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Conceptual design and comparison of hybrid electric propulsion systems for small aircraft

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Abstract

This paper presents a novel conceptual design method for electric and hybrid electric propulsion systems in small aircraft. The effects of key design parameters on the propulsion system performance are analyzed and the advantages and drawbacks of the investigated propulsion systems are discussed on the basis of two sets of thrust requirements. First, the general conceptual design algorithm is outlined. This is followed by a description of the three propulsion systems investigated: the fully electric; the parallel hybrid; and the conventional internal combustion engine. Scalable models of all required propulsion system components are presented, including weight estimation and operating characteristics. Afterwards, the conceptual design algorithm is exemplified for a reference two-seater motorized glider with a cruising speed of 140 kt and a maximum take-off mass of 1000 kg. Key design parameters are identified and their impact on propulsion system mass and cruise efficiency discussed. This study suggests that the parallel hybrid propulsion system is advantage of 12% or a maximum system mass advantage of 10% can be expected, depending on the propeller design. For the chosen cruise range of 300 km, the system mass of the fully electric propulsion system is at least 2.37 times higher when compared to the conventional propulsion system. In summary, a design method for hybrid electric propulsion systems is presented here which may be used for conceptual design. Furthermore, the suitability of the propulsion systems under investigation for different sets of thrust requirements is assessed, which may be helpful for aircraft designers.

Keywords Hybrid · Electric · Propulsion · Conceptual design · General aviation

List of symbols S Slot mech Mechanical Abbreviations р p Axis EM Electric motor q Axis q EPS Electric propulsion system Maximum max IC Internal combustion engine Minimum min IPS Internal combustion engine prop. system Time t MTOM Maximum take-off mass Prop Propeller PHPS Parallel hybrid propulsion system DC Direct current Indices TO Take-off δ Air gap **Symbols** S Phase Tip mach-number $M_{\rm Tip}$ D Diameter T_c 🖂 Jo Köhler Power Electronic casing temp. j.koehler@ist.rwth-aachen.de Effective mean pressure $p_{\rm me}$ Р Power Peter Jeschke jeschke@ist.rwth-aachen.de Discharge volume V_D Ν Rotational speed 1 Institute of Jet Propulsion and Turbomachinery, RWTH Mean piston speed C_m Aachen University, Templergraben 55, 52062 Aachen, Combustion engine stroke Germany S

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b	Combustion engine bore
z	Number of combustion engine cylinders
S	Apparent power
k_w	Winding factor
Α	Electric loading
\hat{B}	Peak magnetic flux density
В	Magnetic flux density
l	Length
Ι	Current
U	Voltage
η	Efficiency
$\cos(\varphi)$	Power factor
Trq	Torque
A_{S}	Slot area
$ au_S$	Slot pitch
φ_S	Copper fill factor
$arphi_{ m Fe}$	Iron fill factor
b_t	Tooth width
ψ_p	Flux linkage
ω	Electric angular speed
С	C-rate
$f_{\rm AF}$	Activity factor
$c_{\rm DL}$	Design lift coefficient
i	Gear ratio
р	Number of pole Pairs
H_P	Degree of hybridisation
т	Mass
J	Current density
SOC	State of charge

1 Introduction

In recent times, interest in electric and hybrid electric propulsion concepts has increased significantly. These new concepts have been investigated for all aviation sectors, from twin aisle aircraft to urban air transport with vertical take-off and landing capabilities. Due to the new propulsion system components, the conceptual design methods for conventional aircraft propulsion systems are no longer applicable. In order to make an adequate evaluation of the advantages and disadvantages of electric and hybrid electric propulsion concepts, a consistent conceptual design method is necessary.

Previous studies have shown that electric and hybrid electric propulsion systems using today's technology are most appropriate for small aircraft, the main reason being the low battery energy and power density. Battery mass mainly depends on two factors. First, it is highly dependent on the operating strategy, as stated by Juretzkok et al. [16]. Secondly, it depends on absolute power demand. A lower power demand accumulates less energy over a specific time period, so a smaller battery can be used. As such, small aircraft will suffer fewer drawbacks in respect of battery mass. The study conducted in this paper will therefore concentrate on the small aircraft sector in the range of 1000-2000 kg maximum take-off mass (MTOM).

For this class of aircraft several scientific and industrial development projects involving hybrid electric propulsion systems are currently being advanced. Among them are the NASA X-57 project [10], the Silent Air Taxi [9] and Ampaire [1], to name but a few. These projects have gone beyond the conceptual design phase, which only goes to highlight the relevance of hybrid electric propulsion in this class of aircraft.

On a conceptual design level, several studies have already been conducted for small aircraft. Kreimeier and Stumpf [18] compared series and parallel hybrid electric configurations with a baseline internal combustion engine propulsion system. The propulsion system was modeled with constant efficiencies and power-to-mass ratios and the aircraft was operated with a constant energy split between conventional hydrocarbon energy and electric energy of 20%. The cruise range was varied and propulsion system mass and operating costs were investigated. The results showed that hybrid electric propulsion systems are much heavier when compared to the conventional baseline, especially for high cruise ranges. Strathoff et al. [31] conducted a similar study, reaching the same conclusion. In this study, however, the propulsion system was modeled in more detail, including operating characteristics of the propulsion system components. In all of the studies mentioned, the thrust ratios between different operating points, e.g. cruise and take-off, were held constant. Nevertheless, one propulsion system may be more effective for one specific set of thrust ratios than the other. For a reasonable comparison this parameter should therefore be varied, to identify possible fields of application for each propulsion system.

Rings et al. [25] varied the thrust ratios and compared a parallel hybrid propulsion system to a conventional combustion engine baseline. The electric system was utilized as a boost during take-off and constant efficiencies and power densities were employed. This study indicates no mass difference between the two propulsion systems, independent of the mission and thrust requirements. Additionally, a fuel advantage for the parallel hybrid system was identified, stemming from a changed aircraft design.

All of these studies are based on the aircraft level. Other than varying the power split between the electric and hydrocarbon energy sources, the design space for the propulsion systems of small aircraft incorporating a internal combustion engine has so far not been investigated. Like in the small aircraft sector, most available studies for bigger aircraft (MTOM > 2000 kg) are also based on the aircraft level. Some publications examine the propulsion system in detail, varying key design parameters (e.g. [11, 33]). Propulsion systems for this type of aircraft incorporate turboprop

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engines or turbofans instead of internal combustion engines and propellers, which results in a different operating behavior and design space altogether.

