_b) \rightarrow \max$.

To achieve the goal, the task of nonlinear programming with inequality constraints was used, and the values of the parameters sought were determined using the Monte Carlo method.

2 Characteristics of the aerodynamic sieve separation process on an inclined terrain. Main problems

The aerodynamic sieve separation process in a grain combine harvester equipped with flat sieves, and working on a hilly terrain is subject to an unfavorable influence from the incline. It is particularly difficult to maintain technological parameters of equilibrium [17, 18]. As the fine grain mass begins to seep, the slope changes the component of the gravitational force with respect to the “downward” direction. The pull causes a downward movement toward the lower side of the sieve of the separator and, in some places, overloads it. On a slope exceeding 10° , the working quality of separation mechanisms (grain combines) equipped mostly with flat lowered sieves significantly deteriorates [1, 6, 19, 20].

A priority for the efficient functioning of a pneumatic sieve is ensuring that the ratio of resultant aerodynamic forces affecting grain mass to unit weight remains constant for the entire working surface of the sieve loaded with the grain mass [1], as expressed in the following equation:

$$\frac{F_{w1}}{G_1} = \frac{F_{w2}}{G_2} = \frac{F_{w3}}{G_3} = \text{const} \quad (1)$$

In practice, the established value remains G , for which the methods of regulating the air stream involve adjusting the aerodynamic force F_w . An increase in the load carried by the sieve should be proportional to the increase in aerodynamic force F_w . The condition under which the output of the fan (or fans) ensures that the aerodynamic force F_w is equivalent to G is called the parameter of applied equilibrium [21, 22]. The greatest purity of grain in aerodynamic sieve separators is obtained by total fluidization of grain mass [23, 24]. Such conditions, considering the present construction of separators, generate losses, which should ideally amount to zero. According to Komarnicki [25], the state of applied equilibrium occurs when the aerodynamic force results in initial fluidization, characterized by bubbling on the surface of the grain mass (Fig. 3). The parameters of the air stream, in which the surface of grain shows a vertical rise of the grain mass (a visible characteristic bubbling), indicate a state of applied equilibrium between cleansing and sifting [26–28].

The longitudinal positioning of grain mass on the moving bed of a sieve separator typical of a grain combine harvester is shown in Fig. 4. These results indicate a linear relationship of height h in the length function of sieve L_s . As expected, the thickness of the grain layer tends to decrease in the direction of its flow [29, 30].

The curves shown in Fig. 5 demonstrate the variability of output Q in the phase of applied equilibrium relative

Fig. 3 Behavior of moving grain mass affected by air stream (red circle: area of the bubble created by initial fluidization)

Fig. 4 Distribution of grain mass on the working surface of a sieve for the main fan, where h =height of the grain layer and L_s =length of sieve. Source: [1]

to the height of the bed, and indicate an initial absence of reaction by the grain layer to the air stream. A 5-mm

Fig. 5 Variability in the applied equilibrium parameter Q versus the height of grain h . MF (main fan) and SF (side fan). Source: [1]

layer revealed localized fluidization only after reaching $1.59 \text{ m}^3 \text{ s}^{-1}$ of output from the main fan (MF) and $1.74 \text{ m}^3 \text{ s}^{-1}$ of output from the side fan (SF).

The preceding discussion indicates that a number of factors influence the process, the most significant of which are the output of the fan and the slope of the working platforms of the sieve [31, 32].

3 Mathematical foundations of simulation and optimization of the separation process

3.1 Description of the Monte Carlo method and formulation of the simulation problem

The Monte Carlo approach is a group of numerical methods, which can simulate physical phenomena [33–38]. A characteristic of the methods described herein is that a string of random or pseudorandom numbers is used for computation [39, 40]. Assuming that F represents the solution to a given problem (real number, string of numbers, binary decision, etc.), then applying the Monte Carlo system:

$$F = f(\{r_1, r_2, \dots, r_n\}; \dots) \quad (2)$$

would produce the resulting value of F where $\{r_1, r_2, \dots, r_n\}$ are random numbers. Despite using random numbers, the problem under examination need not be stochastic. A practical implementation of these simulation methods is contingent upon being able to formulate the problem in such a way that random numbers can be used to solve it [41, 42].

