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# Effects of pretension in bolts on hysteretic responses of moment-carrying timber joints 

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#### Abstract

The adoption of a concept similar to the prestressing technique used in laminated wood decks of bridge structures might increase the initial stiffness or ultimate resistance of dowel-type timber joints by applying pretension to their bolts. This study investigated the effect of pretension in bolts on hysteretic responses and ultimate properties of momentcarrying timber joints with steel side plates. A pretension of 20 kN that yielded a prestress level of 1600 kPa or about $90 \%$ of the allowable long-term end-bearing strength of spruce species was applied to the bolts of prestressed joints. The superiority of the prestressed joint over the non-pre-stressed joint was proved by very high hysteretic damping, equivalent viscous damping ratio, and cyclic stiffness. At any given rotation level, hysteretic damping reduction and moment resistance decrement due to continuously reversed loads were found to be small because bolt pretensioning minimized the pinching effect. This study showed that the hysteresis loop of the prestressed joint can be obtained by adding the frictional hysteresis loop due to pretension force into the hysteresis loop of the non-pre-stressed joint. Despite a great increase of initial stiffness, only slight increments in ductility coefficient and ultimate moment resistance were found in the prestressed joint.


Key words Pretension • Hysteretic response • Bolted joint • Pinching effect

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## Introduction

It has been reported that prestressing develops frictional resistance between laminas and by this an improvement of the structural performances of laminated wood decks can be obtained. ${ }^{1,2}$ In the field of timber connections, especially with dowel-type fasteners, a similar improvement could also be achieved by introducing pretension to their fasteners. Through a monotonic test, Quenneville and Dalen ${ }^{3}$ demonstrated that pretension in bolts enlarged the initial stiffness of split-ring timber joints besides increasing their lateral load-carrying capacities. Because loss of prestress (stress on wood member due to fastener pretensioning) may occur due to various factors, ${ }^{4,5}$ a minimum prestress level of 690 kPa at the time of construction has been recommended. ${ }^{6}$ Higher prestress levels $(1500 \mathrm{kPa}$ and 3000 kPa ) were also applied to the bolts of split-ring joints to improve their structural performances. ${ }^{3}$

In spite of its general application, the dowel-type timber joint is characterized by a very low stiffness at initial loading due to the existence of lead-hole clearance. ${ }^{7,8}$ Increasing the initial stiffness or load-carrying capacity of timber joints with dowel-type fasteners therefore could be done by applying pretension to the bolts. However, those improved performances were required to be further investigated under cyclic or dynamic loadings before being widely implemented. The objective of this study was to examine the effects of pretension in bolts on cyclic properties and ultimate strength of moment-carrying timber joints with steel side plates.

## Materials and methods

Spruce-pine-fir glued laminate of 2.44 m in length was fabricated by using three laminas bonded with resorcinolformaldehyde resin adhesive. Cold pressing was performed at room temperature for 24 h at 0.7 MPa . A joint specimen shown in Fig. 1 is composed of two glued laminated lumbers

Fig. 1. Test setup of momentresisting joint. $a=7 D(84 \mathrm{~mm})$; $b=10 D(120 \mathrm{~mm}) ; c=7.5 D$ $(90 \mathrm{~mm}) ; d=4 D(48 \mathrm{~mm})$; $e=100 \mathrm{~mm} ; f=65 \mathrm{~mm} ; h=$ $280 \mathrm{~mm} ; t=100 \mathrm{~mm} ; D$ is bolt diameter $=12 \mathrm{~mm}$; lead-hole clearance $=1 \mathrm{~mm} ; L$, span length $=3800 \mathrm{~mm} ; L V D T$, displacement transducer; Pm, potentiometer

that were connected to each other by two steel gusset plates of 4 mm thickness and $12-\mathrm{mm}$ bolts. The grade of steel used for the bolts and gusset plates was SS400 according to JIS G 3101. ${ }^{9}$ This joint specimen was supported through a pin system at both ends with a span length $(L)$ of 3800 mm and was loaded at two points as shown in Fig. 1. In this test, three displacement transducers (LVDTs) were used to measure the relative slip between the wood member and the steel plate, which was further used to calculate the joint rotation. In addition, one potentiometer ( Pm ) was used to evaluate the midspan vertical displacement relative to the fixed base. These displacement measurements were performed at the front and back sides of the joint specimen. The pin support system as shown in Fig. 1 was particularly designed without drilling holes in the wood member and it allowed the joint specimen to rotate and slide horizontally at both ends. The applied load was controlled using a displacement-controlled rate of $3.75 \mathrm{~mm} / \mathrm{min}$ for the midspan vertical displacement.