The authors' aim here is to present a conceptual design method for electric and hybrid electric propulsion systems for small aircraft incorporating combustion engines and propellers. Furthermore, the impact of key design parameters of the propulsion system on mass and cruise efficiency are discussed, by applying the conceptual design method to an exemplary use case. Specifically a fully electric and a parallel hybrid propulsion system are chosen for the exemplary study, to highlight the different design trades present for different kind of propulsion systems. For the hybrid electric propulsion system, the parallel hybrid was chosen. In contrast to the series hybrid propulsion system, the parallel hybrid propulsion system does not require the consideration of aerodynamic integration effects (which are out of scope for this paper) to yield advantages over the conventional propulsion system assuming current technology [31]. The investigated propulsion system concepts are compared to a conventional reference propulsion system with respect to two sets of thrust requirements in order to highlight the advantages and drawbacks of each propulsion system respectively.

This paper is structured as follows: First, the methodology used in this paper is detailed. Chapter 3 gives a presentation of the employed component models and propulsion systems under investigation. In chapter 4, a design space exploration is conducted, highlighting the impact of the main design variables on each propulsion system. Finally in chapter 5, the thrust requirements are altered and the propulsion systems compared in terms of overall mass and cruise efficiency for each set of thrust requirements. With this approach, favorable thrust requirements for the investigated propulsion systems are identified.

2 Methodology

The conceptual design of an aircraft propulsion system is primarily determined by the thrust requirements and additional boundary conditions, e.g. ambient conditions and maximum propeller diameter. To identify areas of application which will benefit from the propulsion systems under investigation, scalable thrust requirements, as well as a consistent conceptual design method are needed.

2.1 Definition of thrust requirements

To design the propulsion systems a set of thrust requirements is necessary, which was selected arbitrarily. These requirements are defined by an aircraft design constraint analysis according to Mattingly et al. [21]. The thrust requirements can be estimated by solving the equations of motion for an exemplary aircraft, assuming a drag polar. Thrust requirement scaling is achieved by varying either the wing loading, or the maximum take-off mass of the aircraft. This will change the required thrust ratios between different mission points or the absolute thrust values respectively. This approach ensures that the scaling of thrust ratios is limited to a realistic range. At the propulsion system design stage, it is assumed that a heavier propulsion system will lead to a reduced payload, leaving the MTOM and thrust requirements unchanged.

This study evaluates only those constraints which are most relevant for the propulsion system. The aircraft requirements and assumptions on the drag polar are summarized in Table 1. The parameters of the drag polar have been chosen to resemble a motorized glider. A motorized glider type aircraft was chosen, to facilitate the application of a fully electric propulsion system. This type of aircraft typically requires lower propulsion power and hence less energy, minimizing the battery mass. As will be shown later, the battery mass poses a serious disadvantage for fully electric propulsion systems.

Figure 1 shows the constraint analysis used to generate the thrust requirements. For simplicity only standard day conditions are considered. Two different wing loadings with low and high thrust ratios between take-off and cruise are designated. In chapter 4 the design algorithm is exemplified for the thrust requirements with a high thrust ratio and in chapter 5 the propulsion systems are compared for each set of thrust requirements. The required thrust loading for each mission segment can be evaluated at the intersection of the dotted black line with the mission segment constraint.

Table 1 Assumed requirements and drag polar parameters

Requirements		
Cruising speed	140	kt
Cruising altitude	10000	ft
Stall speed	51	kt
Take-off ground run	690	ft
Initial climb rate	14.7	ft/s
Speed at initial climb	62	kt
Climb rate at mid-climb	9.8	ft/s
Speed at mid-climb	70	kt
Drag polar parameters		
Oswald factor	0.9	-
Aspect ratio	31.0	-
Minimum drag coefficient	0.009	_
Lift coefficient at minimum drag	0.1	-
Maximum lift coefficient	1.8	-



Fig. 1 Constraint analysis with thrust requirements used in this study

2.2 Propulsion system conceptual design algorithm

Inspired by the GasTurb software [12], the conceptual design of the propulsion system is divided into the design and off-design task for the propulsion system and each of its components. A description of the propulsion system models is given in chapter 3. First, a single design point must be chosen for the propulsion system. With a suitable set of design variables, each component is sized according to the required thrust or power. Calculated parameters of interest are the mass and dimensions. The design is followed by offdesign calculations. At this stage the component geometry and mass remain fixed, and the operating characteristic is modelled, e.g. the efficiency in the operating range. All specified mission operating points are calculated. By adopting this approach, it is possible to investigate the impact of design variables on the performance during different operating points in the mission. In the last step of the conceptual design algorithm, the energy source is sized according to the power demand and operating time at each mission operating point. As a result the propulsion system mass and performance for all mission operating points is known for the initially defined design variables. The algorithm is visualized in Fig. 2. The design and off-design task will be described briefly in the subsequent paragraphs.

Design: All propulsion system components described in chapter 3.2 are sized for maximum power. The take-off operating point, which usually requires the highest power throughout the mission, has therefore been chosen as the design point for all propulsion systems. The design starts with the propeller. Rotational speed and torque input is iterated to generate the required take-off thrust. The resulting torque and rotational speed of the propeller shaft are used to size the subsequent components, e.g. the gear box and the



Fig. 2 Conceptual design algorithm

internal combustion engine, one after the other. Only the main propulsion system components are considered in the design method and this study, auxiliary components like the thermal management system are not included.

Off-design: After all components have been sized, the operating points defined by the mission are evaluated in off-design calculations. To calculate an off-design operating point, the power provided by the energy source is iterated such that the propeller can deliver the required thrust. This approach ensures, that the operating ranges of each component are not exceeded. The propeller can deliver the required thrust for different rotational speeds, since a variable pitch propeller is used. The rotational speed of the propeller is numerically optimized for lowest energy consumption, while keeping the blade tip Mach number below a maximum limit.

