Issues Concerning Isometric Deformations of Planar Regions to Curved Surfaces

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Received: 26 August 2017 / Published online: 8 December 2017

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Abstract In this work, we make a distinction between the differential geometric notion of an isometry relationship among two dimensional surfaces embedded in three-dimensional point space and the continuum mechanical notion of an isometric deformation of a two dimensional material surface. We illustrate the importance of separating the abstract theory of surfaces in differential geometry and their related differential geometric features from the physical notion of a material surface which is subject to a deformation from a given reference configuration. In differential geometry, while two surfaces may be isometric, the mapping between them that characterizes the isometry is simply a mapping between the points of the surfaces and not necessarily between corresponding material particles of a single deformed material surface.

We review two equivalent characterizations of a smooth isometric deformation of a flat material surface into a curved surface, and emphasize the requirement that the referential directrix and rulings, and their deformed counterparts, must provide a basis for establishing a complete curvilinear coordinate covering of the material surface in both the reference and deformed states. Because this covering requirement has been overlooked in recent publications concerning the isometric bending of ribbons, we illustrate its importance in properly defining the deformation of a ribbon in the two examples of a flat rectangular material strip that is isometrically deformed into either (i) a portion of a circular cylindrical surface, or (ii) a portion of a circular conical surface. We then show how the accurate calculation of the bending energy in these two examples is influenced by this oversight. In example (i), the

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curvature along the generators of the deformed surface, generally helical in form, is constant. In this special circumstance overlooking the covering requirement, as has been done in the literature by integrating the specific bending energy, dependent only on the curvature, over a domain on the supporting circular cylindrical surface equal in area, though not equal in geometric form, to that of the deformed ribbon, gives the correct bending energy result. In example (ii), the curvature along a generator of the cone is not constant and the calculation of the bending energy is, indeed, compromised by this oversight.

The historically important dimensional reductions that Sadowsky and Wunderlich introduced to study the bending energy and the equilibrium configurations of isometrically deformed rectangular strips have gained classical notoriety within the subject of elastic ribbons and Möbius bands. We show that the Sadowsky and Wunderlich functionals also overlook the covering requirement and that the exact bending energy is underestimated by these functionals, the Sadowsky functional being the lowest. We then show that the error in using these functionals can be great for a rectangular strip of given length ℓ and width w , depending on the form of the isometric deformation and the size of the half-length-to-width ratio $w/2\ell$. The Sadowsky functional is meant to apply to strips for which $w/2\ell$ is sufficiently small, in which case the covering requirement is of little consequence, and for such strips it yields an acceptable approximation of the actual bending energy. In such cases the Wunderlich functional shows only an incremental improvement over the Sadowsky calculation. While the Wunderlich functional is meant to apply accurately for all strips, without restricting the size of $w/2\ell$, we show that in overlooking the covering requirement it greatly underestimates the actual bending energy for many isometrically deformed ribbons. In particular, we show relative errors between the exact, the Wunderlich, and the Sadowsky calculations of the bending energy as a function of $w/2\ell$ for the case of a rectangular strip which is isometrically deformed into a portion of a right circular conical surface, and we observe that the error in approximating the exact bending energy by the Wunderlich functional for reasonable ratios $w/2\ell$ is large and unacceptable. We then give an example of the isometric deformation of a rectangular strip whose Wunderlich functional predicts zero bending energy but for which the exact bending energy can be as large as one pleases.

Finally, contrary to suggestions in the literature, we argue that Kirchhoff rod theory does not generally apply to the study of the isometric deformation of a thin rectangular strip because for this class of problems the through thickness dimension of the strip is assumed to be infinitesimal as compared to its width w . For Kirchhoff rod theory to apply, these dimensions must be comparable.

Keywords Isometry · Unstretchable · Inextensional · Two-dimensional Riemannian manifold · Embedding · Developable

Mathematics Subject Classification 53A05 · 74K15 · 74K35 · 57R40 · 53A45

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1 Introduction

In classical continuum mechanics, a body $\mathcal{B}$ may be viewed, geometrically, as a compact three-dimensional Riemannian manifold endowed with a Riemannian metric. A configuration of a body makes an identification between each body element x belonging to $\mathcal{B}$ and a point $\mathbf{x}$ in a subset $\mathcal{R}_*$ of three-dimensional Euclidean point space $\mathbb{E}^3$. In this setting, $\mathcal{R}_*$ is called the reference configuration of the body. A (smooth) deformation of the ‘body’ from $\mathcal{R}_*$ to some other subset $\mathcal{R}$ of $\mathbb{E}^3$ is then a mapping $\mathbf{x} \mapsto \mathbf{y}$, where $\mathbf{y}$ denotes a generic element of $\mathcal{R}$. In this setting, $\mathcal{R}$ is called the spatial (or deformed) configuration of the body. A deformation is isometric if lengths between all points in $\mathcal{R}_*$ and corresponding points in $\mathcal{R}$ are preserved under the mapping. Because a body cannot generally be embedded in $\mathbb{E}^3$, a three-dimensional Euclidean observer cannot discern its geometrical structure without recourse to special instruments.

An analogous level of clarity is absent from a significant portion of the literature concerned with ribbons and sheets conceived of as two-dimensional bodies, which we refer to as material surfaces. The difference between the abstract mathematical notion of a surface in two-dimensional Euclidean point space $\mathbb{E}^2$ and its embedding into $\mathbb{E}^3$ and the physical notion of the configuration of a material surface as viewed in $\mathbb{E}^3$ appears to have gone unappreciated in much of the literature dealing with configurations of ribbons and sheets in $\mathbb{E}^3$.

As an abstract mathematical entity, a material surface $\mathcal{P}$ may be considered, geometrically, to be a compact two-dimensional manifold endowed with a Riemannian metric. On this basis, an isometry between two distinct material surfaces can be defined and studied. A configuration of a material surface $\mathcal{P}$ rests on an identification between each element x of $\mathcal{P}$ and a point $\mathbf{x}$ in a subset $\mathcal{D}$ of $\mathbb{E}^2$; $\mathcal{D}$ is called the reference configuration of the material surface. A (smooth) deformation of the ‘material surface’ from $\mathcal{D}$ to $\mathcal{S}$ is

then a mapping $x \mapsto y$, where y is an element of $\mathbb{E}^3$. In this setting, $\mathcal{S}$ is called the spatial (or deformed) configuration of the material surface. A deformation from $\mathcal{D}$ to $\mathcal{S}$ is isometric if the length of any curve of points in $\mathcal{D}$ is preserved under the mapping. This stands in contrast to the prevailing view in classical differential geometry, where two surfaces are said to be isometric if they have the same Gaussian curvature at corresponding points and, in particular, a curved surface that is isometric to a planar region must be developable.

What perhaps makes things confusing when considering material surfaces and three-dimensional bodies is that, because of the embedding property, the Riemannian manifold and configurations of a material surface can be discerned by a three-dimensional Euclidean observer. Unless special care is taken in defining an isometry between two surfaces and an isometric deformation of a material surface, the distinction may therefore be easily missed. In particular, the pitfall of misconstruing the notion (and condition) of conservation of Gaussian curvature as the constraint appropriate to characterizing an unstretchable (or inextensional) two-dimensional body must be avoided.

We explain below, by analogy and at a reasonably fundamental level, our understanding of why much of the literature on unstretchable material surfaces is marred by confusion surrounding the distinction between the isometric deformation of a material surface and the differential geometric idea of an isometry between two surfaces. Section 2 is devoted to background. Drawing on the work of Chen, Fosdick and Fried [1], we define precisely what we mean by a smooth isometric deformation of a flat material surface into a curved surface, provide two equivalent conditions that are necessary and sufficient that a deformation of a flat material surface to a curved surface is isometric, and highlight the roles of the geometric objects central to our description. Most crucial among these are the referential and spatial directrices and the referential rulings and the spatial generators, which together provide a basis for establishing meaningful correspondences between parametrizations of the material surface in its reference and deformed configurations. In Sect. 3, we study a class of mappings introduced by Dias and Audoly [2] and show that all such mappings are isometric in the sense defined in Sect. 2. In Sect. 4, we consider issues that arise in connection with parametrizations of the reference and deformed configurations of a material surface. These issues hinge on the importance of ensuring a surjective correspondence between material points and parameter pairs. Dias and Audoly [2] overlook this issue and we find that their approach yields a complete covering of the referential and deformed surfaces only in the simple degenerate case where the reference configuration of the material surface is rectangular and is deformed into a (not necessarily circular) right cylindrical ring. Moreover, we discover that the “edge functions” of Dias and Audoly [2] fail to cure this difficulty. In Sect. 5 we explore how calculations of bending energy may be affected by failing to ensure a surjective correspondence between material points and parameter pairs. To illustrate our point, we compute the bending energy for a rectangular material strip bent to conform to a portion of a right circular conical surface. We then compare that energy to the analogous energy obtained by evaluating Wunderlich’s [3, 4] dimensionally-reduced energy functional, proving that that functional is strictly bounded above by the bending energy. We also find that Wunderlich’s [3, 4] functional provides an accurate estimate of the bending energy only in the limit in which the half-width-to-length ratio of the strip is infinitesimal and that Sadowsky’s [5, 6] functional performs just as well as Wunderlich’s [3, 4] in that regime. This stems from the absence of a surjective correspondence between material points and parameters that is inherent to the parametric approach employed by Wunderlich [3, 4] and those who have emulated his work or utilized his functional. In Sect. 6, we consider the implications of a key assumption in the theory of Kirchhoff rods, namely the assumption that the

cross sections of such a rod are rigid and, thus, in particular, cannot sustain in-plane deformations. Observing that the theory of Dias and Audoly [2] violates this assumption, we argue that their theory applies only to strip-like bodies that have widths comparable to their thicknesses and, thus, to Kirchhoff rods with infinitesimal cross sectional area or framed curves. Finally, in Sect. 7, we discuss and summarize our findings.

2 Preliminaries

Consider a material surface that is identified with an open, connected subset $\mathcal{D}$ of two-dimensional Euclidean point space $\mathbb{E}^2$. Suppose that $\mathcal{D}$ is deformed isometrically into a surface $\mathcal{S}$ in three-dimensional point space $\mathbb{E}^3$. Let $\{\boldsymbol{\iota}_1, \boldsymbol{\iota}_2\}$ be a fixed, positively oriented, orthonormal basis for the translation space $\mathbb{V}^2$ of $\mathbb{E}^2$ and define $\boldsymbol{\iota}_3 := \boldsymbol{\iota}_1 \times \boldsymbol{\iota}_2$ so that $\{\boldsymbol{\iota}_1, \boldsymbol{\iota}_2, \boldsymbol{\iota}_3\}$ is a fixed, positively oriented, orthonormal basis for the translation space $\mathbb{V}^3$ of $\mathbb{E}^3$.

2.1 Deformations

Following Chen, Fosdick and Fried [1], $\mathcal{D}$ can be represented in terms of a referential directrix $\mathcal{C}_0$ and a family of referential rulings that do not intersect in the interior of $\mathcal{D}$. Let $\hat{\mathbf{x}}_0$ be an arclength parametrization of $\mathcal{C}_0$. It is then useful to view $\mathcal{C}_0$ as a curve framed by a moving orthonormal triad $\{\boldsymbol{l}_1, \boldsymbol{l}_2, \boldsymbol{l}_3\}$ with elements

$$\boldsymbol{l}_1 = \boldsymbol{\iota}_3 \times \hat{\mathbf{x}}'_0, \quad \boldsymbol{l}_2 = \boldsymbol{\iota}_3, \quad \boldsymbol{l}_3 = \hat{\mathbf{x}}'_0, \quad (2.1)$$

where a prime denotes differentiation with respect to arclength, and to express the orientation of a generic referential ruling by a unit vector field

$$\frac{\boldsymbol{l}_1 + \nu \boldsymbol{l}_3}{\sqrt{1 + \nu^2}} = \boldsymbol{b}, \quad (2.2)$$

where ν measures the tangent of the angle between $\boldsymbol{l}_1$ and the rulings. With the foregoing provisions, each point $\mathbf{x} \in \mathcal{D} \subset \mathbb{E}^2$ can be expressed in terms of arclength α along $\mathcal{C}_0$ and position β along the rulings through a relation $\mathbf{x} = \hat{\mathbf{x}}(\alpha, \beta)$ with $\hat{\mathbf{x}}$ of the form¹

$$\hat{\mathbf{x}}(\alpha, \beta) = \hat{\mathbf{x}}_0(\alpha) + \beta \boldsymbol{b}(\alpha). \quad (2.3)$$

The directrix $\mathcal{C}_0$ is not necessarily a subset of $\mathcal{D}$. Indeed, for (α, β) to cover $\mathcal{D}$, $\mathcal{C}_0$ must generally contain segments that are disjoint from $\mathcal{D}$.² It is therefore overly restrictive to identify the referential directrix $\mathcal{C}_0$ with a material curve in $\mathcal{D}$.

The surface $\mathcal{S}$ admits a representation analogous to that of the flat region $\mathcal{D}$. In particular, the spatial directrix $\mathcal{C}$ of $\mathcal{S}$ is the image of the referential directrix $\mathcal{C}_0$ and has arclength parametrization $\hat{\mathbf{y}}_0$ satisfying

$$\hat{\mathbf{y}}'_0 = \boldsymbol{R} \hat{\mathbf{x}}'_0, \quad (2.4)$$

for some rotation field $\boldsymbol{R}$. Additionally, the generators of $\mathcal{S}$ are determined simply by rotating the rulings of $\mathcal{D}$ under $\boldsymbol{R}$, so that the generator $\boldsymbol{a}$ of $\mathcal{S}$ corresponding to a referential ruling with orientation $\boldsymbol{b}$ is given by

$$\boldsymbol{a} = \boldsymbol{R} \boldsymbol{b}. \quad (2.5)$$

¹In the present work, we replace the coordinates (η^1, η^2) used by Chen, Fosdick and Fried [1] with (α, β) .