The cyclic test protocol shown in Fig. 2 consisted of six midspan vertical displacement levels: $4.5,9,13.5,18,22.5$, and 27 mm . At any displacement level, the cyclic load was repeated five times before going to the next level. Free vibration test that was carried out by giving an impact load using wooden hammer at the midspan of the joint specimens was conducted before and after the cyclic test to examine the effect of cyclic load application on vibration characteristics. In-plane vertical acceleration during the free vibration test was recorded with some accelerometers (see Fig. 1) and was used to evaluate the natural frequency of the joint specimens by using LabVIEW. ${ }^{10}$ In a quasistatic monotonic test, the specimens were loaded in the downward direction until failure or to 200 mm of midspan vertical displacement.


Fig. 2. Cyclic test protocol

Six non-pre-stressed joints, six prestressed joints, and three standard joints were prepared and directly tested. All these joint specimens had the same joint geometry. An axial bolt pretension of 20 kN was applied to each fastener of the prestressed joints by using a calibrated torque wrench device. Based on the contact area between the wood member and the steel gusset plate, the prestress level on the wood member due to the bolt pretension of 20 kN was approximately equal to 1600 kPa , which is about $90 \%$ of the allowable long-term end-bearing stress of spruce species $(1760 \mathrm{kPa}) .{ }^{11}$ The non-pre-stressed and prestressed joints were subjected to both free vibration and cyclic tests before finally loaded monotonically up to failure. The standard joints were loaded only monotonically up to failure and no axial pretension was applied to their bolts.

After the tests, small wood specimens were cut from the vicinity of the connection to evaluate the moisture content, specific gravity, and dowel-bearing strength. The average moisture content and specific gravity were found to be $13.11 \%$, and 0.38 , respectively. The bearing strength test yielded $5 \%$ offset dowel-bearing strength parallel to the grain of 26.62 MPa with a coefficient of variation of $8.1 \%$. Monotonic tension tests on three replicates of $12-\mathrm{mm}$ bolts showed an average ultimate tension-stress of 440 MPa , and proportional tension-stress of 273 MPa . As a result, the tension-stress in the bolt due to pretension of 20 kN was about 177 MPa and it was still in the elastic range.

## Results and discussion

The standard joint specimens reached an average ultimate moment resistance of 13.99 kNm and failed due to wood member splitting after having some bending deformation on their fasteners. In this study, the center of joint rotation was assumed to be fixed during loading so that the joint rotation could satisfactorily be evaluated only based on measurements of two LVDTs (LVDTs 1 and 2, see Fig. 1). Ultimate joint rotation of the standard specimens ranged from 0.089 to 0.135 radians with an average value of 0.111 radians, while the joint rotation corresponded to 27 mm of midspan vertical displacement as the cyclic test varied from 0.0135 to 0.0161 radians. The maximum rotation of the cyclic test therefore was about $10 \%$ of the ultimate rotation of the standard specimens or was in the serviceability range. ${ }^{12}$ However, the corresponding dowel embedment had reached the inelastic portion of the bearing stress-embedment curve shown in Fig. 3. For instance, the joint rotation of 0.0135 radians embedded the outermost bolts into the wood member by 0.96 mm ; assuming 0.5 mm of lead-hole clearance.


Fig. 3. Result of dowel-bearing test for loading parallel to the grain

Typical hysteresis curves of both non-pre-stressed and prestressed joints obtained from the cyclic test are shown in Fig. 4, where a pinching mechanism or less resistance at rotation close to zero points was observed in both hysteresis loops, especially in the non-pre-stressed joint. The magnitudes of moment resistance in the positive and negative directions of most hysteresis loops were found to be unequal because of natural randomness of lead-hole clearance among the fasteners. In fact, this eccentricity was also found even for single-nailed or single-bolted timber joints and it was strongly dependant on the selected cyclic test protocol. ${ }^{7,13}$ Less moment resistance was found in the non-pre-stressed joint compared with the prestressed joint for the same rotation level.