The algorithm described above is used for the fully electric and internal combustion engine propulsion systems. For the parallel hybrid system, the design algorithm is expanded. Two independent energy sources are incorporated in the parallel hybrid propulsion system (see Fig. 3c), hence an additional degree of freedom for the operation of such a propulsion system is present. The required propeller power can be supplied by the internal combustion engine or the electric motor. To define the operating strategy, the degree of hybridization H_p is defined as

$$H_P = \frac{P_{EM}}{P_{IC} + P_{EM}} \,. \tag{1}$$

This variable is a control variable that defines the power split between the electric motor and the internal combustion engine and has a significant influence on the overall design. In the conceptual design algorithm, the degree of hybridization can be chosen arbitrarily within the component operating limits and for all mission points. With the predefined degree of hybridization, at first, only the subsystem containing the propeller, gear box and internal combustion engine (see Fig. 3c) is sized at the design point. Afterwards

Table 2 Design constraints for all propulsion systems

Design constraint	Value		
Name	Symbol		
Max. prop. tip mach number	$M_{\rm tip}$	0.7	_
Max. prop. diameter	$D_{\rm prop}$	1.65	m
Max. casing temp. power el.	T_c	363.15	Κ
Maximum take-off mass		1000	kg



Fig. 3 Propulsion systems investigated: **a** internal combustion engine propulsion system (IPS), **b** fully electric propulsion system (EPS), **c** parallel hybrid propulsion system (PHPS)

off-design calculations are conducted with the predefined degree of hybridization. The power demand from the electric subsystem containing electric motor, inverter and DC cables is evaluated for all mission points and the electric components are sized for the highest power demand.

All free design variables for the propulsion systems under investigation have been chosen to minimize cruise energy consumption and propulsion system overall mass. As is described in chapter 3, many different component models with numerous free design variables are used as part of the calculation. A variation of all variables is not feasible. This means that only those variables with the highest impact on overall mass and efficiency are considered. These design variables are summarized in chapter 4 for each individual propulsion system and were determined using sensitivity studies. Additional design constraints which are used in this study are summarized in Table 2.

3 Propulsion system models

The following section describes in brief the propulsion systems under investigation along with their individual components. In addition, the physical models used for each component are presented.

3.1 Propulsion systems investigated

Figure 3 displays the propulsion systems under investigation, along with their respective components. The first propulsion system (Fig. 3a) consists of a propeller, gear box, internal combustion engine and fuel tank and will hereafter be referred to as the internal combustion engine propulsion system (IPS). The second propulsion system (Fig. 3b) is a fully electric propulsion system (EPS). The propeller is driven by a single electric motor via a gear box. The electric motor power is supplied by an inverter which is connected to a single battery pack via DC cables. The third propulsion system (Fig. 3c) is a parallel hybrid propulsion system (PHPS) combining the first two propulsion systems. The two power sources are mechanically connected by means of a gear box. In the electric systems, the connection between inverter and the subsequent component is realized via a DC cable of an arbitrary length. As stated in chapter 2.2 models for the conceptual design of other required auxiliary components like the thermal management system are not included. However, inputs are chosen and temperature boundary conditions for the electric components enforced to enable a realistic thermal management system. The relevant inputs and temperature calculations are highlighted in the description of the corresponding models below.

3.2 Component models

The following section describes the models used for the design and off-design task for each propulsion system component.

Propeller: During the design calculation, the geometry of the propeller is defined by the diameter, number of blades, activity factor and the integrated design lift coefficient. The activity factor is a dimensionless parameter related to the propeller blade width. The dimensionless integrated design lift coefficient in turn relates to the camber of the propeller blades. The mass of the propeller is calculated according to [19].

The off-design propeller performance is estimated using the Hamilton-Standard method [30], which is based on a collection of propeller maps. These propeller maps are independent of the propeller diameter, but dependent on number of blades, activity factor and integrated design lift coefficient. The propeller performance for a given number of blades, activity factor and design lift coefficient can be estimated using linear interpolation between the individual propeller maps. In [30] propeller maps are available for three to four propeller blades, activity factors from 80 to 220 and design lift coefficients from 0.15 to 0.7.

For all studies presented here, a three bladed variable pitch propeller is assumed and the diameter, activity factor and design lift coefficient are independent design variables. *Gear box*: The main design variable of a gear box is the gear ratio. It is an independent variable during design calculations, but fixed during off-design calculations. This study assumes a constant efficiency of 0.975 during design and off-design. The mass of the gear box is determined empirically as a function of power as well as input and output speed according to Brown et al. [5].

For the parallel hybrid propulsion system a gear box with two input shafts and one output shaft is required. The mass of the additional input shaft, housing etc. is approximated by the following assumption. For the parallel hybrid propulsion system, temporary gear box masses for both power paths are calculated. It is then assumed that the total gear box mass equals the higher mass, plus half of the lower mass. The latter mass then represents the additionally required input shaft. This is only a rough estimate. However, since the gear box mass is small compared to the other propulsion system components, the possible error introduced by this assumption is small on the system level.

Internal combustion engine: Current scaling methods for internal combustion engines, as described by Balazs [2], Huß [15] or Seibel [28], depend on an extensive database of different engine characteristics and measurements. These are not generally available to the public. In this study therefore, a modified approach based on Rizzoni et al. [26] is taken. This method defines the operational characteristics of the internal combustion engine by means of an efficiency map based on the mean effective pressure p_{me} and the mean piston speed c_m . These parameters do not vary much for one specific type of motor (e.g. turbocharged gasoline engine) and are defined for a four-stroke engine as

$$p_{me} = \frac{2P}{V_D N} \tag{2}$$

and

$$c_m = 2Ns . (3)$$

At the design stage, a scaling reference point in a given efficiency map is defined. For this point, the mean effective pressure and mean piston speed and efficiency are fixed. Assuming a constant bore to stroke ratio b/s and number of cylinders z, the power delivered by the internal combustion engine is only a function of the displacement volume V_D

$$P = \frac{C_m}{4\left[\frac{V_D}{z_{\frac{\pi}{4}, z^2}^{\frac{\mu^2}{3}}}\right]^{\frac{1}{3}}} V_D p_{me} \,.$$
(4)

The displacement volume is then iterated to match the required design power. The torque, rotational speed, and fuel flow of the internal combustion engine are results of the design calculation. The mass of the internal combustion engine is estimated on the basis of a constant power-toweight ratio of 1.0 kW/kg, which is a mean value for engines currently operated in aircraft.

During off-design, the basic geometric properties displacement volume, bore and stroke are fixed. This means that the resultant torque, rotational speed and fuel flow are known for every operating point in the efficiency map. The efficiency map used in this study was extracted from [6].