²Moreover, it is possible and sometimes even necessary to choose $\mathcal{C}_0$ to be disjoint from $\mathcal{D}$.

In view of (2.4) and (2.5), the point $\mathbf{y}$ on $\mathcal{S}$ corresponding to the point $\mathbf{x} = \hat{\mathbf{x}}(\alpha, \beta)$ on $\mathcal{D}$ is determined by a relation $\mathbf{y} = \hat{\mathbf{y}}(\alpha, \beta)$, with $\hat{\mathbf{y}}$ being of the form

$$\hat{\mathbf{y}}(\alpha, \beta) = \hat{\mathbf{y}}_0(\alpha) + \beta \mathbf{a}(\alpha). \quad (2.6)$$

Since, as observed earlier, $\mathcal{C}_0$ must generally include segments that are disjoint from $\mathcal{D}$, a continuity argument leads to the conclusion that $\mathcal{C}$ must generally contain segments that are disjoint from $\mathcal{S}$.

Chen, Fosdick and Fried [1] emphasize that the mappings defined in (2.3) and (2.6) are not meaningful unless each pair (α, β) corresponds to a unique material point $\mathbf{x}$ in $\mathcal{D}$ and the underlying correspondence between such coordinate pairs and points provides a complete covering of $\mathcal{D}$. Granted that these requirements are met, there exist scalar-valued mappings $\tilde{\alpha}$ and $\tilde{\beta}$ defined on $\mathcal{D}$ and a mapping $\tilde{\mathbf{x}}$ from $\mathcal{D}$ to itself such that

$$\mathbf{x} = \tilde{\mathbf{x}}(\mathbf{x}) = \hat{\mathbf{x}}(\tilde{\alpha}(\mathbf{x}), \tilde{\beta}(\mathbf{x})) \quad (2.7)$$

for each $\mathbf{x}$ in $\mathcal{D}$. Moreover, $\tilde{\mathbf{y}}$ defined by

$$\mathbf{y} = \tilde{\mathbf{y}}(\mathbf{x}) = \hat{\mathbf{y}}(\tilde{\alpha}(\mathbf{x}), \tilde{\beta}(\mathbf{x})) \quad (2.8)$$

constitutes a deformation from $\mathcal{D}$ to $\mathcal{S}$.

Let ∇ denote the gradient with respect to position $\mathbf{x}$ in $\mathcal{D}$ and note that the dual basis vectors $\mathbf{g}^1$ and $\mathbf{g}^2$ are given by

$$\mathbf{g}^1 = \nabla \tilde{\alpha} \quad \text{and} \quad \mathbf{g}^2 = \nabla \tilde{\beta}. \quad (2.9)$$

Then, since

$$(\nabla \tilde{\mathbf{x}})^\top \nabla \tilde{\mathbf{x}} = \left| \frac{\partial \tilde{\mathbf{x}}}{\partial \alpha} \right|^2 \mathbf{g}^1 \otimes \mathbf{g}^1 + \frac{\partial \tilde{\mathbf{x}}}{\partial \alpha} \cdot \frac{\partial \tilde{\mathbf{x}}}{\partial \beta} (\mathbf{g}^1 \otimes \mathbf{g}^2 + \mathbf{g}^2 \otimes \mathbf{g}^1) + \left| \frac{\partial \tilde{\mathbf{x}}}{\partial \beta} \right|^2 \mathbf{g}^2 \otimes \mathbf{g}^2 \quad (2.10)$$

and

$$(\nabla \tilde{\mathbf{y}})^\top \nabla \tilde{\mathbf{y}} = \left| \frac{\partial \tilde{\mathbf{y}}}{\partial \alpha} \right|^2 \mathbf{g}^1 \otimes \mathbf{g}^1 + \frac{\partial \tilde{\mathbf{y}}}{\partial \alpha} \cdot \frac{\partial \tilde{\mathbf{y}}}{\partial \beta} (\mathbf{g}^1 \otimes \mathbf{g}^2 + \mathbf{g}^2 \otimes \mathbf{g}^1) + \left| \frac{\partial \tilde{\mathbf{y}}}{\partial \beta} \right|^2 \mathbf{g}^2 \otimes \mathbf{g}^2, \quad (2.11)$$

we see that a deformation $\tilde{\mathbf{y}}$ from $\mathcal{D}$ to $\mathcal{S}$ defined through (2.3) and (2.6) in accord with (2.8) is isometric—in the sense that it preserves the length of any material curve in $\mathcal{D}$ —only if both of the following conditions hold:

- There exist mappings $\tilde{\alpha}$ and $\tilde{\beta}$ such that each point $\mathbf{x}$ in $\mathcal{D}$ and its image $\mathbf{y}$ on $\mathcal{S}$ obey (2.7) and (2.8), where $\hat{\mathbf{x}}$ and $\hat{\mathbf{y}}$ are as defined in (2.3) and (2.6).
- The partial derivatives of $\hat{\mathbf{x}}$ and $\hat{\mathbf{y}}$ defined in (2.3) and (2.6) satisfy

$$\left| \frac{\partial \hat{\mathbf{y}}}{\partial \alpha} \right|^2 = \left| \frac{\partial \hat{\mathbf{x}}}{\partial \alpha} \right|^2, \quad \frac{\partial \hat{\mathbf{y}}}{\partial \alpha} \cdot \frac{\partial \hat{\mathbf{y}}}{\partial \beta} = \frac{\partial \hat{\mathbf{x}}}{\partial \alpha} \cdot \frac{\partial \hat{\mathbf{x}}}{\partial \beta}, \quad \text{and} \quad \left| \frac{\partial \hat{\mathbf{y}}}{\partial \beta} \right|^2 = \left| \frac{\partial \hat{\mathbf{x}}}{\partial \beta} \right|^2 \quad (2.12)$$

for each parameter pair (α, β) corresponding to a point $\mathbf{x}$ in $\mathcal{D}$ or, equivalently, by (2.4)–(2.6), $\mathbf{R}$ and $\mathbf{b}$ satisfy

$$\mathbf{R}'\mathbf{b} = \mathbf{0}, \quad (2.13)$$

for each admissible choice of the only argument α on which they depend.

For all remaining subsections of the present section, we assume that the first of the above bullet items holds and focus only on exploring the ramifications of the condition (2.13) necessary and sufficient to ensure that the second bullet item holds. Later, in Sect. 4, we consider the remaining bullet item.

2.2 Alternative Conditions for Isometry

Since $\mathbf{R}'\mathbf{b} = \mathbf{R}'\mathbf{R}^\top \mathbf{R}\mathbf{b} = \mathbf{R}'\mathbf{R}^\top \mathbf{a}$, it is convenient to introduce the axial vector

$$\mathbf{p} = \text{ax}(\mathbf{R}'\mathbf{R}^\top) \quad (2.14)$$

of the skew tensor $\mathbf{R}'\mathbf{R}^\top$ associated with the rotation $\mathbf{R}$ and to express the isometry condition (2.13) in the equivalent alternative form

$$\mathbf{p} \times \mathbf{a} = \mathbf{0}. \quad (2.15)$$

As an immediate consequence of (2.4), we see that $\tilde{\mathbf{y}}$ defined by (2.8) is an isometric deformation if and only if $\mathbf{p}$ and $\mathbf{a}$ are collinear. Thus, $\mathbf{p}$ must be tangent to $\mathcal{S}$. Moreover, since $|\mathbf{a}| = |\mathbf{R}\mathbf{b}| = |\mathbf{b}| = 1$, (2.13) holds if and only if $\mathbf{p}$ satisfies $\mathbf{p} = (\mathbf{a} \otimes \mathbf{a})\mathbf{p}$ and thus admits the representation³

$$\mathbf{p} = p\mathbf{a}, \quad \text{with} \quad p = \mathbf{p} \cdot \mathbf{a}. \quad (2.16)$$

In accord with its definition (2.14) and representation (2.16), $\mathbf{p}$ is the rate of rotation of the generators of $\mathcal{S}$ along the spatial directrix $\mathcal{C}$. Moreover, that rate has magnitude $|\mathbf{p}| = p$.

Recalling the definition (2.14) of $\mathbf{p}$, we may rewrite (2.13) as a tensorial ordinary-differential-equation

$$\mathbf{R}' = p\mathbf{A}\mathbf{R} \quad (2.17)$$

for $\mathbf{R}$, where $\mathbf{A} = \mathbf{a} \times$ is the skew tensor with axial vector the spatial generator $\mathbf{a}$. To determine the isometric deformation $\tilde{\mathbf{y}}$ from $\mathcal{D}$ to $\mathcal{S}$, it is therefore necessary to first solve (2.17) for the rotation tensor $\mathbf{R}$ subject to an initial condition $\mathbf{R}(0) = \mathbf{R}_0$, with $\mathbf{R}_0$ being a given rotation. This is no easy task. Indeed, (2.17) cannot be solved for $\mathbf{R}$ unless p and $\mathbf{a}$ are given. Although it is known that a solution exists,⁴ no general explicit closed-form solution is available. The alternative is to use approximation schemes such as the classical Magnus [7] expansion method. However, there may be issues related to the dimensions of $\mathcal{D}$ and additional conditions which do not enter into the problem statement of (2.17) that are necessary to assure that no self-intersection occurs. A general solution strategy that overcomes these obstacles does not yet exist. A complete explicit treatment of the particular example of the isometric deformation of a rectangular material strip onto a portion of a right circular conical surface is provided by Chen, Fosdick and Fried [1, Sect. 9].

2.3 Framing of the Spatial Directrix

The orthonormal triad $\{\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3\}$ with elements

$$\mathbf{e}_i = \mathbf{R}\mathbf{l}_i, \quad i = 1, 2, 3, \quad (2.18)$$

³Here, we use p in place of the symbol λ used by Chen, Fosdick and Fried [1].

⁴See, for example, Hartman [9].

provides a framing of the spatial directrix $\mathcal{C}$. From (2.1) and (2.18), we infer that $\mathbf{e}_1$ is tangent to $\mathcal{S}$, $\mathbf{e}_2$ is normal to $\mathcal{S}$, and $\mathbf{e}_3$ is tangent to $\mathcal{C}$ and, thus, also to $\mathcal{S}$. Recalling that $\{\mathbf{l}_1, \mathbf{l}_2, \mathbf{l}_3\}$ is also an orthonormal triad, we see from (2.18) that $\mathbf{R}$ entering (2.18) admits the representation

$$\mathbf{R} = \sum_{i=1}^3 \mathbf{e}_i \otimes \mathbf{l}_i. \quad (2.19)$$

Moreover, using (2.18) in the expression (2.5) for the generator $\mathbf{a}$ of $\mathcal{S}$ corresponding to the ruling $\mathbf{b}$ of $\mathcal{D}$ defined by (2.2), we find that

$$\mathbf{a} = \frac{\mathbf{e}_1 + \nu \mathbf{e}_3}{\sqrt{1 + \nu^2}}. \quad (2.20)$$

Recalling (2.16), we therefore arrive at a representation,

$$\mathbf{p} = \frac{p(\mathbf{e}_1 + \nu \mathbf{e}_3)}{\sqrt{1 + \nu^2}}, \quad (2.21)$$

for the axial vector $\mathbf{p}$ of the skew tensor $\mathbf{R}'\mathbf{R}^\top$ associated with the rotation $\mathbf{R}$.

2.4 Darboux Vector of the Spatial Directrix

From the theory of framed curves (Bishop [8]), it is known that the variation of the triad $\{\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3\}$ along $\mathcal{C}$ is completely described, modulo a rigid transformation, by the differential equation

$$\mathbf{e}'_i = \boldsymbol{\delta} \times \mathbf{e}_i, \quad i = 1, 2, 3, \quad (2.22)$$

where $\boldsymbol{\delta}$ is the Darboux vector of $\mathcal{C}$. The components $\delta_i = \boldsymbol{\delta} \cdot \mathbf{e}_i$, $i = 1, 2, 3$, of $\boldsymbol{\delta}$ relative to $\{\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3\}$ can be found from (2.22) and are given by

$$\delta_1 = -\mathbf{e}'_3 \cdot \mathbf{e}_2, \quad \delta_2 = \mathbf{e}'_3 \cdot \mathbf{e}_1, \quad \delta_3 = \mathbf{e}'_1 \cdot \mathbf{e}_2 = -\mathbf{e}'_2 \cdot \mathbf{e}_1. \quad (2.23)$$

Since $\mathbf{e}_3$ is tangent to $\mathcal{C}$, its arclength derivative $\mathbf{e}'_3$ is the curvature vector of $\mathcal{C}$. While δ_1 and δ_2 measure the curvature of $\mathcal{C}$ about $\mathbf{e}_2$ and $\mathbf{e}_1$, respectively, δ_3 describes the precession of $\{\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3\}$ about $\mathbf{e}_3$.