When the hysteresis loop of the prestressed joint is subtracted by the same amount of the hysteresis loop of the non-pre-stressed joint, a square hysteresis curve is obtained as an indication of frictional damping due to prestressing application. In other words, the hysteresis loop of prestressed joints can be obtained by simply adding the frictional hysteresis loop due to prestress force into the hysteresis loop of non-pre-stressed joints. Besides frictional force due to initial prestressing, frictional force in bolted timber joints may also be introduced by secondary axial force due to bending deformation of laterally loaded bolts. ${ }^{14}$ The hysteresis loop caused by this secondary axial force could be constructed in a similar way as the loop due to initial prestressing force and the magnitude of this loop depends essentially on the static coefficient of friction, which is affected by many parameters. ${ }^{15}$

Three properties were used to define the cyclic performance of the joint: (1) hysteretic damping, $\mathrm{E}_{\mathrm{D}} ;$ (2) equivalent viscous damping ratio, $\zeta_{\mathrm{eq}}$; and (3) cyclic stiffness, $k_{\mathrm{rc}}$. Figure 5 shows a typical hysteresis loop of a non-pre-stressed joint along with information required to calculate the three properties. The equivalent viscous damping ratio evaluated according to Fig. 5 was only an approximation because a


Fig. 4. Typical hysteretic moment rotation curves obtained from cyclic tests

Table 1. Cyclic properties of non-pre-stressed and prestressed joints at some rotation levels

| Joint | $E_{\mathrm{D}}(\mathrm{kNm}-\mathrm{rad})$ |  | $\frac{\zeta_{\mathrm{eq}}}{\text { Final }}$ | $k_{\text {rc }}(\mathrm{kNm} / \mathrm{rad})$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Initial | Final |  | Initial | Final |
| Non-pre-stressed |  |  |  |  |  |
| 0.0047 rads | $0.0108(0.0057-0.0136)^{\text {a }}$ | 0.0100 (0.0054-0.0129) | 0.24 (0.17-0.34) | 334 (196-503) | 313 (175-496) |
| 0.0093 rads | 0.0208 (0.0133-0.0264) | 0.0174 (0.0106-0.0239) | 0.16 (0.14-0.19) | 266 (166-361) | 239 (165-316) |
| 0.0148 rads | 0.0322 (0.0217-0.0390) | 0.0264 (0.0165-0.0323) | 0.10 (0.09-0.10) | 257 (127-358) | 236 (124-326) |
| Prestressed |  |  |  |  |  |
| 0.0047 rads | 0.0517 (0.0466-0.0586) | 0.0514 (0.0466-0.0580) | 0.33 (0.30-0.37) | 1558 (1235-1780) | 1471 (1216-1760) |
| 0.0093 rads | 0.1692 (0.1641-0.1752) | 0.1654 (0.1581-0.1744) | 0.34 (0.32-0.36) | 732 (633-785) | 713 (621-771) |
| 0.0148 rads | 0.2689 (0.2545-0.2816) | 0.2603 (0.2503-0.2708) | 0.32 (0.31-0.33) | 516 (421-617) | 489 (416-575) |

$\overline{E_{\mathrm{D}}}$, Hysteretic damping; $\zeta_{\mathrm{eq}}$, equivalent viscous damping; $k_{\mathrm{rc}}$, cyclic rotational stiffness
${ }^{\text {a }}$ Values in parentheses are minimum and maximum values


Fig. 5. Definition of hysteretic properties of the non-pre-stressed joint. $\zeta_{\text {eq }}$, Equivalent viscous damping
complex damping mechanism is naturally found in doweltype timber joints. ${ }^{16,17}$ The hysteretic damping is the area enclosed by the moment rotation curve, while the potential energy is the area of the triangle shown in Fig. 5. Because most of hysteresis loops were not symmetrical, the area of the triangle in both directions was averaged to accurately evaluate the potential energy. Finally, the cyclic stiffness as defined in Fig. 5 indicates how connection "softens" or degrades during the loading. The values of the cyclic properties for three different cyclic rotation levels are summarized in Table 1 where the hysteretic damping was evaluated by using SPSS SigmaScan. ${ }^{18}$ Cyclic rotations of $0.0047,0.0093$, and 0.0148 radians corresponded to 9,18 , and 27 mm of midspan vertical displacement, respectively.