Electric machines: For the conversion from electric to mechanical power, three phase synchronous machines with permanent magnets are used. For the design, the dimensions and mass of the electric machine are determined. In current literature relating to hybrid aircraft propulsion systems, this is frequently done by assuming a constant power-toweight ratio. This is a valid approach if the motor is always designed for the same rotational speed. However, if a variation in the design speed is necessary, a different approach must be taken. The torque which an electric machine can deliver depends on the volume of the machine. The volume in turn, strongly correlates with the mass of the machine. At the design stage, the volume of the machine is estimated first. According to Müller and Vogt [22], the apparent power *S* is used to determine the basic rotor dimensions with

$$S = \frac{\pi^2}{\sqrt{2}} k_W A \hat{B}_\delta D_\delta^2 l N \,. \tag{5}$$

Except for the volume of the electric machine, all properties on the right-hand side of (5) are assumed constant throughout the design calculation and this study, since their variation with motor size is small within a certain range. Estimates for these values, depending on motor type and cooling, can be extracted from Müller and Vogt [22]. Assuming a constant rotor diameter-to-length ratio of D/l, we can solve (5) for the required rotor diameter and length. The apparent power is linked to the required mechanical power through the power factor $\cos(\varphi)$ and the efficiency of the machine η_{EM} according to

$$P_{\rm EM,mech} = 3U_s I_s \cos{(\varphi)} \eta_{\rm EM} = S \cos{(\varphi)} \eta_{\rm EM} \,. \tag{6}$$

The power factor and the efficiency are estimated inputs of the design calculation. Once the rotor dimensions are known, the stator dimensions can be calculated. The stator consists of ferromagnetic iron sheets with slots for the winding (see Fig. 4). The minimum area necessary for the conductors can be calculated with the maximum allowable current density J_S , which depends on the conductor material and cooling of the machine

$$A_S = \frac{A}{J_S} \frac{\tau_S}{\varphi_S} \,. \tag{7}$$



Fig. 4 Geometry of an electrical machine

The magnetic flux density in the teeth and back iron are limited to maintain low iron losses. Hence the geometry of the magnetic circuit can be determined according to

$$b_t = \frac{\hat{B}_{\delta}}{B_{\text{max,tooth}}} \frac{\tau_S}{\varphi_{\text{Fe}}}$$
(8)

and

$$h_{bi} = \frac{1}{\pi} \frac{\hat{B}_{\delta} \tau_p}{B_{\text{max,bi}} \varphi_{\text{Fe}}} \,. \tag{9}$$

To account for casings and bearings in the final dimensions of the machine, a constant factor on the stator length of 1.1 is used. The design calculation was verified against data from Vollmer [32]. Finally, the mass of the machine can be determined using a mean density for electric motors of 1439 kg/ m³. This mean density was established using a linear regression of weight over volume with data of machines currently operated in electric aircraft [13, 20, 35, 37].

The off-design behavior of the electric machine is modeled through the use of efficiency maps. Within the map, a scaling reference point is defined. This map is then linearly scaled such that the scaling reference point is equivalent to the design properties—like spool speed, torque and efficiency—of the electrical machine. To calculate the necessary phase current and phase voltage at a certain operating point in the map, an equivalent circuit for permanent magnet motors is used. The resulting equation system

$$I_{s,q} = \frac{\text{Trq}}{3p\psi_p\eta_{\text{EM}}} \tag{10}$$

$$U_{s,d} = R_S I_{s,d} - \omega L I_{s,q} \tag{11}$$

$$U_{s,q} = R_S I_{s,q} + \omega L I_{s,d} + \omega \psi_p \tag{12}$$

in the d,q reference frame can be solved at any operating point in the map. For the base speed area, a maximum torque

per ampere control of the motor is assumed. In the field weakening area, it is assumed that the maximum voltage of the inverter is reached. The unknowns in the equation system can be calculated from the design inputs. The efficiency map used in this study was extracted from [27].

Power electronics: For components like the inverter, various circuit designs are available. This study only considers the most basic circuit design and losses. Simplified circuit designs and determination of losses for the design calculation have been explained in detail by Vratny [33] and Wintrich et al. [36]. Only the basic design algorithm will be discussed here. Every power electronics component is built with basic semiconductor devices like IGBTs and diodes. The operating current and voltage of these semiconductors is limited. If these limits are exceeded, the power electronics component is designed for the required voltage and current by connecting semiconductors in series and parallel, effectively lowering the voltage and current for each semiconductor. By summing up all semiconductor masses and using a single calibrated mass factor, the overall mass of the component is calculated. The mass factor was calibrated with data from [8], resulting in a value of 5.56. The loss calculation implemented was verified with results from [29].

In addition to Vratny [33], a simple thermal calculation for all semiconductor modules is employed using

$$T_c = T_{\text{Max}} - P_{\text{Loss}} R_{\text{th}} \tag{13}$$

where T_{Max} is the maximum operating temperature of the semiconductor and R_{th} is the thermal resistance to the semiconductor casing. Both values are commonly stated in manufacturer data sheets. To prevent the semiconductor overheating, the casing temperature T_c calculated with (13) may not be exceeded. This relationship can also be interpreted the other way around: To ensure a realistic cooling system, the losses in the semiconductor have to be limited. In this study, the required casing temperature is kept above 363 K. This value has not been chosen arbitrarily, but represents a mean value of the maximum coolant inlet temperature of power electronic devices which are currently available.

DC cable: In the design calculation, the minimum conductor diameter is determined through the maximum permissible current density J_{DC} of the conductor. As for the electric machines, the permissible current density depends on the maximum operating temperature of the cable. The isolation thickness is calculated according to Cheng [7] and Vratny [33]. Given the length of the cables—which is a design input—it is possible to calculate the cable resistance and hence the losses occurring in the cables.

During off-design, an equivalent circuit model is used with the calculated design resistance. Copper as conduction material and a fixed cable length of 5 m are assumed for all propulsion systems containing DC cables.

Battery pack: A battery pack consists of several battery cells connected in series and parallel. Compared to a single cell, multiple cells connected in series can deliver a higher pack voltage, while cells connected in parallel can provide a higher current and higher capacity. Battery packs are therefore inherently scalable depending on the number of battery cells used. However, each cell has specific properties, such as nominal capacity, maximum C-Rate, cell voltage, mass and volume. Additionally, the operating characteristic may vary considerably depending on the cell design and chemistry, as described by Beard [3]. The modeling of a single cell and the pack design is extended from Vratny et al. [34]. The operating characteristics of the cells are modeled through tabulated discharge curves usually provided by the manufacturer. The other properties mentioned above are also provided on the basis of the manufacturer data.