Bearing in mind the relation (2.18) determining $\{\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3\}$ in terms of the referential triad $\{\mathbf{l}_1, \mathbf{l}_2, \mathbf{l}_3\}$ and the alternative version (2.15) of the isometry condition (2.13), we next consider how the restrictions (2.1) on $\{\mathbf{l}_1, \mathbf{l}_2, \mathbf{l}_3\}$ influence the properties of the Darboux vector $\boldsymbol{\delta}$ of $\mathcal{C}$. Toward this objective, we first define the scalar curvature k of the referential directrix $\mathcal{C}_0$ in accord with

$$\hat{\mathbf{x}}''_0 = k\mathbf{l}'_1. \quad (2.24)$$

Differentiating (2.1) with respect to arclength, we then find that

$$\mathbf{l}'_i = k\mathbf{l}_2 \times \mathbf{l}_i, \quad i = 1, 2, 3. \quad (2.25)$$

Then, using the respective consequences

$$\mathbf{R}'\mathbf{l}_i = \mathbf{R}'\mathbf{R}^\top \mathbf{R}\mathbf{l}_i = \mathbf{p} \times \mathbf{e}_i, \quad i = 1, 2, 3, \quad (2.26)$$

and

$$R(l_i \times l_j) = R((R^\top e_i) \times (R^\top e_j)) = e_i \times e_j, \quad i, j = 1, 2, 3, \quad (2.27)$$

of (2.14) and (2.18) to simplify the condition that ensues upon differentiating (2.18) with respect to arclength, we arrive at a differential equation

$$e'_i = (p + ke_2) \times e_i, \quad i = 1, 2, 3, \quad (2.28)$$

which, when compared with (2.22), leads to the conclusion that the Darboux vector δ must be of the form

$$\delta = p + ke_2. \quad (2.29)$$

By (2.16), (2.20), and (2.29), the components of δ relative to $\{e_1, e_2, e_3\}$ can now be expressed as

$$\delta_1 = \frac{p}{\sqrt{1+v^2}}, \quad \delta_2 = k, \quad \delta_3 = \frac{pv}{\sqrt{1+v^2}}. \quad (2.30)$$

Moreover, by (2.23), the curvature vector $\hat{y}''_0 = e'_3$ of $\mathcal{C}$ takes the form

$$\hat{y}''_0 = ke_1 - \frac{pe_2}{\sqrt{1+v^2}}. \quad (2.31)$$

Since $\{e_1, e_2, e_3\}$ is orthonormal with e_2 being normal to $\mathcal{S}$ and e_3 being tangent to $\mathcal{C}$, we see from (2.30)_{1,2} and (2.31) that, as a curve in the surface $\mathcal{S}$, $\mathcal{C}$ has normal and geodesic curvatures

$$k_n = -\frac{p}{\sqrt{1+v^2}} = -\delta_1 = -\frac{\delta_3}{v} \quad \text{and} \quad k_g = k = \delta_2. \quad (2.32)$$

Lastly, from (2.30)₃, we note that as $\{e_1, e_2, e_3\}$ traverses $\mathcal{C}$ it rotates about e_3 with “angular velocity” $\delta_3 = pv/\sqrt{1+v^2}$.

Since $k = k_g$ by (2.32)₂, the representation (2.29) for the Darboux vector δ of $\mathcal{C}$ can be rewritten in the form $p + k_g e_2$. We thus see that the basic isometry condition (2.13) is equivalent to the requirement that

$$(\delta - k_g e_2) \times a = 0. \quad (2.33)$$

Adapting calculations performed by Chen, Fosdick and Fried [1, Sect. 6] to the present setting, we find that the curvature tensor $L = -\text{grad}_{\mathcal{S}} e_2$ of $\mathcal{S}$ takes the explicit form

$$L = -p\sqrt{1+v^2}a^\perp \otimes a^\perp = k_n(1+v^2)a^\perp \otimes a^\perp, \quad (2.34)$$

where $a^\perp$ is the unit vector field, orthogonal to a , defined by

$$a^\perp = \frac{ve_1 - e_3}{\sqrt{1+v^2}}. \quad (2.35)$$

Since $a \times a^\perp = e_2$, $\{a, a^\perp, e_2\}$ provides an orthonormal basis for $\mathbb{V}^3$ on $\mathcal{S}$.

3 Interpretation of the Class of Mappings Considered by Dias and Audoly [2]

To describe a strip-like material surface in a planar reference configuration and in an associated deformed configuration obtained by bending, Dias and Audoly [2, equation (5)] introduce, in their notation, ruled parametrizations

$$Y = X(S) + V Q(S) \quad (3.1)$$

and

$$y = x(S) + V q(S), \quad (3.2)$$

where S measures arclength along the directrices parametrized by X and x and V determines position along the referential rulings and spatial generators that are respectively parallel to Q and q .⁵ The vector quantities Q and q are taken to be of the form

$$Q = D_1 + \eta D_3, \quad q = d_1 + \eta d_3, \quad (3.3)$$

where $\{D_1, D_2, D_3\}$ and $\{d_1, d_2, d_3\}$ are moving orthonormal triads for the referential and spatial directrices, D_3 and d_3 satisfy the conditions⁶

$$D_3 = X', \quad d_3 = x', \quad (3.4)$$

which are familiar from the description of framed curves (and Kirchhoff rods), and η is the tangent of the shared angle between D_1 and Q and between d_1 and q . Here, and in the remainder of this section, a superscripted prime is used to denote differentiation with respect to the arclength parameter S . Since in differential geometry, the surfaces that are isometric to planar regions are the special ruled surfaces that are developable, and these surfaces are not necessarily isometric deformations of one another, and since the work of Dias and Audoly [2] is based on this differential geometric concept of isometry, further explanation of their work seems necessary.

Dias and Audoly [2] refer to S and V as “longitudinal” and “transverse” coordinates. From (3.4), S provides a one-to-one correspondence between points on the referential directrix, which Dias and Audoly [2] identify as a locus of material points,⁷ and their images on the spatial directrix. An analogous statement does not, however, apply to V . This raises a need to properly characterize the inescapably material points that lie on the edges of the material surface. To address this need, Dias and Audoly [2] introduce “edge functions” $V_{\pm}(\eta, S)$ and in this regard state that “ S varies in the interval $0 \leq S \leq L$, where L is the curvilinear length of the center-line,” and that “ V varies in a domain $V_{-}(\eta, S) \leq V \leq V_{+}(\eta, S)$ ”, where “ $V_{\pm}(\eta, S)$ encode the relative positions of the edges of the material surface with respect to the center-line”.⁸ However, they do not explain whether—or, if so, how—the ordered pairs (S, V) correspond to material points.

⁵Dias and Audoly [2] refer to the directrices as “center-lines”.

⁶While (3.4)₁ appears in the penultimate sentence of the first paragraph in Sect. 2.1 in the paper of Dias and Audoly [2], (3.4)₂ is (2*) of that paper.

⁷See the third sentence in the first paragraph of Sect. 2.1 of Dias and Audoly [2].

⁸See the discussion beginning in the final paragraph on page 52 of Dias and Audoly [2].

Because Dias and Audoly [2] require that the directrices are loci of material points, their framework does not apply in situations where the directrices include segments that lie outside the boundaries of the referential and spatial manifestations of the material surface. For this reason, only under special circumstances would the corresponding collection

$$\{(S, V) : 0 \leq S \leq L, V_-(\eta, S) \leq V \leq V_+(\eta, S)\} \quad (3.5)$$

of all admissible combinations of S and V cover the material surface in either of its configurations. The ramifications of this lapse are addressed in Sect. 4.

From (3.3), we see that $\{\mathbf{D}_1, \mathbf{D}_2, \mathbf{D}_3\}$ and $\{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\}$ are selected so that $\mathbf{D}_2$ is normal to the plane occupied by the material surface in the reference configuration and that $\mathbf{d}_2$ is normal to the tangent space of the spatial surface occupied by the material surface in the deformed configuration. This assumption leads to supplemental conditions of the form⁹

$$\mathbf{D}'_3 \cdot \mathbf{D}_1 = \mathbf{d}'_3 \cdot \mathbf{d}_1 = \kappa_g, \quad \mathbf{d}'_1 \cdot \mathbf{d}_2 = \eta \mathbf{d}'_2 \cdot \mathbf{d}_3. \quad (3.6)$$

While (3.6)₁ expresses the requirement that the geodesic curvature of the spatial directrix must equal the curvature of the (flat) referential directrix, (3.6)₂ expresses the additional requirement that, in the spatial configuration, the material surface can manifest only developable shapes.

Dias and Audoly [2] do not explicitly state how a ruling parallel to $\mathbf{Q}$ must be rotated to obtain the corresponding generator $\mathbf{q}$. However, an implicit feature of their description is that the spatial triad $\{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\}$ is obtained by transforming the referential triad $\{\mathbf{D}_1, \mathbf{D}_2, \mathbf{D}_3\}$ by a counterpart rotation

$$\boldsymbol{\Theta} = \sum_{i=1}^3 \mathbf{d}_i \otimes \mathbf{D}_i \quad (3.7)$$

of the rotation $\mathbf{R}$ defined in (2.19). A particular consequence of this is that, analogous to the relation (2.5) between the referential rulings and spatial generators,

$$\mathbf{q} = \boldsymbol{\Theta} \mathbf{Q}. \quad (3.8)$$

In view of (3.8), there is an unmistakable correspondence between (3.1)–(3.2) and (2.3)–(2.6). Indeed, (3.1)–(3.2) can be transformed into (2.3)–(2.6) by making the simple change of variables¹⁰

$$S \rightarrow \alpha, \quad V \rightarrow \frac{\beta}{\sqrt{1+v^2}}, \quad (3.9)$$

and introducing the identifications

$$\left. \begin{aligned} \mathbf{X} &\longleftrightarrow \hat{\mathbf{x}}_0, & \{\mathbf{D}_1, \mathbf{D}_2, \mathbf{D}_3\} &\longleftrightarrow \{\mathbf{l}_1, \mathbf{l}_2, \mathbf{l}_3\}, \\ \mathbf{x} &\longleftrightarrow \hat{\mathbf{y}}_0, & \{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\} &\longleftrightarrow \{\mathbf{e}_1, \mathbf{e}_2, \mathbf{e}_3\}, \end{aligned} \right\} \quad (3.10)$$

⁹The conditions (3.6)₁ and (3.6)₂ are conveyed in constraints (10*) and (11*) of Dias and Audoly [2], where their (7*) is used to express the components of their Darboux vector $\boldsymbol{\omega}$ in terms of the triad $\{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\}$. Notice, however, that (3.6)₁ involves the additional condition $\mathbf{D}'_3 \cdot \mathbf{D}_1 = \kappa_g$ which appears in the penultimate sentence of the second paragraph in Sect. 2.3 of the paper of Dias and Audoly [2].

¹⁰From (3.9)₂, we see that, contrary to what Dias and Audoly [2] assume, it is the product $\sqrt{1+\eta^2}V$, rather than V alone, that measures distance along the referential rulings and spatial generators.

from which it follows that

$$\eta \longleftrightarrow \nu, \quad \kappa_g \longleftrightarrow k_g, \quad \text{and} \quad \omega \longleftrightarrow \delta, \quad (3.11)$$

where ω is the Darboux vector entering (6)–(7*) of Dias and Audoly [2]. On this basis, we see that the developability constraints (3.6)₁ and (3.6)₂ that they impose are implied by (2.32)₂ and (2.30)_{1,3}, respectively. Moreover, from the identity

$$\omega = \omega_1 \mathbf{q} + \kappa_g \mathbf{d}_2, \quad (3.12)$$

which arises from (7*), (10*), and (11*) of Dias and Audoly [2], we deduce that

$$(\omega - \kappa_g \mathbf{d}_2) \times \mathbf{q} = \mathbf{0}. \quad (3.13)$$

Thus, with reference to (3.9)–(3.11), we infer that the condition (2.33) necessary and sufficient to ensure isometry of the deformation is satisfied. Granted that the conditions (3.4) and (3.6) apply, the mapping $\mathbf{Y} \mapsto \mathbf{y}$ determined implicitly by (3.1) and (3.2) therefore satisfies the second of the bullet items entering the characterization of an isometric deformation provided in the final paragraph of Sect. 2.1.