Table 1 shows that the hysteretic damping of the prestressed joint is several times greater than that of the non-pre-stressedjoint.Hysteretic damping of the non-pre-stressed joints decreased as the number of cycles increased and its reduction occurred with a greater rate when the joints were cycled at higher rotation levels. The average hysteretic damping reductions of the non-pre-stressed joint at cyclic
rotations of $0.0047,0.0093$, and 0.0148 radians were about $5.17 \%, 9.66 \%$, and $17.27 \%$, respectively. On the other hand, the hysteretic damping of the prestressed joints during initial and last cycles was not much different because the frictional resistance minimized the inelastic slip component of viscoelastic damage to the wood fibers. Although the amount of damping reduction increased as the cyclic rotation level increased, the maximum reduction of hysteretic damping of the prestressed joint was found to be less than $5 \%$.

The equivalent viscous damping ratio shown in Table 1 was evaluated using the hysteresis loop of the last cycle. The equivalent viscous damping ratio of non-pre-stressed joints decreased sharply as the cyclic rotation level increased and it was essentially caused by narrowing of the hysteresis loop at rotations close to the zero point. The equivalent viscous damping ratio of prestressed joints remained fairly constant for all cyclic rotation levels tested in this study. Frictional damping due to friction between joint members greatly increases the equivalent viscous damping given in Table 1, especially in the case of prestressed joints. For an earth-quake-resistance design, therefore, the magnitude of equivalent viscous damping ratio summarized in Table 1 is required to be further examined at some higher rotation levels and also by considering loss of prestress force (stress relaxation).

Within a given rotation level, the continuously reversed cycles caused stiffness to decrease from the initial cycle to the final cycle. Decrease in cyclic stiffness from the initial to the final cycle ranged from $6 \%$ to $10 \%$ at non-pre-stressed joints, and from $2 \%$ to $6 \%$ at prestressed joints. The remaining inelastic slip component of viscoelastic damage of wood fibers during previous cycles potentially caused this cyclic stiffness degradation, which was also manifested through the decrease of moment resistance as shown in Fig. 6. In Figure 6, the moment resistance data at all cycles was converted to moment retention $\left(M_{n} / M_{1}\right)$, that is, the ratio of moment resistance at $n$th cycle to the moment resistance at the first cycle. By increasing the number of cycles, moment resistance of the non-pre-stressed joint decreased exponentially and it can be assumed to finally stabilize after four or five cycles. Moment resistance degradation of the prestressed joint, however, occurred almost linearly up to the


Fig. 6. Moment retention of non-pre-stressed and prestressed joints. $M_{n} / M_{1}$, Moment retention (ratio of moment resistance at the $n$th cycle to the moment resistance at the first cycle)


Time (s)
Fig. 7. In-plane vertical acceleration during free-vibration test. $f$, Natural frequency; $T$, period
last cycle with a lower degradation rate than that of the non-pre-stressed one. When cyclically loaded up to joint rotation set at 0.0148 radians, the average moment reduction of the non-pre-stressed and prestressed joints was about $17.5 \%$ and $5.25 \%$, respectively.


Fig. 8. Moment rotation curve obtained from quasistatic monotonic test. $k_{\mathrm{ri}}$, Initial rotational stiffness; $M_{\max }$, maximum moment resistance; $\theta_{M_{\max }}$, joint rotation at moment resistance $M_{\max } ; \theta_{\max }$, ultimate joint rotation

Table 2. Fundamental natural frequency of non-pre-stressed and prestressed joints

| Joint type | $f(\mathrm{~Hz})$ |
| :--- | :--- |
| Non-pre-stressed joint |  |
| $\quad$ Before cyclic load | $31.35(29.49-34.38)^{\mathrm{a}}$ |
| After cyclic load | $28.55(21.73-31.84)$ |
| $\quad$ Reduction | $9.13 \%(2.51-10.69)$ |
| Prestressed joint | $34.28(32.81-36.13)$ |
| Before cyclic load | $32.88(31.84-3.61)$ |
| After cyclic load | $4.05 \%(2.24-7.03)$ |
| Reduction |  |

${ }^{a}$ Values in the parentheses are minimum and maximum values

Even though the prestressed joint had greater hysteretic damping and cyclic stiffness than the non-pre-stressed one, only slight difference of fundamental natural frequency was found between the joints (see Table 2). The free vibration test results shown in Fig. 7 indicated that that the in-plane vertical acceleration of the non-pre-stressed joint was found to decrease more rapidly than that of the prestressed joint. Assuming that the free vibration test was carried out at a very small rotation level, the significant pinching mechanism observed in the non-pre-stressed joint seemed to have a positive effect in maintaining the overall dynamic response of joint specimens. Average reduction of fundamental natural frequency as an indicator of stiffness decrement was found to be about $9.13 \%$ and $4.05 \%$ for the non-pre-stressed and prestressed joints, respectively. Higher reduction of the natural frequency found in the non-pre-stressed joint was essentially affected by the joint deformation at zero resistance as observed during initial loading of the quasistatic monotonic test.