To design the battery pack, mission power demand is integrated over time. First, the total required energy, the maximum voltage and the maximum power throughout the mission are calculated. Together with the nominal capacity and maximum C-Rate of a single cell, this gives a first estimate of the required cells in parallel (required capacity and maximum power) and series (required voltage). Subsequently, a discharge cycle over the mission is simulated. With the given discharge characteristic, if any of the conditions

$$C_{\max} > C_t \tag{14}$$

$$U_{\min} < U_t \tag{15}$$

$$SOC_{min} < SOC_t$$
 (16)

at mission time *t* are not met, the number of cells in series or parallel are adapted accordingly. If all requirements are met, the number of cells is known and the total cell mass and volume can be calculated. To account for additional mass of the structure, battery management system, cooling system, fuse box and power cabling within the battery pack, factors on mass and volume are used. Based on Nordmann [23], the factor on cell mass is estimated to be 1.25. To determine the battery pack volume, no factor on cell volume could be extracted from the literature, therefore it was estimated at 1.3.

Two different cells and their respective properties are considered during this study. For the parallel hybrid propulsion system, a cell optimized for high power demand [17] is used, since the electric system will only be utilized as a boost during take-off. For the fully electric system, a cell optimized for high gravimetric energy density [24] is employed.

Tank: The fuel demand of the internal combustion engine is integrated over time to calculate the required fuel mass.

Table 3 Thrust requirements atvarious operating points	Oppoint	Thrust-req.
	Take-off	2500 N
	Init. climb	1600 N
	Mid-climb	825 N
	Cruise	320 N

 Table 4 Design variables for the internal combustion engine propulsion system

Design variables		Final value	
Name	Symbol		
Propeller diameter	$D_{\rm Prop}$	1.65	m
Propeller activity factor	$f_{\rm AF}$	110	_
Propeller design lift coef.	$c_{\rm DL}$	0.525	-

The additional mass of the fuel system is considered according to Gudmundsson [14].

4 Design space exploration

This chapter analyzes the effects of the most important design variables on system mass and cruise efficiency. The optimum set of design variables for each propulsion system is first summarized. It is on the basis of these optimum values that parametric studies are conducted to highlight the impact of the design variables on mission performance and overall mass of the propulsion system. If a design variable is not changed during a parametric study, it is kept constant at the final value indicated in the respective table.

All propulsion systems in this chapter are optimized for the high thrust ratio requirements in Fig. 1. A MTOM of 1000 kg result in the thrust requirements summarized in Table 3. Since the propeller is the only component affected by the ambient conditions and the propulsion systems are only compared relative to each other, standard day conditions are assumed for simplicity. The cruise range is assumed to be 300 km. This value was chosen arbitrarily, to enable a battery sizing. A variation of this requirement will be briefly discussed in chapter 5. For the sake of simplicity, additional mission phases like descend or an alternate route are not considered in this study.

4.1 Internal combustion engine propulsion system

The most important design variables for the IPS were extracted from a sensitivity study and are summarized in Table 4. The propeller design has the biggest impact on the required take-off power and maximum cruise efficiency. The gear ratio would usually be a free design variable as well. During this study, however, it is iterated in such a way that the maximum power of the internal combustion engine is reached at take-off without violating the maximum tip Mach number constraint of the propeller (see Table 2).

During the conceptual design of the IPS, the diameter and the design lift coefficient are first optimized. For the given mission requirements, these two design variables have the biggest impact on propulsion system weight and cruise efficiency, while the activity factor is less important and hence optimized last.

The effect of propeller diameter and design lift coefficient on the propulsion system mass—excluding fuel mass—is shown in Fig. 5. Small propellers lead to a high power requirement during take-off, since the propulsive efficiency declines and the propeller has to be operated at high incidence angles. The mass of the propulsion system components grows in proportion to this power requirement. Assuming a constant propeller diameter, a high design lift coefficient will lead to a better take-off performance, since the propeller can be operated at low incidence while generating the required thrust, resulting in a higher propeller efficiency. These observations correspond to the recommendations of Borst [4], that is, the use of high design lift coefficients to achieve the smallest propeller diameter and mass.

The influence on system cruise efficiency can be seen in Fig. 6. As already discussed, high take-off power is required for small propeller diameters and design lift coefficients. Therefore a big combustion engine is necessary, which has to operate in deep part load during cruise conditions. This results in low cruise efficiencies. Big propeller diameters and design lift coefficients on the other hand, require small take-off powers and hence a small combustion engine. To efficiently operate big propellers with high design lift coefficients during cruise, low rotational speeds are required. Yet the small combustion engine can only provide the cruise power with high rotational speeds, again resulting in



Fig. 5 Influence of propeller diameter D_{Prop} and design lift coefficient c_{DL} on IPS mass

Contours: Sys. Eff. Cruise η_{Cruise} 0.70 \Box 0.65650 Lift Coeff. c_{DL} 0.600.55Final Design 0.50Des. 0.450.401.51.61.71.8

Fig. 6 Influence of propeller diameter D_{Prop} and design lift coefficient c_{DL} on IPS system cruise efficiency

Prop. Diameter D_{Prop} [m]

a decreasing system cruise efficiency. Thus for each propeller diameter an optimal design lift coefficient can be chosen.

Since a bigger propeller diameter leads to better cruise efficiency and lower propulsion system mass, a maximum permissible diameter of 1.65 m (see Table 2) has been chosen.

Finally, the activity factor is optimized together with the design lift coefficient for the given propeller diameter. The impact on the propulsion system mass and cruise efficiency is much smaller and an activity factor of 110 was chosen for optimum cruise efficiency. The final set of design variables is summarized in Table 4.

4.2 Fully electric propulsion system

The most important design variables of the EPS were extracted from a sensitivity study and are summarized in Table 5. Analogous to the IPS, the propeller geometry is first optimized. This fixes the power demand in each mission point. The gear box and electric components are subsequently optimized for the propeller power requirements.

Figure 7 shows the influence of propeller diameter and design lift coefficient on the EPS mass, excluding the battery mass. In comparison to the IPS in Fig. 5, the total mass

Table 5	Design	variables	for the	fully	electric	prop	oulsion	system
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Design variables		Value	
Name	Symbol		
Propeller diameter	$D_{\rm Prop}$	1.65	m
Propeller activity factor	$f_{\rm AF}$	110	_
Propeller design lift coef.	$c_{\rm DL}$	0.6	_
Gear ratio	$i_{\rm EM}$	5.0	-
EM number of pole pairs	р	3	_
DC voltage level	$U_{\rm DC}$	400.0	V



Fig. 7 Influence of propeller diameter D_{Prop} and design lift coefficient c_{DL} on EPS mass, excluding battery

is significantly lower, since the mass specific power of the electric components is almost twice that of a conventional internal combustion engine. This relationship is also the reason for the rather low sensitivity of the EPS mass to the propeller design, indicating that the EPS has an advantage over the IPS if the propeller diameter is restricted.