The Monte Carlo methods may be applied to many technical problems. If the problem in question is related to probability or stochastics, the solution would be a direct simulation. Formally speaking, all Monte Carlo calculations constitute integration. If F is the result of the Monte Carlo calculation and at the same time F is a function of random numbers r_i , then.

$$F = F(r_1, r_2, \dots, r_n). \quad (3)$$

Equation (3) is an unbiased estimator of a multidimensional integral

$$I = \int_0^1 \dots \int_0^1 F(x_1, x_2, \dots, x_n) dx_1 dx_2 \dots dx_n \quad (4)$$

or

$$E(F) = I \quad (5)$$

where integral I is the expected value of F .

Let us take at random n numbers u_i with uniform probability within the range (a, b) , and for every u_i calculate the value of function $f(u_i)$. In accordance with the law of large numbers,

$$\frac{1}{n} \sum_{i=1}^n f(u_i) \xrightarrow{n \rightarrow \infty} E(f) = \frac{1}{b-a} \int_a^b f(u) du. \quad (6)$$

If function f is an integrated function, which is always finite and continuous (allowing for a finite number of discontinuities), then the left side of Eq. (6) is the estimator compatible with the integral on the right side. Hence, using the Monte Carlo method of computation provides the correct value if the extent of the sample becomes infinitely large.

The Monte Carlo estimator has the following properties:

- If variance $V(f) < \infty$, then the estimator is compatible (i.e., convergent) with the true value of the integral for large n .
- The estimator is unbiased for all n ; i.e., the expected value of the estimator is the true value of the integral.
- The estimator distribution is asymptotically normal.
- The standard deviation of the estimator may be described as

$$\sigma = \frac{1}{\sqrt{n}} \sqrt{V(f)}. \quad (7)$$

If

$$I \equiv \frac{1}{b-a} \int_a^b f(x) dx = E(f) \quad (8)$$

then the Monte Carlo estimator of the integral in Eq. (8) is.

$$\hat{I} \equiv \frac{1}{n} \sum_{i=1}^n f(x_i), \quad x_i \in U(a, b). \quad (9)$$

If the value of the integral in Eq. (8) is known, then the variance of estimator $\hat{I}$ can be computed from the following equation:

$$V(\hat{I}) = \frac{1}{n} V(f) = \frac{1}{n} \left\{ E(f^2) - [E(f)]^2 \right\} = \frac{1}{n} \left\{ \frac{1}{b-a} \int_a^b f^2(x) dx - I^2 \right\}. \quad (10)$$

In practice, variance is estimated using Eq. (9):

$$\hat{V}(\hat{I}) = \frac{1}{n} \hat{V}(f), \quad \hat{V}(f) = \frac{1}{n-1} \sum_{i=1}^n \left[f(x_i) - \frac{1}{n} \sum_{i=1}^n f(x_i) \right]^2. \quad (11)$$

Therefore, the Monte Carlo estimator of standard deviation appears as:

$$\hat{\sigma} = \sqrt{\hat{V}(\hat{I})} \quad (12)$$

The Monte Carlo method has its advantages and disadvantages. The advantages include:

- gives the opportunity to solve difficult problems;
- a simple form of replacing analytical solutions;
- the growing computing power of computers allows for more and more common use of the method;
- frees the user from complicated theory and formulas, allowing you to focus on the essence of the task.

The disadvantages of the Monte Carlo method include:

- experiments for a finite number of trials;
- the (simulation) results will always be an approximation;
- the results depend on the quality of the pseudorandom number generator.

3.2 Implementation of general simulation methods in optimization of new separation system parameters

Let the given string be $X \subset \mathbb{R}^m$ and function $F: X \rightarrow \mathbb{R}$. Let $x = (x^1, x^2, \dots, x^m) \in X$. Then, the optimization problem, in accordance with the Monte Carlo method, may be a matter of searching for a point for which.

$$x_{opt} \in X : \forall_{x \in X} F(x) \geq F(x_{opt}). \quad (13)$$

Note that a problem formulated in this way differs from the classical search for the minimum of function F in that the search for the minimum value is limited to string X , which is of primary significance in constructing algorithms [34, 35, 43–49].