A more direct alternative to the foregoing argument hinges on showing that $\boldsymbol{\Theta}' \mathbf{Q} = \mathbf{0}$ or, equivalently, on demonstrating that the axial vector of the skew tensor $\boldsymbol{\Theta}' \boldsymbol{\Theta}^\top$ is collinear with $\mathbf{q}$. Specifically, calculating $\boldsymbol{\Theta}'$ and $\boldsymbol{\Theta}^\top$ from (3.7), using the orthonormality of the triads $\{\mathbf{D}_1, \mathbf{D}_2, \mathbf{D}_3\}$ and $\{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\}$, the analog

$$\mathbf{d}'_i = \omega \times \mathbf{d}_i, \quad i = 1, 2, 3, \quad (3.14)$$

of (2.22), and the constraints (3.6), we see that

$$\begin{aligned} \boldsymbol{\Theta}' \boldsymbol{\Theta}^\top &= \sum_{i,j=1}^3 (\mathbf{d}'_i \otimes \mathbf{D}_i + \mathbf{d}_i \otimes \mathbf{D}'_i) (\mathbf{D}_j \otimes \mathbf{d}_j) \\ &= \sum_{i=1}^3 \mathbf{d}'_i \otimes \mathbf{d}_i + \sum_{i,j=1}^3 (\mathbf{D}'_i \cdot \mathbf{D}_j) \mathbf{D}_i \otimes \mathbf{D}_j \\ &= \sum_{i=1}^3 (\omega \times \mathbf{d}_i) \otimes \mathbf{d}_i + \kappa_g \sum_{i,j=1}^3 ((\mathbf{D}_2 \times \mathbf{D}_i) \cdot \mathbf{D}_j) \mathbf{d}_i \otimes \mathbf{d}_j \\ &= (\omega \times) \sum_{i=1}^3 \mathbf{d}_i \otimes \mathbf{d}_i + \kappa_g \sum_{i,j=1}^3 (\mathbf{D}_2 \cdot (\mathbf{D}_i \times \mathbf{D}_j)) \mathbf{d}_i \otimes \mathbf{d}_j \\ &= (\omega \times) \sum_{i=1}^3 \mathbf{d}_i \otimes \mathbf{d}_i - \kappa_g (\mathbf{d}_3 \otimes \mathbf{d}_1 - \mathbf{d}_1 \otimes \mathbf{d}_3) \\ &= (\omega - \kappa_g \mathbf{d}_2) \times \\ &= \omega_1 \mathbf{q} \times, \end{aligned} \quad (3.15)$$

from which we deduce that $\text{ax}(\boldsymbol{\Theta}' \boldsymbol{\Theta}^\top) = \omega_1 \mathbf{q}$, confirming thereby that, granted (3.4) and (3.6), the mapping $\mathbf{Y} \mapsto \mathbf{y}$ determined by (3.1) and (3.2) constitutes an isometric deforma-

tion as defined by Chen, Fosdick and Fried [1] and recapitulated in Sect. 2.1 of the present paper.¹¹

4 Importance of the Correspondence Between Curvilinear Coordinate Pairs and Material Points

We next focus on the requirement that each coordinate pair (α, β) corresponds to a unique material point x in $\mathcal{D}$ and, moreover, that the underlying correspondence between coordinate pairs and points provides a complete covering of $\mathcal{D}$.¹² Together, these requirements ensure that the relationship between coordinate pairs and material points is surjective.

4.1 General Considerations

Although Dias and Audoly [2] do not mention this requirement, an appreciation of its importance is evident from their introduction of “edge functions” $V_{\pm}(\eta, S)$. For the particular case in which the reference configuration of the material surface is a rectangular strip, Dias and Audoly [2] choose $V_{\pm}(\eta, S)$ so that the collection of parameter pairs (S, V) covers the region occupied by the material surface in its reference configuration, but that choice restricts the referential rulings to be perpendicular to the referential directrix, which they take to be midway between the long edges of the strip, and consequently limits the parametrizations (3.1) and (3.2) to describe only the bending of a rectangular material strip into a right circular cylindrical form. It does not, for instance, encompass the bending of a rectangular material strip into a helical ribbon coincident with a portion of a right cylindrical surface or into a conical ribbon coincident with a portion of a right circular conical surface, wherein the referential rulings must be inclined relative to the referential directrix (as discussed later in this section).

Returning to the general difficulty raised in the second paragraph of Sect. 3, because Dias and Audoly [2] stipulate that their longitudinal coordinate S satisfies $0 \leq S \leq L$, the measures that they introduce in connection with their edge functions $V_{\pm}(\eta, S)$ rarely suffice to ensure that the collection (3.5) of (S, V) pairs completely covers the planar region occupied by the material surface in the reference configuration and, thus, the curved surface it occupies in the spatial configuration. This difficulty is evident even in the first figure of their paper, a partial counterpart of which appears in our Fig. 1. For the particular directrices, rulings, and generators depicted in that image, two corners of the material surface in the reference configuration go uncovered by rulings and the same is true of the associated corners of the deformed material surface. No element (S, V) of (3.5) can describe a material point in either of those corners. With reference to Fig. 1, this consequence of stipulating that the referential and spatial directrices are loci of material points,¹³ can be rectified by extending the range of the transverse coordinate S beyond $0 \leq S \leq L$, resulting in directrices that include nonmaterial segments outside the boundaries of the material surface. Again making reference to Fig. 1, this must be accompanied by the introduction of corresponding

¹¹The presentation of Dias and Audoly [2] is not convincing as it stands because it emphasizes the differential geometric notion of isometry and does not clearly define the deformation of material points from a reference form to a spatial form and explicitly distinguish and characterize the *deformation* as isometric.

¹²See the first bullet item in the final paragraph of Sect. 2.1.

¹³See Sect. 2.1 and, in particular, the statements in the final sentences of the paragraphs containing (2.3) and (2.6).

Fig. 1 Schematic of (a) a planar material surface and (b) an isometric deformation of that material surface. (Inspired by Dias and Audoly [2, Fig. 1].) In the reference configuration, two corners of the material surface go unruléd unless the referential directrix includes segments that are disjoint from the referential region occupied by the material surface. The corresponding corners of the surface identified with the material surface in its deformed configuration do not possess generators unless the spatial directrix includes corresponding segments that are disjoint from that surface. The segments of the directrices that lie on the material surface and, thus, consist of material points, are indicated in *white*, as are the rulings and generators needed to fully cover the material surface in the reference and deformed configurations. The segments of the directrices that are disjoint from the material surface and, thus, do not contain material points, are indicated in *red*. The rulings and generators that intersect those segments are *red* outside of the referential and spatial manifestations of the material surface and *white* inside those manifestations

rulings and generators that reach from the nonmaterial portions of the directrices into the previously uncovered portions of the material surface. It then remains to correctly delineate the range of V for the rulings and generators corresponding to each point, whether material or nonmaterial, on the directrices.

4.2 Example: Isometric Deformation of a Rectangular Material Strip to a Helical Sector of a Cylindrical Surface

We consider the isometric deformation of a rectangular material strip $\mathcal{D}$ onto a helical sector $\mathcal{S}$ of a right circular cylindrical surface $\mathcal{Y}$ of radius r_0 . To simplify the discussion, we identify the spatial manifestation of $\mathcal{D}$ with $\mathcal{S}$ and refer to $\mathcal{S}$ as a ‘helical ribbon’.

Employing the definitions and notation introduced at the outset of Sect. 2, we identify the reference configuration of the strip with the planar region

$$\mathcal{D} = \left\{ (x_1, x_2) : 0 \leq x_1 \leq \ell, -\frac{w}{2} \leq x_2 \leq \frac{w}{2} \right\} \subset \mathbb{E}^2, \quad (4.1)$$

where x_1 and x_2 serve as rectilinear coordinates for $\mathbb{E}^2$ in the directions of $\mathbf{t}_1$ and $\mathbf{t}_2$. Each ordered pair of (x_1, x_2) in $\mathcal{D}$ then corresponds to a unique material point $\mathbf{x} = \mathbf{o} + x_1\mathbf{t}_1 + x_2\mathbf{t}_2$, where $\mathbf{o}$ denotes the point at the origin. Moreover, we rule $\mathcal{D}$ with a family of parallel straight lines inclined at an angle θ_0 , satisfying

$$0 < \theta_0 < \pi/2 \quad (4.2)$$

Fig. 2 A rectangular material strip $\mathcal{D}$ of length ℓ and width w identified with the planar region $\{(x_1, x_2) : 0 \leq x_1 \leq \ell, -w/2 \leq x_2 \leq w/2\}$ and ruled by a family of *parallel lines* oriented at angle θ_0 satisfying $0 < \theta_0 < \pi/2$ when measured clockwise from the line $x_2 = 0$. The strip is completely covered by the collection $\mathcal{P}$ of (α, β) coordinate pairs defined in (4.7). The curvilinear coordinates (α, β) of a material point $\mathbf{x}$ in $\mathcal{D}$ with rectilinear coordinates (x_1, x_2) are given by (4.4)

if measured counterclockwise from the $\mathbf{t}_1$ direction, and take the referential directrix $\mathcal{C}_0$ to be a closed subinterval of the line $x_2 = 0$, as illustrated in Fig. 2. Aside from rulings that pass through the material points on the midline $\mathcal{C}_0 \cap \mathcal{D}$ of $\mathcal{D}$, there exist rulings that are connected to nonmaterial points on the segments of $\mathcal{C}_0$ that lie on either side of $\mathcal{C}_0 \cap \mathcal{D}$. Without these nonmaterial segments of $\mathcal{C}_0$, it is impossible to ensure that each coordinate pair (α, β) corresponds to a unique material point $\mathbf{x}$ in $\mathcal{D}$ and, moreover, that the underlying correspondence between coordinate pairs and points covers $\mathcal{D}$ completely.

Following Chen, Fosdick and Fried [1, Sect. 7.1], an isometric deformation $\tilde{\mathbf{y}}$ from $\mathcal{D}$ to $\mathcal{S}$ can be expressed as

$$\tilde{\mathbf{y}}(\mathbf{x}) = \hat{\mathbf{y}}_0(\tilde{\alpha}(\mathbf{x})) + \tilde{\beta}(\mathbf{x})\mathbf{t}_3, \quad (4.3)$$

where $\tilde{\alpha}$ and $\tilde{\beta}$ are given, for each $\mathbf{x} = x_i \mathbf{t}_i$ in $\mathcal{D}$ and each θ_0 satisfying (4.2) by

$$\tilde{\alpha}(\mathbf{x}) = x_1 - x_2 \cot \theta_0 \quad \text{and} \quad \tilde{\beta}(\mathbf{x}) = x_2 \csc \theta_0, \quad (4.4)$$

and the parametrization $\hat{\mathbf{y}}_0$ of the spatial directrix $\mathcal{C}$ of $\mathcal{S}$ is defined according to

$$\hat{\mathbf{y}}_0(\alpha) = -r_0 \cos\left(\frac{\alpha \sin \theta_0}{r_0} + \theta_0\right) \mathbf{t}_1 + r_0 \sin\left(\frac{\alpha \sin \theta_0}{r_0} + \theta_0\right) \mathbf{t}_2 + \alpha \cos \theta_0 \mathbf{t}_3. \quad (4.5)$$

The spatial directrix $\mathcal{C}$ of $\mathcal{S}$ is a helix of lead angle $\pi/2 - \theta_0$ and the generators of $\mathcal{S}$ are parallel to the axis of $\mathcal{Y}$. Moreover, $\mathcal{S}$ is right-handed according to the direction of $\mathbf{t}_3$. An image showing the shapes determined by using $\tilde{\mathbf{y}}$ defined through (4.3)–(4.5), for $\theta_0 = \pi/6$ and $\theta_0 = \pi/3$, to isometrically deform a rectangular strip of length $\ell = 5r_0$ and width $w = r_0/2$ onto a right circular cylindrical surface $\mathcal{Y}$ of radius r_0 appear in Fig. 3.

With reference to Fig. 2 and bearing in mind that, by (4.4)₁, $x_1 = \alpha$ if $x_2 = 0$, a trigonometric exercise shows that the referential directrix $\mathcal{C}_0$ is given by

$$\mathcal{C}_0 = \{(x_1, x_2) : -w \cot \theta_0/2 \leq x_1 \leq \ell + w \cot \theta_0/2, x_2 = 0\}. \quad (4.6)$$

Consistent with the statements immediately after (4.2), the referential and spatial directrices must therefore include segments that extend beyond the edges of $\mathcal{D}$. Moreover, the collection of (α, β) coordinate pairs needed to describe each material point $\mathbf{x}$ belonging to $\mathcal{D}$ can be expressed as

$$\mathcal{P} = \left\{ (\alpha, \beta) : -\beta \cos \theta_0 \leq \alpha \leq \ell - \beta \cos \theta_0, -\frac{w \csc \theta_0}{2} \leq \beta \leq \frac{w \csc \theta_0}{2} \right\}. \quad (4.7)$$

Fig. 3 Helical ribbons determined by using $\tilde{\mathbf{y}}$ defined through (4.3)–(4.5), with $\theta_0 = \pi/6$ and $\theta_0 = \pi/3$, to isometrically deform a rectangular strip of length $\ell = 5r_0$ and width $w = r_0/2$ onto a right circular cylinder $\mathcal{Y}$ with axis parallel to $\mathbf{e}_3$ and radius r_0

In the current notation and with an essential correction of their edge functions,¹⁴ the specialization of (3.5) that Dias and Audoly [2] would consider has the form (Fig. 4)

$$\bar{\mathcal{P}} = \left\{ (\alpha, \beta) : 0 \leq \alpha \leq \ell, -\frac{w \csc \theta_0}{2} \leq \beta \leq \frac{w \csc \theta_0}{2} \right\}. \quad (4.8)$$

In view of (4.4), the subset of the (x_1, x_2) -plane covered by $\bar{\mathcal{P}}$ is not $\mathcal{D}$ but rather

$$\bar{\mathcal{D}} = \left\{ (x_1, x_2) : x_2 \cot \theta_0 \leq x_1 \leq \ell + x_2 \cot \theta_0, -\frac{w}{2} \leq x_2 \leq \frac{w}{2} \right\}. \quad (4.9)$$

Hence, $\bar{\mathcal{P}}$ fails to cover the triangular subregions

$$\left\{ (x_1, x_2) : 0 \leq x_1 < x_2 \cot \theta_0, 0 \leq x_2 \leq \frac{w}{2} \right\} \quad (4.10)$$

and

$$\left\{ (x_1, x_2) : \ell + x_2 \cot \theta_0 < x_1 < \ell, -\frac{w}{2} \leq x_2 \leq 0 \right\} \quad (4.11)$$

of $\mathcal{D}$. Additionally, $\bar{\mathcal{P}}$ contains ordered pairs (α, β) that correspond to nonmaterial points in the triangular regions

$$\left\{ (x_1, x_2) : x_1 \leq x_2 \cot \theta_0 < 0, -\frac{w}{2} \leq x_2 \leq 0 \right\} \quad (4.12)$$

¹⁴See the discussion in the second paragraph of Sect. 4. For a helical ribbon with spatial directrix being a helix of lead angle $\pi/2 - \theta_0$, the referential rulings must be inclined at an angle θ_0 when measured counterclockwise from the referential directrix. In the present setting, the edge functions $\pm w/2$ that Dias and Audoly [2] introduce in their discussion of rectangular material strips must therefore be replaced by $\pm w \csc \theta_0/2$.