This advantage, however, is evened out if the battery is included and the total EPS mass $m_{\text{EPS,total}}$ is considered as shown in Fig. 8. Since the target cruise range is 300 km, the battery only has a nominal energy content of roughly 58 kWh, limiting the maximum power of the battery due to the maximum C-rate of the cells. Yet the power requirement at take-off quickly rises with small design lift coefficients and propeller diameters. The battery has to be sized for power and not for energy demand, resulting in very high battery masses. It must be emphasized that this effect depends on the energy content of the battery.

The trend towards high battery masses can be weakened if an additional high power battery is considered. The



Fig. 8 Influence of propeller diameter D_{Prop} and design lift coefficient c_{DL} on EPS total mass, including battery

additional battery mass of the power battery however, will not be compensated by propeller mass savings.

An operational advantage of the EPS over the IPS is shown in Fig. 9. The system cruise efficiency which can be attained is almost independent of the propeller diameter. This can be explained through the off-design characteristic of the electric motor. If necessary, electric motors can be operated at zero rotational speed while still producing a significant amount of torque. The minimum speed where the electric motor is able to produce the required power is only limited by the maximum torque the motor can produce. In contrast to the electric motor, the internal combustion engine has a minimum speed limit at which a significant torque can be produced. By using an electric motor, the propeller can therefore be operated at low rotational speeds and high efficiencies in cruise conditions. Furthermore, the efficiency of the electric motor is much less dependent on speed variation compared to the internal combustion engine.

For the EPS, the propeller design variables have not been chosen for the highest cruise efficiency, but rather the lowest total mass, including all propulsion system components and the battery. Taking into account Fig. 8 and a parametric study of activity factor and design lift coefficient analogue to the IPS, the propeller variables are fixed according to Table 5. In the next conceptual design step, the electric system is optimized.

Two of the three main design variables considered in this study are the gear ratio $i_{\rm EM}$ and the number of pole pairs *p* of the electric motor. Since the motor mass primarily scales with torque, a high gear ratio and low torque are advantageous. A high number of pole pairs also decreases the electric motor weight, since the required back iron height h_{bi} will be smaller (see Eq. (9)). High rotational speed and a high number of pole pairs will result however, in high electric frequencies, increasing the losses in the



Fig. 9 Influence of propeller diameter D_{Prop} and design lift coefficient c_{DL} on EPS system cruise efficiency



Fig. 10 Influence of number of pole pairs p and gear ratio $i_{\rm EM}$ on EPS total mass, including battery. Inverter casing temperature limited to 363 K

inverter. For this reason they have to be chosen carefully and cannot be optimized separately.

Figure 10 shows the influence of the gear ratio $i_{\rm EM}$ and the number of pole pairs *p* on the total EPS mass. Additionally, the limit for the inverter casing temperature is drawn: This can also be interpreted as a constant inverter efficiency line.

Since the operating temperature of the semiconductors is limited, high losses in the inverter will require low semiconductor casing temperatures. This ensures a sufficient heat flux to the cooling system. In the area highlighted in grey, the required casing temperature is below 363 K. This would result in a heavy cooling system, since the temperature delta between coolant and ambient decreases and big heat exchangers are required. This limit can be shifted if more power modules are used and the losses are distributed, while increasing the inverter mass.

It can be observed that the mass decreases with increasing gear ratio, especially at low gear ratios. The mass decrease stems from the lighter electric motor, which has to deliver less torque. For higher gear ratios, the mass savings are quickly saturated however, because

$$Trq_{\rm EM} \propto \frac{1}{i}$$
 (17)

Combining a high number of pole pairs and a high gear ratio results in a decreasing inverter efficiency due to the required high switching frequency. The decreasing electric motor mass is overcompensated by the increasing battery mass.

The lowest total mass can be achieved with a high gear ratio and low number of pole pairs. However, to avoid a complex gear box, three pole pairs and a gear ratio of five have been chosen, resulting in a reasonable trade between additional mass and gear box complexity.

Finally, the DC voltage level is optimized. Figure 11 shows its impact on the total propulsion system mass. The mass increases towards low DC voltage levels. The reason is the additional mass of the DC cables, the inverter and the battery. A lower voltage level results in higher currents if the power is constant. Cable cross sections at constant current density increase, resulting in higher cable masses. The operating current of the semiconductors in the inverter are limited, which means that more semiconductors in parallel are necessary, increasing the weight of the inverter. In addition, higher currents through the inverter increase the conduction losses, which results in a higher battery mass. The electric motor mass is not affected by the change in DC voltage. Since a constant electric loading is assumed in 5, only a rewinding of the motor takes place to yield the higher operating voltage.

Increasing the DC voltage is therefore always desirable. However, it increases the arching probability and may lead to certification issues, therefore an intermediate DC voltage of 400 V is chosen. The final set of design variables is summarized in Table 5.

4.3 Parallel hybrid electric propulsion system

For the PHPS the design variables of the IPS and EPS are combined. They are summarized in Table 6. The gear ratio of the internal combustion engine i_{IC} has once again been chosen, so that the engine can generate maximum power during take-off. In addition the take-off degree of hybridization defined in 1 is a design variable. During take-off the maximum power is required from the propulsion system, sizing the internal combustion engine as well as the electric components. For all other mission operating points the degree of hybridization is minimized to achieve the lowest possible battery mass. This operating approach was chosen due to the results of the fully electric propulsion system, for which the high battery mass poses a serious disadvantage.