In practice, many methods of searching for the optimal value may be used. Among the most commonly implemented ones are the “heads-and-tails method,” sequential methods, statistical optimization and genetic algorithms [35, 50–57]. Considering the space limitations of this paper, only the heads-and-tails method and sequential methods will be covered in detail. The computational scheme of the heads-and-tails method is presented as follows:

- Randomly select N points $(x_1, x_2, \dots, x_N) \in X$ in the uniform distribution system P .
- Compute the value of function F for every point, i.e., $F_1 = F(x_1), F_2 = F(x_2), \dots, F_N = F(x_N)$.
- Find $F^* = \min(F_1, F_2, \dots, F_N)$.

The solution to the optimization problem is

$$x_j : F(x_j) = F^*. \quad (14)$$

It can be observed that sequence $\{F_k = F(x_k)\}$, $k = 1, 2, \dots, N$ is one of independent variables in the same order as the cumulative distribution.

$$G(f) = P\{F_j < f\}. \quad (15)$$

If $F^* = \min_{1 \leq j \leq N} F_j$, to $G(F^*)$, we have

$$P\{G(F^*) < \xi\} = P\{F^* < G^{-1}(\xi)\}, \quad 0 < \xi \leq 1. \quad (16)$$

Considering that F^* represents positional statistics from random distribution $(F_1, F_2, \dots, F_N)$, then

$$\begin{aligned} P\{F^* < G^{-1}(\xi)\} &= N \int_{-\infty}^{G^{-1}(\xi)} [1 - G(x)]^{N-1} g(x) dx = \\ &= N \int_0^\xi [1 - u]^{N-1} du = 1 - (1 - \xi)^N, \end{aligned} \quad (17)$$

where g is the density of random variable F_j ; i.e., $g(f) = \frac{dG(f)}{df}$. It can be said that with a probability $1 - (1 - \xi)^N$, point x_{opt} was located exactly with respect to the set whose volume is less than ξ . The smaller the volume of ξ , the better the ability to locate point x_{opt} .

A basic sequential algorithm is presented as follows:

- Establish the point of departure $x_1 \in X$ (e.g., choose from a certain probability distribution of set X).
- If points $x_1, x_2, \dots, x_n$ have already been established, choose an auxiliary point ζ_n in distribution P_n and calculate

$$x_{n+1} = \begin{cases} x_n, & \text{dla } F(x_n + \zeta_n) \geq F(x_n) - \varepsilon \\ x_n + \zeta_n, & \text{dla } F(x_n + \zeta_n) < F(x_n) - \varepsilon \end{cases} \quad (18)$$

where ε is a positive constant. In addition, assume that $\{\zeta_n\}$ is an array of independent variables and that $(x_n + \zeta_n) \in X$ [7].

Using the preceding algorithm, a string of points $\{x_n\}$ is obtained so that.

$$F(x_1) \geq F(x_2) \geq F(x_3) \geq \dots \geq F(x_n) \geq F(x_{n+1}) \geq \dots \quad (19)$$

If function F is limited from the bottom, this string is convergent. If in string $\{x_n\}$ there appears point x' for which $F(x_{opt}) < F(x') < F(x_{opt}) + \varepsilon$, then in accordance with the preceding algorithm it would be repeated indefinitely; i.e., $\lim_{k \rightarrow \infty} x_k = x'$. Note that changing the value of parameter ε makes it possible to change the value of x' .

If

$$A_\varepsilon = \{x : F(x) < F(x_{opt}) + \varepsilon\} \quad (20)$$

then string $\{x_n\}$ is convergent with x_{opt} if it is convergent with any point from set A_ε .

In our article, to optimize the separator operation parameters, we use two criteria: grain loss and grain purity. These criteria are not used separately. The optimal solution in the Pareto sense is called the $x' \in D$ solution, in that there is no other solution $x \in D$ that gives an improvement in the value of one or more objective functions, which do not cause deterioration of other objective functions. The optimal solution in the Pareto sense is also called efficient or effective solution. If in our case the main criterion was the grain loss ($S = f(x_1, x_2, x_3, x_4, x_5) \rightarrow \min$), then the secondary criterion in the form of grain purity was included in the search set as a limitation (inequality: $C \geq 92.5$). In the second case, the opposite task was carried out (and we have replaced the roles of these criteria), and the solutions were verified. This is a well-known approach in the literature often referred to as the “main and secondary criterion” method. It is used when for a decision maker one criterion is basic (main) and the other is less important (secondary). The best solution is then sought for the main criterion, while ensuring a certain level of the secondary criteria implementation. Setting a compromise decision boils down to solving the task:

$$\begin{cases} f_1(x) \rightarrow \min \text{ (or max)} \\ f_k(x) \geq p_k, \text{ for } k = 2, \dots, s \\ x \in D \end{cases} \quad (21)$$

where f_1 —main criterion and p_k —satisfactory level of implementation of the k -th secondary criterion.