Fig. 4 The set $\bar{\mathcal{D}}$ of (x_1, x_2) pairs determined by all combinations of (α, β) pairs in the set $\bar{\mathcal{P}}$ defined in (4.8). Material points in the triangular subregions of (4.10) and (4.11) of $\mathcal{D}$ are missing from $\bar{\mathcal{D}}$; moreover, $\bar{\mathcal{D}}$ contains spurious nonmaterial points belonging to triangular regions (4.12) and (4.13) disjoint from $\mathcal{D}$

and

$$\left\{ (x_1, x_2) : \ell < x_1 \leq x_2 \cot \theta_0 < 0, -\frac{w}{2} \leq x_2 \leq 0 \right\}, \quad (4.13)$$

both of which are disjoint from $\mathcal{D}$ but have areas equal to those of the missing regions (4.10) and (4.11) respectively, (Fig. 4). The collection of coordinate pairs in $\bar{\mathcal{P}}$ is therefore not only insufficient to describe all material points in $\mathcal{D}$ but also includes spurious elements that are not needed to describe material points in $\mathcal{D}$.

In the bending of a rectangular material strip into a helical ribbon, we therefore conclude that the parametric description of Dias and Audoly [2] fails to completely cover both the planar subset $\mathcal{D}$ of $\mathbb{E}^2$ occupied by the strip in its reference configuration and the curved surface $\mathcal{S}$ in $\mathbb{E}^3$ identified with the helical ribbon. Moreover, that description includes extraneous coordinate pairs that do not correspond to material points of $\mathcal{D}$. Although we have exposed these drawbacks in the context of a special example, we emphasize that, as discussed in the previous section, their generic nature is evident from Fig. 1 of Dias and Audoly [2].

4.3 Example: Isometric Deformation of a Rectangular Strip to a Sector of a Conical Surface

Now, let us consider the isometric deformation of a rectangular material strip $\mathcal{D}$ of length ℓ and width w to a ribbon coincident with a sector $\mathcal{S}$ of a right circular conical surface $\mathcal{K}$.¹⁵ For convenience, we continue to employ the definitions and notation introduced at the outset of Sect. 2 and, as in our treatment of the isometric deformation of a rectangular material strip to a helical ribbon, we identify the reference configuration of $\mathcal{D}$ with the planar region

$$\mathcal{D} = \left\{ (x_1, x_2) : 0 \leq x_1 \leq \ell, -\frac{w}{2} \leq x_2 \leq \frac{w}{2} \right\} \subset \mathbb{E}^2. \quad (4.14)$$

Additionally, we rule $\mathcal{D}$ with a family of straight lines that intersect at a point outside of $\mathcal{D}$ and take the referential directrix $\mathcal{C}_0$ to be a closed subinterval of the line $x_2 = 0$, as illustrated in Fig. 5. As with the bending of a rectangular material strip into a helical ribbon presented

¹⁵For a complete description of that deformation and derivations of all kinematical objects used in the subsequent calculations, we refer the reader to Chen, Fosdick and Fried [1, Sect. 9].

Fig. 5 The rectangular material strip $\mathcal{D}$ of length ℓ and width w showing some members of the family of intersecting straight lines which become the lines of zero principal curvature of the subset $\mathcal{S}$ of a right circular conical surface $\mathcal{K}$. The corresponding referential rulings of $\mathcal{D}$ are the *white segments* of those *lines*. The referential directrix C_0 is indicated in *red*. The length of a generator of the portion of the right circular conical surface $\mathcal{K}$ that lies above the plane spanned by t_1 and t_2 is denoted by Λ . The curvilinear coordinates $(\alpha, \beta) = (\tilde{\alpha}(x), \tilde{\beta}(x))$ of the material point $x \in \mathcal{D}$ with rectilinear coordinates (x_1, x_2) are given by (4.20)

in Sect. 4.2, there exist not only rulings that pass through material points on the portion $C_0 \cap \mathcal{D}$ of the referential midline C_0 that lies within $\mathcal{D}$ but also rulings that are connected to nonmaterial points belonging to segments of C_0 that lie on either side of $C_0 \cap \mathcal{D}$. Moreover, these nonmaterial segments of C_0 are essential to ensure that each coordinate pair (α, β) corresponds to a unique material point x in $\mathcal{D}$ and that the underlying correspondence between coordinate pairs and points covers $\mathcal{D}$ completely. Emulating our treatment of the example considered in Sect. 4.2, we identify the spatial manifestation of the material surface with the subset $\mathcal{S}$ of $\mathcal{K}$ and refer to $\mathcal{S}$ as a ‘conical ribbon’ (see Fig. 6).

4.3.1 Characterization of the Conical Surface

We assume that $\mathcal{K}$ opens downward from Ht_3 and intersects the (x_1, x_2) -plane to form a circle of radius R , in which case it has tip angle 2φ , with

$$\tan \varphi = \frac{R}{H}, \quad (4.15)$$

and the generators of the portion of $\mathcal{K}$ that is situated above the (x_1, x_2) -plane are of length

$$\Lambda = \sqrt{R^2 + H^2}. \quad (4.16)$$

With reference to Fig. 5, we denote the angle between t_1 and the referential ruling, with orientation $b(0)$, that passes through the left-hand end point $(x_1, x_2) = (0, 0)$ of the midline

Fig. 6 The right circular conical surface $\mathcal{K}$ and the rectangular material strip $\mathcal{D}$ shown attached at $R\iota_1$ and tangent to $\mathcal{K}$ along the generator defined by $\mathbf{a}$. Each point $\mathbf{x}$ of $\mathcal{D}$ is translated by the constant vector $R\iota_1$ and the vector between the origin and that point is then rotated about the point $R\iota_1$ by the linear transformation $\mathbf{Q}_0$ defined in (9.14) of Chen, Fosdick and Fried [1], the effect of which is to map the ruling of $\mathcal{D}$ parallel to $\mathbf{b}(0)$ to the generator of $\mathcal{S}$ parallel to $\mathbf{a} = \mathbf{Q}_0\mathbf{b}(0)$

of $\mathcal{D}$ by θ_0 and without loss of generality require that

$$0 < \theta_0 < \frac{\pi}{2}. \quad (4.17)$$

To ensure that for all lengths $\ell > \Lambda \cos \theta_0$, $\mathcal{S}$ is wrapped smoothly onto $\mathcal{K}$ without intersecting with the tip of $\mathcal{K}$, we restrict the width w of $\mathcal{D}$ to satisfy the inequality $w < 2\Lambda \sin \theta_0$. To be definite in all subsequent calculations, we shall assume that $\ell > \Lambda \cos \theta_0$ so that both of the following inequalities hold:

$$\ell > \Lambda \cos \theta_0 \quad \text{and} \quad w < 2\Lambda \sin \theta_0. \quad (4.18)$$

4.3.2 Characterization of the Deformation

Following Chen, Fosdick and Fried [1, Sect. 9.1], the isometric deformation $\tilde{\mathbf{y}}$ that bends the rectangular strip $\mathcal{D}$ onto the right circular conical surface $\mathcal{K}$ in such a way that the left-hand endpoint of its midline lies on the circle of radius R along which $\mathcal{K}$ intersects the (x_1, x_2) -plane, as depicted in Fig. 6 can be expressed as

$$\tilde{\mathbf{y}}(\mathbf{x}) = \hat{\mathbf{y}}_0(\tilde{\alpha}(\mathbf{x})) - \tilde{\beta}(\mathbf{x})(\sin \varphi(\cos \omega(\tilde{\alpha}(\mathbf{x}))\mathbf{t}_1 - \sin \omega(\tilde{\alpha}(\mathbf{x}))\mathbf{t}_2) - \cos \varphi\mathbf{t}_3), \quad (4.19)$$

where $\tilde{\alpha}$ and $\tilde{\beta}$ are given, for each (x_1, x_2) in $(0, \ell) \times (-w/2, w/2)$ and each θ_0 satisfying (4.17), by

$$\tilde{\alpha}(\mathbf{x}) = \frac{x_1 \sin \theta_0 - x_2 \cos \theta_0}{\sin \theta_0 - x_2/\Lambda} \quad \text{and} \quad \tilde{\beta}(\mathbf{x}) = x_2 \sqrt{1 + \left(\frac{\cos \theta_0 - x_1/\Lambda}{\sin \theta_0 - x_2/\Lambda} \right)^2}, \quad (4.20)$$

and the spatial directrix $\mathcal{C}$ of $\mathcal{S}$ is parametrized by $\hat{\mathbf{y}}_0$ defined according to

$$\begin{aligned} \hat{\mathbf{y}}_0(\alpha) = & \left(R - \int_0^\alpha (\sin \varphi \cos \theta(\zeta) \cos \omega(\zeta) + \sin \theta(\zeta) \sin \omega(\zeta)) d\zeta \right) \mathbf{t}_1 \\ & - \left(\int_0^\alpha (\sin \varphi \cos \theta(\zeta) \sin \omega(\zeta) - \sin \theta(\zeta) \cos \omega(\zeta)) d\zeta \right) \mathbf{t}_2 \\ & + \left(\cos \varphi \int_0^\alpha \cos \theta(\zeta) d\zeta \right) \mathbf{t}_3, \end{aligned} \quad (4.21)$$

in which $\theta(\alpha)$ satisfying

$$\tan \theta(\alpha) = \frac{\sin \theta_0}{\cos \theta_0 - \alpha/\Lambda}, \quad (4.22)$$

is the angle formed by the referential ruling with orientation $\mathbf{b}(\alpha)$ and the orientation $\mathbf{t}_1$ of the referential directrix and ω is related linearly to θ through

$$\omega = (\theta - \theta_0) \csc \varphi. \quad (4.23)$$

4.3.3 Coverage of the Strip by Curvilinear Coordinates

Referring to Fig. 5 and noticing that, by (4.20)₁, $x_1 = \alpha$ if $x_2 = 0$, we find on the basis of trigonometric exercises that the referential directrix $\mathcal{C}_0$ is given by

$$\mathcal{C}_0 = \{(x_1, x_2) : \alpha_- \leq x_1 \leq \alpha_+, x_2 = 0\}, \quad (4.24)$$

where α_- and α_+ are defined according to

$$\alpha_- = -\frac{w \cos \theta_0}{2(\sin \theta_0 - w/2\Lambda)} \quad \text{and} \quad \alpha_+ = \ell + \frac{w(\ell/\Lambda - \cos \theta_0)}{2(\sin \theta_0 - w/2\Lambda)}, \quad (4.25)$$

and, consistent with the restrictions (4.18) on ℓ and w and the observation that $\mathcal{C}_0$ must extend beyond $\mathcal{D}$, obey

$$\alpha_- < 0 \quad \text{and} \quad \ell < \alpha_+. \quad (4.26)$$

We next determine the collection of (α, β) coordinate pairs that belong to each edge of $\mathcal{D}$. For clarity, we label those edges as follows:

$$\left. \begin{aligned} \mathcal{L}_0 &= \{(x_1, x_2) : x_1 = 0, -w/2 \leq x_2 \leq w/2\}, \\ \mathcal{L}_\ell &= \{(x_1, x_2) : x_1 = \ell, -w/2 \leq x_2 \leq w/2\}, \\ \mathcal{L}_\pm &= \{(x_1, x_2) : 0 \leq x_1 \leq \ell, x_2 = \pm w/2\}. \end{aligned} \right\} \quad (4.27)$$

Focusing first on $\mathcal{L}_0$ and $\mathcal{L}_\ell$, we see from (4.20) that the values of the α coordinate at the material points $(x_1, x_2) = (0, -w/2)$ and $(x_1, x_2) = (\ell, -w/2)$ at the lower left and lower right corners of $\mathcal{D}$ are

$$\alpha_0 = \frac{w \cos \theta_0}{2(\sin \theta_0 + w/2\Lambda)} \quad \text{and} \quad \alpha_\ell = \ell - \frac{w(\ell/\Lambda - \cos \theta_0)}{2(\sin \theta_0 + w/2\Lambda)}, \quad (4.28)$$

respectively. Applying the restrictions (4.18) on ℓ and w to (4.28), we see further that α_0 and α_ℓ must be such that

$$0 < \alpha_0 < \alpha_\ell < \ell. \quad (4.29)$$

Since the values of the α coordinate at the material points $(x_1, x_2) = (0, w/2)$ and $(x_1, x_2) = (\ell, w/2)$ at the upper left and right corners of $\mathcal{D}$ are α_- and α_+ , respectively, we thus conclude from (4.20) that $\mathcal{L}_0$ and $\mathcal{L}_\ell$ can be described as

$$\mathcal{L}_0 = \{(\alpha, \beta) : \alpha_- \leq \alpha \leq \alpha_0, \beta = \beta_0(\alpha)\} \quad (4.30)$$

and

$$\mathcal{L}_\ell = \{(\alpha, \beta) : \alpha_\ell \leq \alpha \leq \alpha_+, \beta = \beta_\ell(\alpha)\}, \quad (4.31)$$

where β_0 and β_ℓ are defined according to

$$\beta_0(\alpha) = -\frac{\alpha\rho(\alpha)\sin\theta_0}{\cos\theta_0 - \alpha/\Lambda} \quad \text{and} \quad \beta_\ell(\alpha) = \frac{(\ell - \alpha)\rho(\alpha)\sin\theta_0}{\cos\theta_0 - \alpha/\Lambda}, \quad (4.32)$$

with ρ given by

$$\rho(\alpha) = \sqrt{1 + \left(\frac{\cos\theta_0 - \alpha/\Lambda}{\sin\theta_0}\right)^2}. \quad (4.33)$$