The relationship between moment resistance and joint rotation up to failure shown in Fig. 8 clearly shows that the pretension in bolts largely increases the initial rotational stiffness, given by the slope of the linear part at initial moment rotation curve, but slightly enhances the maximum


$$
\begin{aligned}
M_{s} & =2^{*} F_{s}{ }^{*} m \quad(2 \text { indicates two shear planes }) \\
& =2^{*} 0.5^{*}\left(1600^{*} 0.3^{*} 280^{*} 285\right)^{*} 142.5 \\
& =5.46 \mathrm{kNm}
\end{aligned}
$$

Fig. 9. Evaluation of sliding moment resistance $\left(M_{\mathrm{s}}\right) . F_{\mathrm{s}}$, frictional force $=\mu \times p \times A ; \mu$, coefficient of static friction; $p$, prestress on wood member; $A$, friction area; $m$, distance between friction force $=142.5 \mathrm{~mm}$
moment resistance. The fastener of the prestressed joint was bent with a smaller final bending angle than that of the fastener of the non-pre-stressed joint. Thus, additional resistance due to steel plate embedment into the wood member or so-called secondary friction of the fastener bending deformation was higher in the case of the non-pre-stressed joint than that of the prestressed one. The ultimate joint rotation defined as the rotation that corresponded to failure rotation or at $80 \%$ of maximum resistance, whichever occurred first in the test, was almost the same for both non-pre-stressed and prestressed joints. However, the prestressed joint reached the maximum moment resistance at a smaller joint rotation than did the non-pre-stressed joint. As a result, the ductility coefficient (the ratio of ultimate rotation to joint rotation at maximum moment resistance) of the prestressed joint was higher than that of the non-pre-stressed joint. The ductility coefficients were 1.23 and 1.54, respectively, for non-pre-stressed and prestressed joints. The quasistatic monotonic test of the non-pre-stressed joint yielded an average maximum moment resistance of 13.42 kNm , which is roughly the same as the maximum moment resistance of the standard joints. The introduction of cyclic loads up to joint rotation of $10 \%$ of the ultimate rotation did not have adverse effects on the maximum moment resistance.

Figure 8 shows a sudden decrease of rotational stiffness of the prestressed joints as an indication of slip between steel and wood members. The same slip occurrence is also observed in the hysteresis loop shown in Fig. 4 and occurred as the applied moment exceeded the frictional/sliding moment resistance. From six specimens, the average moment resistance at this slip occurrence was found to be about 4.78 kNm . Sliding moment resistance $\left(M_{\mathrm{s}}\right)$ of the joint due to frictional resistance between the wood member and the steel plate could be evaluated based on two frictional forces shown in Fig. 9. If a static friction coefficient of 0.3 is assumed, ${ }^{19}$ this moment resistance will be 5.46 kNm and is about $14 \%$ higher than the experimental result. This deviation might be caused by instant stress relaxation experienced by the fasteners because there was an interval between prestressing and the cyclic test.

## Conclusions

Quasistatic cyclic tests performed in this study showed that the introduction of pretension force to fasteners improves the structural performances of moment-carrying timber joints. The area enclosed by hysteresis loop, the so-called hysteretic damping, of the prestressed joint was several times greater than that of the non-pre-stressed joint. Pinching or narrowing of the hysteresis loop around zero rotation points as generally found in the non-pre-stressed joint was largely minimized. The positive effects of bolt pretensioning were further proved by a lower rate of reduction of moment resistance when cyclically loaded multiple times at same rotation level. Continuously reversed loads at any designated rotation level caused the moment resistance of the prestressed joint to decrease linearly with a lower degradation rate than that of the non-pre-stressed joint. The application of cyclic loads up to a rotation level of 0.0148 radians reduced the moment resistance by $17.25 \%$ and $5.25 \%$, respectively, for the non-pre-stressed and prestressed joints. However, it did not have adverse effects on the maximum moment resistance. A large increase of initial stiffness due to bolt pretensioning was followed by slight increases in ultimate resistance, ductility coefficient, and natural frequency.

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