So to sum up: these criteria were not considered separately, and our work is completely in line with the commonly accepted methodology.

4 Material and methods

The starting material for research was a grain mixture taken at random from a chaff riddle in a combine harvester. The mixture contained *Muza* wheat and natural contaminants (chaff, waste product, straw, ears, weed seeds, etc.). Before each experiment, the mass was stirred so as to obtain a structurally homogenous mixture of material cleansed in a combine harvester. Bench testing was conducted for selected combinations of setups and parameters of the cleansing system ($\alpha, \beta, \delta, Q_g, Q_b$).

The tests were conducted on a simulation station designed and made in the Institute of Agricultural Engineering of the Wrocław University of Environmental and Life Sciences, a description of the station can be found in [6]. The station

Fig. 6 Laboratory tests station—aerodynamic system: 1—main fan, 2, 5—reduction (transition from a rectangular to circular section), 3—PCT throttle valve, 4—flexible cord, 6—nozzle directing the airflow, 7—right-side fan, 8—left-side fan, 9—louvered sieve

was modernized by the introduction of a model aerodynamic system (Fig. 6). The aerodynamic sieve system was composed of a conventional, central airflow and additional side aerodynamic systems directing the airstream to the top louvered sieve.

There were the following fans in the installation: 1—main fan being $3.05 \text{ m}^3 \text{ s}^{-1}$ capacity and two side fans 7 and 8— $2.24 \text{ m}^3 \cdot \text{s}^{-1}$ capacity. A change in the medium flow rate was obtained by the introduction of throttling elements in the form of multiblade dampers 3. A characteristic feature of dampers is the fact the airflow rate is decreased proportionally with its openness degree. The directed introduction of airflows under the surface of the louvered sieve was possible thanks to steering nozzles. The tests were conducted for the variable condition of fans operation:

- angle of steering gear in nozzles $\delta \in \langle +9^\circ; +20^\circ \rangle$,
- main air flow (MF) $Q_g \in \langle 0.67 \text{ m}^3 \text{ s}^{-1}; 1.90 \text{ m}^3 \text{ s}^{-1} \rangle$,
- side air flow (SF) $Q_b \in \langle 0 \text{ m}^3 \text{ s}^{-1}; 1.95 \text{ m}^3 \text{ s}^{-1} \rangle$.

The operation quality of the aerodynamic sieve system was assessed on the basis of the grain loss indicator. A screen with a chute was mounted on the research object to collect the mass of plump wheat grain falling outside the sieve basket. In the assessment of sieve operation, grain separation losses S [%] were determined according to the following dependence:

$$S = \frac{m_{zo}}{m_w} \cdot 100 \text{ [%]} \quad (22)$$

where m_{zo} —grain mass in waste [kg],
 m_w —starting material mass [kg].

After starting the operation of the tilt test station, under the influence of the airstream the cereal mass was falling into measurement containers distributed under the sieve basket. The content of the containers was weighed, and the next waste mass was separated. Grain cleanliness C was assumed to the percentage of seeds in base material per average sample of starting material, fraction or product:

$$C = \frac{m_w - m_z}{m_w} \cdot 100 \text{ [%]} \quad (23)$$

where m_w —average starting material mass [kg],

m_z —average mass of impurities in the product [kg].

During the determination of grain cleanliness indicators, the following sieve system operation conditions were maintained:

- sieve operating slit $s = 7 \text{ mm}$ (top sieve—wheat),
- lateral inclination (roll rate) of the sieve basket $\alpha \in \langle 0^\circ; +10^\circ \rangle$
- longitudinal inclination (pitch angle) of the sieve basket $\beta \in \langle -15^\circ; +5^\circ \rangle$
- sender of grain mass 55 kg ,
- grain supply to the sieve 3.5 kg s^{-1} ,
- separation time t was 15 s .