Fig. 11 Influence of DC voltage level $U_{\rm DC}$ on EPS total mass, including battery

 Table 6
 Design variables for the parallel hybrid propulsion system

Design variables	Value			
Name Symbol				
Propeller diameter	D _{Prop}	1.65	m	
Propeller activity factor	$f_{\rm AF}$	110	-	
Propeller design lift coef.	$c_{\rm DL}$	0.36	-	
Degree of hybridization	$H_{P,\mathrm{TO}}$	0.46	-	
EM gear ratio	$i_{\rm EM}$	4.0	-	
EM number of pole pairs	р	4	-	
DC voltage level	$U_{\rm DC}$	400.0	V	



Fig. 12 Effect of downsizing the internal combustion engine

For the following study, the electric system will be used to assist the internal combustion engine during the high power demands (in this particular case during take-off and init. climb conditions). This enables a downsizing of the combustion engine, which may be beneficial if power demands differ strongly between take-off and cruise conditions. The downsizing process of the combustion engine not only influences the efficiency during low power demands, but also the operating range of the combustion engine. This relationship shall be explained in the following and is qualitatively illustrated in Fig. 12, where the power demands at take-off and cruise are shown as well as the operating limits of a reference motor and a downsized motor. For the sake of simplicity, it is assumed that the combustion engine is downsized by eliminating cylinders, keeping all other geometric parameters constant. Therefore the maximum and minimum speed limits of the motor are constant as well. Furthermore it is assumed that the propeller design and therefore the power demands for take-off and cruise remain constant. The following discussion therefore only incorporates changes to the combustion engine.

Due to the downsizing, the maximum torque produced by the engine decreases. The downsized motor cannot provide the take-off power any more. The power gap between required take-off power and maximum power of the combustion engine is filled by the electric motor. The downsized motor can then operate at optimum efficiency during cruise, while the bigger motor has to be operated at part load with a lower efficiency.

At the same time, the rotational speed range in which the motor can provide the cruise power, is much smaller for the downsized motor. Assuming a constant gear ratio chosen at take-off, this will result in a high rotational speed of the propeller at cruise conditions and, depending on the propeller design, low propeller efficiencies. We can deduce from this relationship that the degree of hybridization and the propeller geometry cannot be chosen independently.

Apart from choosing an appropriate propeller design, another option may remedy this problem: Instead of using a conventional spur gear box, a planetary gear box – summing the rotational speeds of the electric motor and combustion engine – can be employed. This will give an additional degree of freedom, since the gear ratio of the combustion engine can be chosen freely, while still using the engine to its full potential during take-off. However, this option is outside the scope of this study.

Figure 13 shows the influence of the three design variables – propeller diameter, design lift coefficient and degree of hybridization – on system cruise efficiency and the propulsion system mass. In the system mass, only the mass of the power converting components and the battery mass are included. Fuel mass is not included. Three parametric studies at three different take-off degrees of hybridization between 0 and 0.5 were conducted. Zero hybridization is equal to the IPS. In each study, the propeller diameter was varied between 1.45 and 1.65 m and the design lift coefficient between 0.36 and 0.6.

With an increasing degree of hybridization, the propulsion system mass decreases, since the mean power density of all the electric components is higher than the power density of the internal combustion engine. In this case the optimum degree of hybridization regarding mass and cruise efficiency is close to 0.5. With significantly higher degrees of hybridization the combustion engine can no longer provide the required mid-climb power. As a result the battery mass would increase significantly. In addition, the cruise efficiency of the combustion engine could only be improved marginally.

The aforementioned downsizing effect on propeller efficiency during cruise is visible for high degrees of hybridization. The propeller efficiency decreases quickly for big propeller diameters and design lift coefficients, overcompensating the efficiency gains in the internal combustion engine and resulting in a lower overall system cruise efficiency.

A comparison of the PHPS ($H_{P,TO} = 0.5$) and the IPS ($H_{P,TO} = 0$) enables a discussion on the potential mass savings and efficiency gains between the two propulsion systems. The propeller of the PHPS can either be designed for

Fig. 13 Influence of propel-

ler diameter D_{Prop} , design lift coefficient c_{DL} and take-off

PHPS mass and system cruise

efficiency



lowest propulsion system mass, resulting in a low cruise efficiency, or the highest cruise efficiency, resulting in a higher propulsion system mass. Compared to the IPS, at the most, 10% of propulsion system weight can be saved. In terms of cruise efficiency, at the most 2.5 percentage points can be gained, corresponding to 11% in fuel savings during cruise. The propulsion mass saving potential should be interpreted as a maximum potential, since the thermal management is not included in the mass. If liquid cooling is used, two separate cooling systems for the electric components and combustion engine might be necessary, since the temperature levels are very different. These circumstances will lead to a lower mass saving potential.

During this study, the PHPS is optimized for cruise efficiency, bearing in mind the trade-off between efficiency and mass previously discussed. After fixing the propeller geometry and degree of hybridization, the electric system is optimized by adopting the same approach presented in chapter 4.2. Since the underlying design trades are the same and are already explained in chapter 4.2, they are not detailed again. The final set of design variables is summarized in Table 6.

5 Propulsion system comparison under varying mission requirements

This section compares the propulsion systems investigated under varying thrust requirements. Each propulsion system was designed according to the method described in chapter 4. To vary the mission thrust requirements, consideration is given to the two sets of thrust requirements shown in Fig. 1. For the low thrust ratio requirement, the take-off to cruise thrust ratio is roughly 4.5 while the same ratio for the high thrust ratio requirement is 7.8. As metrics for comparison, the cruise efficiency and the total propulsion system mass, including energy storage, were chosen. The final values are shown in Table 7. In addition, values normalized to the IPS are given to facilitate comparison. The absolute results stated are depending on the specific use case. Due to the power and energy dependency of the battery, the relative comparison between the IPS and the EPS are also depending on the specific use case. Especially it is dependent on the relation between mission energy and take-off power demand, as will be shown later. However the relative comparison of mass and efficiency between the IPS and PHPS mainly depends on the thrust and power ratios and is therefore independent of the specific use case investigated in this paper.

For low thrust ratios between take-off and cruise, the PHPS is slightly heavier than the IPS. Nevertheless, it does offer a small improvement in cruise efficiency by 2%. However, this marginal advantage does not justify the additional complexity of the PHPS. For high thrust ratios between takeoff and cruise, again, the total mass of the PHPS and IPS are comparable. However, the PHPS has a cruise efficiency advantage of 12%.

Table 7 Comparison of propulsion systems for varying thrust requirements

Low thrust ratio	IPS	EPS	PHPS
m _{total}	117 kg	400 kg	119 kg
η_{Cruise}	0.253	0.825	0.259
Mass rel. to IPS	1.00	3.42	1.02
Eff. rel. to IPS	1.00	-	1.02
High thrust ratio	IPS	EPS	PHPS
m _{total}	150 kg	355 kg	148 kg
η_{Cruise}	0.230	0.81	0.257
Mass rel. to IPS	1.00	2.37	0.99
Eff. rel. to IPS	1.00	-	1.12

The efficiency advantage for high thrust ratios stems from the widely spread propeller power requirements in the different mission points. Converting the thrust requirements via the propeller to power requirements, the low thrust ratio requirement results in a take-off to cruise power ratio of roughly 2.2 while the same ratio for the high thrust ratio requirement is 4.5. A big power ratio results in a poor cruise efficiency for the IPS, since the internal combustion engine is operated in deep part load. By using an additional power source during take-off, the internal combustion engine can be downsized and a better cruise efficiency reached (see also Fig. 12).