From (4.20), we next see that $\mathcal{L}_\pm$ can be described as

$$\mathcal{L}_\pm = \{(\alpha, \beta) : \alpha_0 \leq \alpha \leq \alpha_\ell, \beta = -\beta_\pm(\alpha)\}, \quad (4.34)$$

where $\beta_\pm$ are defined by

$$\beta_\pm(\alpha) = \pm \frac{w\rho(\alpha)}{2}. \quad (4.35)$$

Each referential ruling connects two edges of $\mathcal{D}$ and lies on a straight line defined by a point $(\alpha, 0)$, with $\alpha_- \leq \alpha < \alpha_+$, on the referential directrix and the nonmaterial point $(x_1, x_2) = (\Lambda \cos \theta_0, \Lambda \sin \theta_0)$ corresponding to the apex of the right circular conical surface $\mathcal{K}$. The point $(\alpha, 0)$ is material if $0 \leq \alpha \leq \ell$ and nonmaterial if $\alpha_- \leq \alpha < 0$ or $\ell < \alpha \leq \alpha_+$. The referential rulings

$$\{(\alpha, \beta) : \alpha = \alpha_0, \beta_-(\alpha_0) \leq \beta \leq \beta_+(\alpha_0)\} \quad (4.36)$$

and

$$\{(\alpha, \beta) : \alpha = \alpha_\ell, \beta_-(\alpha_\ell) \leq \beta \leq \beta_+(\alpha_\ell)\} \quad (4.37)$$

connect the lower left and right corners $(x_1, x_2) = (0, -w/2)$ and $(x_1, x_2) = (\ell, -w/2)$ of $\mathcal{D}$ to the upper edge $\mathcal{L}_+$ of $\mathcal{D}$ and naturally decomposes $\mathcal{D}$ into a union of three subregions, each distinguished by the features of its rulings, as depicted in Fig. 7. This decomposition consists of:¹⁶

¹⁶These subregions are complementary to the extent that their interiors are disjoint.

Fig. 8 Covering of the (x_1, x_2) -plane provided by the set $\mathcal{A}$ of (α, β) pairs defined in (4.41). In addition to missing all material points belonging to the triangular subregions (4.44) and (4.45) of $\mathcal{D}$, $\mathcal{A}$ contains pairs (α, β) that correspond to spurious nonmaterial points in the triangular regions (4.47) and (4.48) disjoint from $\mathcal{D}$

of $\mathcal{T}_0$ and $\mathcal{T}_\ell$ are absent from $\mathcal{A}$; hence, $\mathcal{A}$ fails to cover the triangular subregions

$$\left\{ (x_1, x_2) : 0 \leq x_1 < x_2 \cot \theta_0, 0 < x_2 \leq \frac{w}{2} \right\} \quad (4.44)$$

and

$$\left\{ (x_1, x_2) : \ell - \frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0} x_2 \leq x_1 < \ell, 0 < x_2 \leq \frac{w}{2} \right\} \quad (4.45)$$

of $\mathcal{D}$, as illustrated in Fig. 8. Additionally, since for each choice of α_* satisfying $0 \leq \alpha_* < \alpha_-$ or $\alpha_+ < \alpha_* \leq \ell$, the ruling

$$\{(\alpha, \beta) : \alpha = \alpha_*, \beta_-(\alpha_*) \leq \beta \leq \beta_+(\alpha_*)\} \quad (4.46)$$

extends beyond the boundary of $\mathcal{D}$, $\mathcal{A}$ contains ordered pairs (α, β) that correspond to spurious nonmaterial points in the triangular regions

$$\left\{ (x_1, x_2) : x_2 \cot \theta_0 \leq x_1 < 0, -\frac{w}{2} \leq x_2 < 0 \right\} \quad (4.47)$$

and

$$\left\{ (x_1, x_2) : \ell \leq x_1 < \ell - \frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0} x_2, -\frac{w}{2} \leq x_2 < 0 \right\}, \quad (4.48)$$

both of which are disjoint from $\mathcal{D}$, as illustrated in Fig. 8. Importantly, the areas of the spurious regions (4.47) and (4.48) are the same as those of the missing subregions (4.44) and (4.45), respectively.

5 Bending Energies

In a tradition that traces back at least to Sadowsky's [5, 6] formulation of a variational problem for determining the equilibrated shape of a Möbius band in the absence of external loading, it is common to take the bending energy E stored in bending an unstretchable material surface with planar rectangular reference shape $\mathcal{D}$ in $\mathbb{E}^2$ into a curved surface $\mathcal{S}$ in $\mathbb{E}^3$ to be given by

$$E = 2\mu \int_{\mathcal{S}} H^2(\mathbf{y}) \, d\mathbf{a}(\mathbf{y}), \quad (5.1)$$

where $\mu > 0$ is the bending modulus of the material surface and H and $d\mathbf{a}$ denote the mean curvature and area element of $\mathcal{S}$. In (5.1), E is evaluated in the deformed configuration of the material surface. Alternatively, it is possible to parametrize E using the rectilinear or curvilinear coordinates of the material points of $\mathcal{D}$. For the second of these options, it is however essential to ensure that the limits of integration are determined consistent with the need to ensure that the curvilinear coordinates completely cover $\mathcal{D}$!

To illustrate the importance of the foregoing statement, we next consider three particular examples involving isometric deformations of rectangular material strips.

5.1 Example 1: Bending Energy of a Rectangular Strip Isometrically Deformed to a Ribbon Coincident with a Sector of a Conical Surface

In this first example, we consider the calculation of the stored energy for a rectangular strip that is isometrically deformed into a portion of a right circular conical surface and compare this calculation to those predicted by the related Sadowsky and Wunderlicht functionals.

5.1.1 Rectilinear Parametrization of the Bending Energy

Since the deformation $\tilde{\mathbf{y}}$ defined by (4.19) is isometric, the mean curvature H of the conical ribbon $\mathcal{S}$ is equal to one half the nonvanishing principal curvature of $\mathcal{S}$. Moreover, the area element $d\mathbf{a}(\mathbf{y})$ of $\mathcal{S}$ obeys $d\mathbf{a}(\mathbf{y}) = d\mathbf{a}(\mathbf{x}) = dx_1 dx_2$. Using (9.59) of Chen, Fosdick and Fried [1], which gives the nonvanishing principal curvature of $\mathcal{S}$ as a function of the rectilinear coordinates (x_1, x_2) of the material points in $\mathcal{D}$, we thus find that the bending energy E of $\mathcal{S}$ obtained by specializing (5.1) to the deformation $\tilde{\mathbf{y}}$ defined by (4.19)–(4.23) can be expressed as

$$E = \frac{\mu \cot^2 \varphi}{2\Lambda^2} \int_{-w/2}^{w/2} \int_0^\ell \frac{dx_1 dx_2}{(\cos \theta_0 - x_1/\Lambda)^2 + (\sin \theta_0 - x_2/\Lambda)^2}. \quad (5.2)$$

Evaluating the innermost integral in (5.2), which is elementary for each x_2 in $(-w/2, w/2)$, introducing the change of variables

$$u = \frac{\cos \theta_0}{\sin \theta_0 - x_2/\Lambda}, \quad (5.3)$$

and defining

$$u_{\pm} = \frac{\cos \theta_0}{\sin \theta_0 \mp w/2\Lambda} \quad \text{and} \quad a = \frac{\ell/\Lambda - \cos \theta_0}{\cos \theta_0}, \quad (5.4)$$

we see that

$$E = \frac{\mu \cot^2 \varphi}{2} \int_{u_-}^{u_+} \frac{(\arctan u + \arctan au) du}{u}. \quad (5.5)$$

The integral (5.5) is not elementary but can be expressed in terms of the inverse tangent integral Ti_2 , as defined by (Lewin [10, Chap. II])

$$\text{Ti}_2(\xi) = \int_0^\xi \frac{\arctan u du}{u}. \quad (5.6)$$

Specifically, using (5.6) and recalling the definitions (5.4) of $u_\pm$ and a , we arrive at the representation

$$\begin{aligned} E = \frac{\mu \cot^2 \varphi}{2} & \left(\text{Ti}_2 \left(\frac{\cos \theta_0}{\sin \theta_0 - w/2\Lambda} \right) + \text{Ti}_2 \left(\frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0 - w/2\Lambda} \right) \right. \\ & \left. - \text{Ti}_2 \left(\frac{\cos \theta_0}{\sin \theta_0 + w/2\Lambda} \right) - \text{Ti}_2 \left(\frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0 + w/2\Lambda} \right) \right). \end{aligned} \quad (5.7)$$

Using the definitions (4.25) of $\alpha_\pm$ and (4.28) of α_0 and α_ℓ in (5.4), we see that (5.2) takes the alternative form

$$\begin{aligned} E &= \frac{\mu \cot^2 \varphi}{2} \left(\int_{\cot \theta(\alpha_0)}^{\cot \theta(\alpha_-)} \frac{\arctan u du}{u} + \int_{\cot \theta(\alpha_+)}^{\cot \theta(\alpha_\ell)} \frac{\arctan u du}{u} \right) \\ &= \frac{\mu \cot^2 \varphi}{2} (\text{Ti}_2(\cot \theta(\alpha_-)) - \text{Ti}_2(\cot \theta(\alpha_+)) - \text{Ti}_2(\cot \theta(\alpha_0)) + \text{Ti}_2(\cot \theta(\alpha_\ell))). \end{aligned} \quad (5.8)$$

Modulo the scale factor $\mu \cot^2 \varphi / 2$ determined by the bending modulus μ of the material surface and the opening angle 2φ of the right circular conical surface $\mathcal{K}$, the bending energy E stored in isometrically deforming a rectangular material strip $\mathcal{D}$ of length ℓ and width w into a ribbon $\mathcal{S}$ on $\mathcal{K}$ is therefore completely determined by the angles of the lines formed by the four corners of $\mathcal{D}$ and the point $(x_1, x_2) = (\Lambda \cos \theta_0, \Lambda \sin \theta_0)$ corresponding to the apex of the right circular conical surface $\mathcal{K}$. The alternative representation (5.8) for E can also be derived directly from (5.7) on the basis of (4.25), (4.28), the periodicity of the cotangent, and the observation that Ti_2 is an odd function of its argument.

5.1.2 Curvilinear Parametrization of the Bending Energy

Using (9.58) of Chen, Fosdick and Fried [1], which gives the nonvanishing principal curvature of $\mathcal{S}$, we obtain the mean curvature of $\mathcal{S}$ as a function $\hat{H}$ of the curvilinear coordinates (α, β) of the form

$$\hat{H}(\alpha, \beta) = -\frac{\cot \varphi}{2\Lambda \sin \theta_0 (\rho(\alpha) - \beta/\Lambda \sin \theta_0)}, \quad (5.9)$$

where ρ is defined as in (4.33). Additionally, referring to (9.59) of Chen, Fosdick and Fried [1], the Jacobian $\hat{j}$ of the transformation underlying the correspondence between material points in $\mathcal{D}$ and curvilinear coordinates (α, β) is given by

$$\hat{j}(\alpha, \beta) = \frac{\rho(\alpha) - \beta/\Lambda \sin \theta_0}{\rho^2(\alpha)}. \quad (5.10)$$

Taking advantage of the decomposition of $\mathcal{D}$ into a trapezoidal subregion and two triangular subregions described in Sect. 4.3.3 and invoking the definitions (4.38), (4.39), and (4.40) of the corresponding collections $\mathcal{T}_c$, $\mathcal{T}_0$, and $\mathcal{T}_\ell$ of coordinate pairs (α, β) , we see that the bending energy E of $\mathcal{S}$ can also be expressed as

$$\begin{aligned} E = & \frac{\mu \cot^2 \varphi}{2\Lambda^2 \sin^2 \theta_0} \int_{\alpha_-}^{\alpha_0} \frac{1}{\rho^2(\alpha)} \left(\int_{\beta_0(\alpha)}^{\beta_+(\alpha)} \frac{d\beta}{\rho(\alpha) - \beta/\Lambda \sin \theta_0} \right) d\alpha \\ & + \frac{\mu \cot^2 \varphi}{2\Lambda^2 \sin^2 \theta_0} \int_{\alpha_0}^{\alpha_\ell} \frac{1}{\rho^2(\alpha)} \left(\int_{\beta_-(\alpha)}^{\beta_+(\alpha)} \frac{d\beta}{\rho(\alpha) - \beta/\Lambda \sin \theta_0} \right) d\alpha \\ & + \frac{\mu \cot^2 \varphi}{2\Lambda^2 \sin^2 \theta_0} \int_{\alpha_\ell}^{\alpha_+} \frac{1}{\rho^2(\alpha)} \left(\int_{\beta_\ell(\alpha)}^{\beta_+(\alpha)} \frac{d\beta}{\rho(\alpha) - \beta/\Lambda \sin \theta_0} \right) d\alpha, \end{aligned} \quad (5.11)$$

where $\alpha_\pm$, α_0 , α_ℓ , β_0 , β_ℓ , and $\beta_\pm$ are defined by (4.25), (4.28)₁, (4.28)₂, (4.32)₁, (4.32)₂, and (4.35), respectively. Each of the innermost integrals in (5.11) can be evaluated in closed form and identities connecting the integrals over α that remain can be expressed in terms of the dilogarithm Li_2 (Lewin [10, Chap. I]). A well-established identity (Lewin [10, equation (2.3)]) connecting Li_2 to the inverse tangent integral Ti_2 can then be used to recover (5.7) or, equivalently, (5.8) from (5.11).