4.1 Experimental verification of the proposed optimization solution: results and discussion

Monte Carlo method can be implemented to establish an optimal set of technological parameters for a separator functioning on inclined terrain. So as to achieve this, the relationship between grain purity and the output of the main fan, the output of the side fan and the angle of lateral and longitudinal incline may be approximated, with sufficient accuracy, by a second-degree polynomial as follows:

$$C = 96.46 - 0.203\alpha - 0.1\beta - 4.86Q_b + 0.724Q_g^2 + 2.898Q_b^2 \quad (24)$$

$$R^2 = 0.95$$

Figure 5 shows the distribution of indicators of grain mass purity C relative to the incline of terrain α, β (simulation of working across the field and uphill) and the output of the main fan Q_g and side fan Q_b . The effect produced by the angle of the terrain α, β to decrease the purity indicators is shown in Fig. 7a, b. Figure 7a clearly points to the negative effect of a lateral slope of sieve plane α on grain purity C . The movement of the grain mass across the sieve resulted in the deterioration of purity indicators to 92.5% with $\alpha = 10^\circ$. This condition was improved by increasing the output of the side fan Q_b to $1.95 \text{ m}^3 \text{ s}^{-1}$ at which point the maximum purity grew to 95.5%. The best result for the working quality

Fig. 7 Distribution of grain purity indicators C with respect to **a** output of the side fan Q_b (SF) and angle α ; **b** output of the side fan Q_b (SF) and angle β ; **c** output of the side fan Q_b (SF) and the main fan Q_g (MF)

of the aerodynamic sieve method, 97.5%, was obtained when the sieve was on a transverse plane at $\alpha = 0^\circ$ and with maximum output Q_b . Changing the angle of the incline of the terrain β from 5° to -15° resulted in an increase in purity indicators to 97.9% (Fig. 7b).

The lowest grain purity of 92.6% was obtained in the simulation of the combine harvester working on a downward slope ($\beta = 5^\circ$). It improved only when the output of the side fan Q_b was increased from 0.8 to $1.95 \text{ m}^3 \text{ s}^{-1}$; then, the maximum purity reached 97.5%. The effect of two fans functioning on the constant incline ($\alpha = 5^\circ$, $\beta = -5^\circ$) is shown in Fig. 7c. The highest values of purity indicators (exceeding 99%) were obtained with the maximum output of the fans at the levels of $1.9 \text{ m}^3 \text{ s}^{-1}$ for the main fan and $1.95 \text{ m}^3 \text{ s}^{-1}$ for the side fan.

The amount of loss is affected by the angle of lateral incline α , the angle of longitudinal incline of terrain β , the angle set for the main fan controls δ , and the outputs of the main fan Q_g and side fan Q_b . The matrix of coefficients of

correlation between losses S and the analyzed independent factors is presented in Table 1.

The model used independent variables, which have a strong correlation with the losses and a weak correlation with each other (Table 2). Based on the results of the research tests, a mathematical model of the correlation of the amounts of loss S (%) from the previously analyzed factors was constructed. As a result of a statistical analysis, the parameters of the model for losses were estimated to be:

$$S = -1.58 + 2.602Q_g + 0.013\beta^2 - 0.141\beta Q_g + 1.697Q_b Q_g. \quad R^2 = 0.80 \quad (25)$$

The formulas presented in Eqs. (24 and 25) comply with the variability:

$$\begin{aligned} x_1 &= \alpha \in \langle 0^\circ; +10^\circ \rangle \\ x_2 &= \beta \in \langle -15^\circ; +5^\circ \rangle \\ x_3 &= \delta \in \langle +9^\circ; +20^\circ \rangle \\ x_4 &= Q_g \in \langle 0.67 \text{ m}^3 \text{ s}^{-1}; 1.90 \text{ m}^3 \text{ s}^{-1} \rangle \end{aligned}$$

Table 1 Matrix of coefficients of correlation between losses S and analyzed independent factors (statistically significant values are **bold** marked)

	α	β	δ	Q_b	Q_g	α^2	$\alpha\beta$	$\alpha\delta$	αQ_b	αQ_g	β^2	$\beta\delta$	βQ_b	βQ_g	δ^2	δQ_g	Q_b^2	$Q_b Q_g$	Q_g^2
S	0.045	-0.509	0.070	0.243	0.381	0.045	-0.045	0.068	0.244	0.126	0.495	-0.503	-0.244	-0.667	0.070	0.335	0.290	0.271	0.384

$$x_5 = Q_b \in \langle 0 \text{ m}^3 \text{ s}^{-1}; +1,95 \text{ m}^3 \text{ s}^{-1} \rangle.$$