The efficiency advantage therefore, depends on the power ratios between the different mission points. In turn, the power ratios depend on the trust ratios in conjunction with the propeller efficiencies at the respective operating conditions, since

$$P_{\rm Prop} = \frac{Fc_0}{\eta_{\rm Prop}} \,. \tag{18}$$

Hence for a given thrust ratio, the use of a PHPS may be particularly advantageous if the propeller design is limited by additional constraints. This may be a maximum diameter limit or a lower tip Mach number requirement during take-off in order to limit noise. Both lead to low propeller efficiencies during take-off and therefore to a bigger power ratio between take-off and cruise. The increasing advantages of the PHPS for a limited propeller diameter can also be observed in Fig. 13.

Compared to the IPS, the EPS is much heavier for both sets of thrust requirements. The reason is the heavy battery, although the required cruise range was only set to 300 km. Increasing the cruise range would also increase the battery mass. Not surprisingly, a fully electric propulsion system is therefore only feasible for short cruise ranges.

For high thrust ratios between take-off and cruise however, the factor between IPS and EPS total mass is much smaller. The difference can be explained as follows: The increased take-off power results in a mass increase from 117 to 150 kg for the IPS. Excluding the battery, the mass of the electric components of the EPS only increases from 47 to 59 kg, reducing the mass difference. The battery mass however is the predominant reason. It decreases from 352 to 296 kg. While the (short) take-off power requirement increases with wing loading, all other power requirements (e.g. mid-climb, cruise) decrease. This results in a much lower energy demand during the mission and consequently in a smaller battery. While the design for low thrust ratios requires a battery with a nominal energy of 70 kWh, the design for high thrust ratios only requires 59 kWh.

The conclusion might be made that—ignoring the stall speed limit—a higher wing loading than the highest one

chosen (see Fig. 1) is even better for the EPS, since the cruise thrust required then decreases even further. However, this is not the case. The reduced capacity of the battery also limits the maximum power output of the battery due to the maximum C-rate of the used cells. A further increase in the wing loading, and therefore the take-off power requirement, will quickly increase the battery mass, since it has to be sized by power and not by energy demand.

The optimum ratio between maximum power demand and mission energy can be estimated by calculating the maximum battery power according to

$$P_{\text{Bat,max}} = E_{\text{Bat}} C_{\text{max}} \,, \tag{19}$$

with E_{Bat} being the nominal energy stored in the battery and C_{max} the maximum C-rate of the used cells. This minimum mass criterion is only valid if a single battery is used. It may be feasible to use an additional high power battery pack to reduce the power requirement for the high energy battery pack during take-off.

In terms of cruise efficiency, the EPS has one major advantage, since it is not limited by the Carnot efficiency. A direct comparison is therefore not feasible.

In conclusion, with today's technology, the EPS has a severe mass disadvantage when compared to the IPS, even for short ranges. This disadvantage is minimized by balancing the energy and power demand throughout the mission. Nevertheless, it remains significant even for the aerodynamically very efficient aircraft type that was assumed and at a small cruise range of 300 km.

6 Summary and outlook

In this paper, a conceptual design method for hybrid electric propulsion systems in small aircraft is proposed and exemplified. Three propulsion systems are investigated, namely the internal combustion engine, parallel hybrid and the fully electric propulsion system. For each of these propulsion systems, the most important design variables are varied and their impact on the overall mission performance and system mass is investigated. The conducted studies highlight the importance of a profound system knowledge paired with a suitable tool during the conceptual design stage. Only if both aspects are satisfied, an optimal system design is possible. For instance it is shown, that for a parallel hybrid propulsion system the degree of hybridization and the propeller geometry cannot be optimized independently. The propeller geometry has an impact on the optimum spool speed during cruise, while high degrees of hybridization may limit the operating speed range of the internal combustion engine. Finally, the hybrid and electric propulsion system are compared

to the conventional internal combustion engine system. Potential fields of application are highlighted by varying the thrust requirements.

It is shown that for specific power requirements the parallel hybrid propulsion system may offer advantages in terms of system mass and cruise efficiency over the conventional internal combustion engine system. The propeller design is decisive if either efficiency or system mass advantages are to be fully exploited. The key parameter in determining whether a parallel hybrid is a beneficial solution is the maximum power ratio required throughout the mission, that is, the highest power demand in relation to the lowest power demand. This study suggests that advantages can be expected if the power ratio is bigger than 2.2. For the systems investigated here, at a power ratio of 4.5, the cruise efficiency could be improved by 12 % in relation to the internal combustion engine system.

For the fully electric propulsion system, one advantage over the internal combustion engine system in terms of operability is shown. The spool speed range of the electric motor which drives the propeller is much wider, compared to the internal combustion engine. In addition, the efficiency changes of the electric motor are small. Throughout the mission, the propeller can therefore be operated at maximum efficiency for the required thrust. However, the battery is responsible for a severe mass disadvantage. It is shown, that the battery mass can be minimized if the power and energy requirements throughout the mission are carefully balanced. Nevertheless, for the specific use case with a target cruise range of 300 km at 260 km/h, the fully electric system mass is higher by a factor of three in comparison to the lightest internal combustion engine system. This mass disadvantage may decrease with future battery developments, but given today's battery technology, the fully electric propulsion system would not seem feasible, especially for long cruise ranges. However, additional design targets, like minimum CO2 emissions or low operating costs may be a driver in implementing the heavier fully electric system with future battery technology.

Subsequent publications will investigate the series propulsion system. So far, only preliminary data is available. Compared to a simple combustion engine, efficiency is much lower, although the efficiency of the individual electric components connecting propeller and combustion engine is rather high. The reason for this is the high number of components necessary for the dual power conversion. In addition, more components have to be sized for the total power required, resulting in a heavier system. Nevertheless, the series propulsion system offers new possibilities of propulsion system integration, which may be desirable. Funding Open Access funding enabled and organized by Projekt DEAL.

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