5.1.3 Wunderlich's Functional

Wunderlich's [3, 4] functional is a dimensionally reduced version of the bending energy E defined in (5.1), with its domain of integration being the midline of the spatial manifestation $\mathcal{S}$ of the material surface. We next specialize that functional to the problem of isometrically deforming the rectangular material strip $\mathcal{D}$ to the conical ribbon $\mathcal{S}$ and compare the resulting expression with the bending energy E of $\mathcal{S}$ in either the form (5.7) determined by the rectilinear parametrization or its equivalent alternative form (5.11) determined by the curvilinear parametrization.

With reference to the material in the first two paragraphs on page 279 of Wunderlich's [3] original paper, we gather identifications of the form

$$L \longleftrightarrow \ell, \quad b \longleftrightarrow \frac{w}{2}, \quad \vartheta \longleftrightarrow \theta, \quad \frac{\tau}{\kappa} \longleftrightarrow \cot \theta, \quad \text{and} \quad V \longleftrightarrow \frac{\sin^2 \theta_0}{\theta'} \quad (5.12)$$

where κ and τ denote the curvature and torsion of $\mathcal{C}$. In particular, from (4.21), we see that κ^2 takes the form

$$\kappa^2 = |\tilde{\mathbf{y}}_0''|^2 = \frac{\cot^2 \varphi}{\Lambda^2 \sin^2 \theta_0} \frac{1}{\rho^6}, \quad (5.13)$$

with ρ as defined in (4.33). Moreover, from (9.52)₂ of Chen, Fosdick and Fried [1], the relation (4.22) defining the angle θ between the referential rulings and the referential director $\mathcal{C}_0$, and (4.33), we see that

$$\sin^2 \theta = \frac{1}{\rho^2} \quad \text{and} \quad \theta' = \frac{1}{\Lambda \sin \theta_0} \frac{1}{\rho^2}. \quad (5.14)$$

Multiplying (6) of Wunderlich [3, 4] by a factor of $\mu/2$ to ensure a meaningful comparison to E and invoking (5.12)–(5.29), we thus find that, for the conical ribbon $\mathcal{S}$, Wunderlich's [3, 4] functional specializes to¹⁷

$$\begin{aligned} E_W &= \frac{\mu}{2} \int_0^\ell \frac{\kappa^2(\alpha)(1 + \cot^2 \theta(\alpha))^2 \sin^2 \theta(\alpha)}{\theta'(\alpha)} \log \left(\frac{\sin^2 \theta(\alpha) + w\theta'(\alpha)/2}{\sin^2 \theta(\alpha) - w\theta'(\alpha)/2} \right) d\alpha \\ &= \frac{\mu \cot^2 \varphi}{2\Lambda \sin \theta_0} \log \left(\frac{\sin \theta_0 + w/2\Lambda}{\sin \theta_0 - w/2\Lambda} \right) \int_0^\ell \frac{d\alpha}{\rho^2(\alpha)}. \end{aligned} \quad (5.15)$$

Recalling the definition (4.33) of ρ , we notice that the remaining integral in (5.15) is elementary and, since $\arctan(\cot \theta_0) = \pi/2 - \theta_0$ for $0 < \theta < \pi$, we arrive at the closed-form representation

$$E_W = \frac{\mu \cot^2 \varphi}{2} \left(\frac{\pi}{2} - \theta_0 + \arctan \left(\frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0} \right) \right) \log \left(\frac{\sin \theta_0 + w/2\Lambda}{\sin \theta_0 - w/2\Lambda} \right). \quad (5.16)$$

Since (5.16) is predicated upon integrating over the collection of (α, β) coordinate pairs that belong to $\mathcal{A}$ instead of the union $\mathcal{T}_- \cup \mathcal{T}_c \cup \mathcal{T}_+$ that ensures complete coverage of $\mathcal{D}$, we anticipate that E_W will underestimate the correct bending energy, as we will see in Sect. 5.1.5.

5.1.4 Sadowsky's Functional

Prior to Wunderlich [3, 4], Sadowsky's [5, 6] derived a dimensional reduced version of the general bending energy E defined in (5.1) under the assumption, $w \ll \ell$, that the width of the rectangular material strip is negligible in comparison to its length. Using (5.12)₄ and (5.13), the Sadowsky [5, 6] functional takes the form

$$E_S = \frac{\mu}{2} \int_0^\ell \kappa^2(\alpha) (1 + \cot^2(\alpha))^2 d\alpha, \quad (5.17)$$

where, to facilitate comparisons with E and E_W , we have again introduced a factor of $\mu/2$. Following steps analogous to those leading from (5.15) to (5.16), we find that, for the problem of isometrically deforming the rectangular material strip $\mathcal{D}$ to the conical ribbon $\mathcal{S}$, Sadowsky's [5, 6] functional takes the form

$$E_S = \frac{\mu \ell \cot^2 \varphi}{\Lambda \sin \theta_0} \left(\frac{\pi}{2} - \theta_0 + \arctan \left(\frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0} \right) \right) \frac{w}{2\ell}. \quad (5.18)$$

¹⁷It is also possible to directly obtain (5.16) from the version of Wunderlich's [3, 4] functional that appears in (30*) of Dias and Audoly [2]. To do that, we require the appropriate specializations of their quantities η and ω_1 . With this in mind, we recall from (3.11)₁ that the quantity ν introduced in (2.2) is equal to η of Dias and Audoly [2]. Since $\nu = \cot \theta$ for the deformation $\tilde{\mathbf{y}}$ of $\mathcal{D}$ to $\mathcal{S}$ defined in (4.19), we thus see from (4.22) that η of Dias and Audoly [2] is given by

$$\eta = \cot \theta. \quad (\dagger)$$

From (3.11)₂, we find moreover that our Darboux vector δ is equal to the Darboux vector ω of Dias and Audoly [2]. Thus, by (2.30)₁, (4.23), and (9.22) of Chen, Fosdick and Fried [1], we determine that ω_1 of Dias and Audoly [2] is given by

$$\omega_1 = \frac{p}{\sqrt{1 + \eta^2}} = \frac{\cot \varphi}{\Lambda \sin \theta_0} \frac{1}{\sqrt{1 + \eta^2}} \frac{1}{\rho^2}. \quad (\ddagger)$$

Using $(\dagger)$ and $(\ddagger)$ in (30*) of Dias and Audoly [2], we obtain (5.15) and, therefore, upon integration with respect to α , arrive at (5.16).

Fig. 9 Covering of the coordinate pairs (α, β) on S which represent the collection $\mathcal{A}$ defined in (4.41). Triangular-like regions of S above the midline of S are not covered and triangular-like regions on K below the midline of S , not part of S , are covered. The areas of these regions are equal, but their associated bending energies are not

Granted that Λ is of the same order as ℓ , (5.18) can also be obtained as the leading-order term in the Taylor expansion, about $w/2\ell = 0$, of either the rectilinear parametrization (5.2) of E or the closed-form expression (5.16) for E_W .

Like Wunderlich's [3, 4] functional (5.15), Sadowsky's [5, 6] functional (5.17) involves integrating only over the material portion of the spatial directrix $\mathcal{C}_0$, namely the midline of S . However, granted that $w \ll \ell$, the portions of the directrices that extend beyond the strip and its image are of negligible length. In this sense, the range of integration in (5.17) is consistent with the hypothesis $w \ll \ell$. An analogous statement does not apply to Wunderlich's [3, 4] functional, which is meant to provide an accurate approximation for the bending energy in situations where w need not be infinitesimal small in comparison to ℓ . We anticipate that, for $w \ll \ell$, the error incurred by using Sadowsky's [5, 6] functional should be comparable to that incurred by using Wunderlich's [3, 4] functional. Granted that these observations are accurate, Wunderlich's [3, 4] functional would appear to be of negligible utility. For $w \ll \ell$, Sadowsky's [5, 6] functional should suffice and otherwise—the error associated with failing to completely cover $\mathcal{D}$ —neither of the functionals in question is generally fit to provide a good approximation to the bending energy.¹⁸

¹⁸If the nonvanishing principal curvature of S is uniform along the generators, so that S lies on a cylindrical surface, then Wunderlich's [3, 4] functional correctly determines the bending energy. However, this exceptional case is of little or no interest.

5.1.5 Bounds on the Wunderlich and Sadowsky Functionals

We now demonstrate that E_W and E_S given by (5.16) and (5.18) obey the bounds

$$0 < E_S < E_W < E, \quad (5.19)$$

with E being the bending energy defined in (5.2) or, equivalently, (5.11). The upper bounds on E_S and E_W in (5.19) are consistent with intuitive expectations. It seems plausible that a finite contribution to E_W is lost passing from E_W to E_S under the assumption that $w \ll \ell$, in which case the bound $E_S < E_W$ should follow. The rationale in support of the bound $E_W < E$ hinges on the observation that E_W involves integration not over the entire referential directrix $\mathcal{C}_0$ but, rather, integration only over the portion of $\mathcal{C}_0$ that coincides with the midline of $\mathcal{D}$. More precisely, E_W results from integrating over a collection $\mathcal{A}$ of (α, β) coordinate pairs that fails to completely cover $\mathcal{D}$ while the triangular corners of $\mathcal{D}$ that $\mathcal{A}$ fails to cover are above the referential directrix and the spurious triangular regions outside of $\mathcal{D}$ that $\mathcal{A}$ covers instead are below the referential directrix (Fig. 9). Since the areas of the missing and spurious triangular regions are identical and the curvature of the conical surface $\mathcal{K}$ on any generator increases monotonically as β increases, we anticipate that E_W should underestimate E .

The lower and upper bounds $0 < E_S$ and $E_S < E_W$ on E_S follow directly from the consequences

$$\theta_0 < \frac{\pi}{2} + \arctan\left(\frac{\ell/\Lambda - \cos\theta_0}{\sin\theta_0}\right) \quad (5.20)$$

and

$$\frac{\ell}{2\Lambda \sin\theta_0} \frac{w}{\ell} = \frac{w}{2\Lambda \sin\theta_0} < \log\left(\frac{1 + w/2\Lambda \sin\theta_0}{1 - w/2\Lambda \sin\theta_0}\right) \quad (5.21)$$

of the inequalities $0 < \theta_0 < \pi/2$, $\ell > \Lambda \cos\theta_0$, and $w < 2\Lambda \sin\theta_0$, recorded in (4.17), (4.18)₁, and (4.18)₂.

Toward verifying the strict upper bound $E_W < E$, it is first convenient to express E_W as an iterated integral over α and β . Since, by the definitions (4.33) and (4.35) of ρ and $\beta_{\pm}$,

$$\rho(\alpha) - \frac{\beta_{\pm}(\alpha)}{\Lambda \sin\theta_0} = \frac{\rho(\alpha)}{\sin\theta_0} \left(\sin\theta \mp \frac{w}{2\Lambda} \right); \quad (5.22)$$

we consequently see that

$$\frac{1}{\Lambda \sin\theta} \int_{\beta_-(\alpha)}^{\beta_+(\alpha)} \frac{d\beta}{\rho(\alpha) - \beta/\Lambda \sin\theta_0} = \log\left(\frac{\sin\theta_0 + w/2\Lambda}{\sin\theta_0 - w/2\Lambda}\right) \quad (5.23)$$

and, thus, notice from (5.15) that E_W can also be expressed as a double integral:

$$E_W = \frac{\mu \cot^2 \varphi}{2\Lambda^2 \sin^2 \theta_0} \int_0^\ell \frac{1}{\rho^2(\alpha)} \left(\int_{\beta_-(\alpha)}^{\beta_+(\alpha)} \frac{d\beta}{\rho(\alpha) - \beta/\Lambda \sin\theta_0} \right) d\alpha. \quad (5.24)$$

Comparing (5.24) to the curvilinear parametrization (5.11) of E , we recognize that E_W and each of the terms on the right-hand side of (5.11) share the same essential structure. Moreover, E_W differs from the middle term on the right-hand side of (5.11) only by the limits of integration of the outermost integral.