Quality optimization of the purification process involves nonlinear programming with inequality limits:

- $0 \leq x_1 \leq 10$
- $-15 \leq x_2 \leq +5$
- $+9 \leq x_3 \leq +20$
- $0,667 \leq x_4 \leq 1,90$
- $0 \leq x_5 \leq 1,95$
- $96,46 - 0,203\alpha - 0,1\beta - 4,86Q_b + 0,724Q_g^2 + 2,898Q_b^2 \geq 92,5$

as well as the goal function $S = f(x_1, x_2, x_3, x_4, x_5) \rightarrow \min$.

An application of the Monte Carlo method determined the values of the searched parameters (α , β , δ , Q_g , and Q_b), which account for all the limits (with the inclusion of grain purity) and produce minimum losses. For such a problem of nonlinear optimization, the solution is a vector:

$\alpha = 10,0^\circ$ (the value of the maximum lateral inclination was assumed)

$$\beta = +2,8^\circ$$

$\delta = 9,0^\circ$ (the minimum value of the angle of main fan's blades was assumed)

$$Q_g = 0,67 \text{ m}^3 \text{ s}^{-1}$$

$$Q_b = 1,86 \text{ m}^3 \text{ s}^{-1}$$

$$C = 97,44\%.$$

Then, losses amount to $S = 2,12\%$.

If the angle of area inclination would change to -5° , despite the optimization of the remaining parameters, the grain loss will exceed the admissible value (Fig. 8):

$\alpha = 10,0^\circ$ (the value of the maximum lateral inclination was assumed)

$$\beta = +5,0^\circ$$

$\delta = 9,0^\circ$ (the minimum value of the angle of main fan's blades was assumed)

$$Q_g = 0,96 \text{ m}^3 \text{ s}^{-1}$$

$$Q_b = 1,95 \text{ m}^3 \text{ s}^{-1}$$

$$C = 97,14\%$$

$$S = 3,75\%.$$

If the criterion of optimization is to maximize grain purity, we assume that the maximum losses will be:

$$-1,58 + 2,602Q_g + 0,013\beta^2 - 0,141\beta Q_g + 1,697Q_b Q_g \leq 2,5 \quad (26)$$

for $\delta = 9^\circ$ (the minimum value of the angle of main fan's blades was assumed). Other restrictions are as in the above task. Goal function: $C = f(x_1, x_2, x_3, x_4, x_5) \rightarrow \max$.

For such a problem of nonlinear optimization, the solution is a vector:

$$\alpha = 0,0^\circ$$

$$\beta = -1,2^\circ$$

$$\delta = 9,0^\circ$$

$$Q_g = 0,667 \text{ m}^3 \text{ s}^{-1}$$

$$Q_b = 1,95 \text{ m}^3 \text{ s}^{-1}$$

Table 2 Matrix of interaction of the analyzed independent factors (statistically significant values are **bold** marked)

	β	Q_b	Q_g	αQ_b	β^2	$\beta \delta$	βQ_b	βQ_g	δQ_g	Q_b^2	$Q_b Q_g$	Q_g^2
β		0.000	0.000	0.000	-0.806	0.940	0.000	0.914	0.000	0.000	0.000	0.000
Q_b	0.000		-0.191	1.000	-0.252	0.072	-1.000	0.048	-0.252	0.979	0.990	-0.240
Q_g	0.000	-0.191		-0.190	0.051	-0.015	-0.191	-0.253	0.673	-0.177	-0.166	0.990
αQ_b	0.000	1.000	-0.190		-0.252	0.072	-1.000	0.048	-0.252	0.979	0.990	-0.240
β^2	-0.806	-0.252	0.051	-0.252		-0.700	0.252	-0.750	-0.093	-0.235	-0.250	0.064
$\beta \delta$	0.940	0.072	-0.015	0.072	-0.700		-0.072	0.864	-0.248	0.067	0.071	-0.018
βQ_b	0.000	-1.000	-0.191	-1.000	0.252	-0.072		-0.048	0.252	-0.979	-0.990	0.240
βQ_g	0.914	0.048	-0.253	0.048	-0.750	0.864	-0.048		-0.170	0.040	0.041	-0.251
βQ_g	0.000	-0.252	0.673	-0.252	-0.093	-0.248	0.252	-0.170		-0.234	-0.237	0.673
Q_b^2	0.000	0.979	-0.177	0.979	-0.235	0.067	-0.979	0.040	-0.234		0.970	-0.223
$Q_b Q_g$	0.000	0.990	-0.166	0.990	-0.250	0.071	-0.990	0.041	-0.237	0.970		-0.219
Q_g^2	0.000	-0.240	0.990	-0.240	0.064	-0.018	0.240	-0.251	0.673	-0.223	-0.219

Fig. 8 Distribution of grain loss indicators with respect to the output Q_b of the side fan (SF) and the longitudinal angle of incline of terrain β

$S = 2.5\%$.

Then, purity $C = 98.45\%$.

The obtained result was not satisfactory for the authors. This result applies mainly to running the machine on a flat surface ($\alpha = 0^\circ$) with a slight lateral inclination ($\beta = 1.2^\circ$). Authors in the final conclusions will remain with the results of the first optimization task (objective function: minimum grain losses, and reduce of grain purity). It is better to keep more grains even at the expense of increasing pollution—this was the premise. An increase in the output of the main fan caused an increase in purity and an increase in losses, with the losses growing intensely, as shown in Fig. 9.

The optimization of the values of purity and loss with respect to the output of the side fan (Fig. 10) demonstrated

Fig. 9 Variability of optimized purity indicators C and grain loss S with output of the main fan (MF) **Fig. 10** Distribution of optimized purity indicators C and grain loss S with the output of side fan (SF)

that an increase results in the improvement of the purity indicators and in the deterioration of the indicators of grain loss. The level of the purity indicators decreases from 0 to $0.7 \text{ m}^3 \text{ s}^{-1}$, after which it starts to increase, reaching the maximum value of 96.1% (as seen in Fig. 8 which demonstrates these changes for constant values $\alpha = 10.0^\circ$; $\beta = -1.2^\circ$; $\delta = 9.0^\circ$ and $Q_g = 0.66 \text{ m}^3 \text{ s}^{-1}$). However, as we can see, for the maximum side fan output, the level of grain losses did not exceed the permissible 2.5% value. All optimal values of the cleaning unit operating parameters have been verified on the laboratory test station (Fig. 6) at significance level $\alpha = 0.05$ (no significant differences between the predicted and obtained values).

5 Conclusions

1. Multifactor regression analysis made it possible to determine mathematical models describing quantitative effects—the loss of grain S and qualitative purity C of the separation process of the grain,

$$S \text{ and } C = f(\alpha, \beta, \delta, Q_g, Q_b),$$

allow for the selection of optimal fan output (Q_g, Q_b) for the stipulated angles of incline of field surface α, β and positioning of the steering mechanisms δ so that.

$$\Sigma C \geq 97.5\%$$

$$\Sigma S \leq 2.5\%.$$

2. Applying Monte Carlo methods enabled the determination of a set of optimal technological parameters for a separator, the implementation of which should ensure normative loss indicators of 2.1% and 97.5% purity; angles of $\alpha = 10.0^\circ$, $\beta = +2.8^\circ$, and $\delta = 9.0^\circ$; main fan output of $0.67 \text{ m}^3 \text{ s}^{-1}$ and side fan output of $1.86 \text{ m}^3 \text{ s}^{-1}$.

In conclusion: on the one hand, the novelty of this article is the use of the presented modernized separation system, whose separating action on the sieve is supported by additional air streams. On the other hand, we showed in the article how we can determine the optimal parameter values in a relatively simple way and thus the best working conditions of this new separating system. The same could be achieved by solving systems of differential equations. Only what for? The Monte Carlo method allows you to "shoot" random values in a given space of variables D (a huge number of values), and this allows you to designate a counter-domain for the indicated criteria. A set of values that meets the set criteria is also a collection of optimal solutions. Is not that simpler? Through this article, we show that in this difficult task, this method can be useful with great success. This approach allows us to achieve the best operating conditions of the separation system under changing machine operating conditions (change of angles: α and β).

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Compliance with ethical standards

Ethical approval This article does not contain any studies with human participants or animals performed by any of the authors.

Conflict of interest Jerzy Detyna declares that he has no conflict of interest. Jerzy Bieniek declares that he has no conflict of interest. Piotr Komarnicki declares that he has no conflict of interest.

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