To simplify the study of the difference $E - E_W$, it is convenient to introduce U defined by

$$U(A_1, A_2; B_1, B_2) = \frac{\mu \cot^2 \varphi}{2\Lambda^2 \sin^2 \theta_0} \int_{A_1}^{A_2} \frac{1}{\rho^2(\alpha)} \left(\int_{B_1(\alpha)}^{B_2(\alpha)} \frac{d\beta}{\rho(\alpha) - \beta/\Lambda \sin \theta_0} \right) d\alpha. \quad (5.25)$$

Thus, using (5.25) in (5.11) and (5.24), we obtain succinct expressions,

$$E = U(\alpha_-, \alpha_0; \beta_0, \beta_+) + U(\alpha_0, \alpha_\ell; \beta_-, \beta_+) + U(\alpha_\ell, \alpha_+; \beta_\ell, \beta_+) \quad (5.26)$$

and

$$E_W = U(0, \ell; \beta_-, \beta_+) \quad (5.27)$$

for the bending energy and the Wunderlich [3, 4] functional of the ribbon $\mathcal{S}$ obtained by isometrically deforming the rectangular strip $\mathcal{D}$ to conform to the right circular conical surface $\mathcal{K}$. Referring to the definitions (4.38), (4.39), and (4.40) of $\mathcal{T}_c$, $\mathcal{T}_0$, and $\mathcal{T}_\ell$, we find that the difference $E - E_W$ can be expressed as

$$\begin{aligned} E - E_W &= U(\alpha_-, \alpha_0; \beta_0, \beta_+) - U(0, \alpha_0; \beta_-, \beta_+) + U(\alpha_\ell, \alpha_+; \beta_\ell, \beta_+) - U(\alpha_\ell, \ell; \beta_-, \beta_+) \\ &= U(\alpha_-, 0; \beta_0, \beta_+) - U(0, \alpha_0; \beta_-, \beta_0) + U(\ell, \alpha_+; \beta_\ell, \beta_+) - U(\alpha_\ell, \ell; \beta_-, \beta_\ell). \end{aligned} \quad (5.28)$$

To make further progress, we employ changes of variables that convert each integral contributing to the second expression for $E - E_W$ in (5.28) to an integral over the rectilinear coordinates (x_1, x_2) . Toward this, we first introduce $\tilde{U}$, $\hat{U}$, and $\check{U}$ defined by

$$\tilde{U}(x_1^-, x_1^+; x_2^-, x_2^+) = \frac{\mu \cot^2 \varphi}{2L^2} \int_{x_1^-}^{x_1^+} \int_{x_2^-}^{x_2^+} \frac{dx_2 dx_1}{(\cos \theta_0 - x_1/\Lambda)^2 + (\sin \theta_0 - x_2/\Lambda)^2}, \quad (5.29)$$

$$\hat{U}(x_1^-, x_1^+; x_2^-, x_2^+) = \frac{\mu \cot^2 \varphi}{2L^2} \int_{x_1^-}^{x_1^+} \int_{x_2^-}^{x_2^+} \frac{dx_2 dx_1}{(\cos \theta_0 + x_1/\Lambda)^2 + (\sin \theta_0 + x_2/\Lambda)^2}, \quad (5.30)$$

and

$$\begin{aligned} \check{U}(x_1^-, x_1^+; x_2^-, x_2^+) \\ = \frac{\mu \cot^2 \varphi}{2L^2} \int_{x_1^-}^{x_1^+} \int_{x_2^-}^{x_2^+} \frac{dx_2 dx_1}{(\ell/\Lambda - \cos \theta_0 + (\ell - x_1)/\Lambda)^2 + (\sin \theta_0 + x_2/\Lambda)^2}. \end{aligned} \quad (5.31)$$

Next, recalling the relations (4.20) determining the curvilinear coordinates (α, β) corresponding to the rectilinear coordinates (x_1, x_2) of each material point $\mathbf{x}$ in $\mathcal{D}$, we write, with a slight abuse of notation,

$$\alpha = \tilde{\alpha}(\mathbf{x}) = \tilde{\alpha}(x_1, x_2), \quad \beta = \tilde{\beta}(\mathbf{x}) = \tilde{\beta}(x_1, x_2). \quad (5.32)$$

With the change of variables $(\alpha, \beta) \rightarrow (\tilde{\alpha}(x_1, x_2), \tilde{\beta}(x_1, x_2))$, the first and third terms in the ultimate expression for $E - E_W$ on the right-hand side of (5.28) transform to

$$U(\alpha_-, 0, \beta_0, \beta_+) = \tilde{U}(0, w \cot \theta_0/2; x_1 \tan \theta_0, w/2) \quad (5.33)$$

and

$$U(\ell, \alpha_+, \beta_\ell, \beta_+) = \tilde{U}(\ell_-, \ell; \zeta(x_1), w/2), \quad (5.34)$$

where ℓ_- and ζ are defined by

$$\ell_- = \ell - \frac{w(\ell/\Lambda - \cos \theta_0)}{2 \sin \theta_0} \quad \text{and} \quad \zeta(x_1) = \frac{(\ell - x_1) \sin \theta_0}{\ell/\Lambda - \cos \theta_0}. \quad (5.35)$$

Similarly, with the change of variables $(\alpha, \beta) \rightarrow (\tilde{\alpha}(-x_1, -x_2), \tilde{\beta}(-x_1, -x_2))$, the second term in the ultimate expression for $E - E_W$ on the right-hand side of (5.28) transforms to

$$U(0, \alpha_0; \beta_-, \beta_0) = \hat{U}(0, w \cot \theta_0/2; x_1 \tan \theta_0, w/2), \quad (5.36)$$

while, with the change of variables $(\alpha, \beta) \rightarrow (\tilde{\alpha}(2\ell - x_1, -x_2), \tilde{\beta}(2\ell - x_1, -x_2))$, the final term in the ultimate expression for $E - E_W$ on right-hand side of (5.28) transforms to

$$U(\alpha_\ell, \ell; \beta_-, \beta_\ell) = \check{U}(\ell_-, \ell; \zeta(x_1), w/2). \quad (5.37)$$

Next, since, by (4.18)₂, $x_1 \tan \theta_0 \leq w/2$ if $0 < x_2 < w \cot \theta_0/2$ and

$$(\cos \theta_0 + x_1/\Lambda)^2 + (\sin \theta_0 + x_2/\Lambda)^2 > (\cos \theta_0 - x_1/\Lambda)^2 + (\sin \theta_0 - x_2/\Lambda)^2 \quad (5.38)$$

for any choice of (x_1, x_2) in the region of integration of $\hat{U}(0, w \cot \theta_0/2; x_1 \tan \theta_0, w/2)$ and $\bar{U}(0, w \cot \theta_0/2; x_1 \tan \theta_0, w/2)$, we find that

$$\begin{aligned} & U(\alpha_-, 0, \beta_0, \beta_+) - U(0, \alpha_0; \beta_-, \beta_0) \\ &= \bar{U}(0, w \cot \theta_0/2; x_1 \tan \theta_0, w/2) - \hat{U}(0, w \cot \theta_0/2; x_1 \tan \theta_0, w/2) > 0. \end{aligned} \quad (5.39)$$

Similarly, since, by (4.18), $\zeta(x_1) \leq w/2$ for $\ell_- < x_1 < \ell$ and

$$\begin{aligned} & (\ell/\Lambda - \cos \theta_0 + (\ell - x_1)/\Lambda)^2 + (\sin \theta_0 + x_2/\Lambda)^2 \\ & > (\cos \theta_0 - x_1/\Lambda)^2 + (\sin \theta_0 - x_2/\Lambda)^2 \end{aligned} \quad (5.40)$$

for any choice of (x_1, x_2) in the region of integration for $\check{U}(\ell_-, \ell; \zeta(x_1), w/2)$ and $\bar{U}(\ell_-, \ell; \zeta(x_1), w/2)$, we find that

$$\begin{aligned} & U(\ell, \alpha_+, \beta_\ell, \beta_+) - U(\alpha_\ell, \ell; \beta_-, \beta_\ell) \\ &= \bar{U}(\ell_-, \ell; \zeta(x_1), w/2) - \check{U}(\ell_-, \ell; \zeta(x_1), w/2) > 0. \end{aligned} \quad (5.41)$$

Finally, using (5.39) and (5.41), we conclude that the second inequality, namely $E_W < E$, in (5.19) holds.

Our analysis demonstrates that Wunderlich's [3, 4] functional E_W underestimates the bending energy E of a conical ribbon $\mathcal{S}$ obtained by applying the isometric deformation $\tilde{\mathbf{y}}$ defined in (4.19) to the rectangular material strip $\mathcal{D}$ for all admissible choices of the length and width, ℓ and w , of $\mathcal{D}$, the angle θ_0 determining the inclination of the referential ruling that emanates from the left-hand endpoint of the midline of the strip, and the length Λ of the generator of the portion of the right circular conical surface $\mathcal{K}$ that lies above the (x_1, x_2) -plane. Additionally, Sadowsky's [5, 6] functional underestimates Wunderlich's [3, 4] functional in an analogous fashion.

5.1.6 Illustrative Comparisons

To highlight the implications of the bounds established in Sect. 5.1.5, we restrict attention to situations where the isometric deformation of $\mathcal{D}$ to $\mathcal{S}$ is such that the distance Λ between the left-hand endpoint $(x_1, x_2) = (0, 0)$ of the midline of $\mathcal{S}$ and the apex $(x_1, x_2) = (\Lambda \cos \theta_0, \Lambda \sin \theta_0)$ of $\mathcal{K}$ is comparable to the length ℓ of $\mathcal{D}$, so that

$$\frac{\ell}{\Lambda} = O(1). \quad (5.42)$$

Granted this provision, we first determine how E and E_S behave for $w \ll \ell$ and $w \rightarrow 2\Lambda \sin \theta_0$ corresponding, respectively, to situations in which the width of $\mathcal{D}$ is significantly smaller than its length ℓ and in which the width of $\mathcal{D}$ approaches the maximum value dictated by (4.18)₂, in which case a point on the upper edge of $\mathcal{S}$ closely approaches the apex of $\mathcal{K}$.

Treating E and E_W as functions of $w/2\ell$, we compute their Taylor expansions about $w/2\ell = 0$. Truncating these expansions at $O((w/2\ell)^5)$ and invoking the expression (5.18) for E_S , we obtain

$$\frac{2(E - E_S)}{\mu \cot^2 \varphi} \sim (f(\ell/\Lambda, \theta_0) + g(\ell/\Lambda, \theta_0)) \left(\frac{w}{2\ell} \right)^3 \quad (5.43)$$

and

$$\frac{2(E_W - E_S)}{\mu \cot^2 \varphi} \sim g(\ell/\Lambda, \theta_0) \left(\frac{w}{2\ell} \right)^3, \quad (5.44)$$

with f and g defined according to

$$\begin{aligned} f(\ell/\Lambda, \theta_0) = \frac{\ell^3}{3\Lambda^3 \sin^3 \theta_0} & \left(\sin 2\theta_0 + \frac{2(\ell/\Lambda - \cos \theta_0) \sin \theta_0}{(\ell/\Lambda - \cos \theta_0)^2 + \sin^2 \theta_0} \right. \\ & \left. - \sin \theta_0 \cos^3 \theta_0 - \frac{(\ell/\Lambda - \cos \theta_0)^3 \sin \theta_0}{((\ell/\Lambda - \cos \theta_0)^2 + \sin^2 \theta_0)^2} \right) \end{aligned} \quad (5.45)$$

and

$$g(\ell/\Lambda, \theta_0) = \frac{\ell^3}{3\Lambda^3 \sin^3 \theta_0} \left(\frac{\pi}{2} - \theta_0 + \arctan \left(\frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0} \right) \right). \quad (5.46)$$

Eliminating E_S between (5.43) and (5.44), we also see that

$$\frac{2(E - E_W)}{\mu \cot^2 \varphi} \sim f(\ell/\Lambda, \theta_0) \left(\frac{w}{2\ell} \right)^3. \quad (5.47)$$

We therefore infer from (5.43), (5.44), and (5.47) that, for $w \ll \ell$, each of the differences $E - E_W$, $E - E_S$, and $E_W - E_S$ increases with the cube of $w/2\ell$. Although the expansions of E and E_W about $w/2\ell = 0$ have the same leading-order term—the Sadowsky [5, 6] functional E_S , we also see from (5.47) that the coefficients of the cubic terms in those expansions differ. We may use trigonometric identities in conjunction with the restrictions (4.17) and (4.18)₁ on θ_0 and ℓ/Λ to show that the coefficient $f(\ell/\Lambda, \theta_0)$ defined in (5.45) is positive and thus, with reference to (5.47), confirm that, consistent with the bound $E_W < E$ established in Sect. 5.1.5, $E > E_W$ for $w \ll \ell$.

Next, to obtain information about the behavior of E as $w \rightarrow 2\Lambda \sin \theta_0$, we apply the asymptotic relation (Lewin [10, equation (2.6)])

$$\mathrm{Ti}_2(\xi) \sim \frac{\pi}{2} \log \xi \quad \text{as} \quad \xi \rightarrow +\infty \quad (5.48)$$

to (5.7) and thereby deduce that

$$E \sim \frac{\mu\pi \cot^2 \varphi}{2} \log \left(\frac{\sin \theta_0 + w/2\Lambda}{\sin \theta_0 - w/2\Lambda} \right) \quad \text{as} \quad w \rightarrow 2\Lambda \sin \theta_0. \quad (5.49)$$

Specializing the expression (5.16) for E_W to incorporate the assumption (5.42) and recalling, from (4.17) and (4.18)₁, that $0 < \theta_0 < \pi/2$ and $\Lambda \cos \theta_0 < \ell$, we see that

$$\frac{\pi}{2} - \theta_0 + \arctan \left(\frac{\ell/\Lambda - \cos \theta_0}{\sin \theta_0} \right) < \pi \quad (5.50)$$

and thus that E and E_W exhibit identical divergent behavior as $w \rightarrow 2\ell \sin \theta_0$. However, consistent with the bound $E_W < E$ established in Sect. 5.1.5, E_W underestimates E even for values of w less than but approaching $2\Lambda \sin \theta_0$. Since Sadowsky's [5, 6] functional is meaningful only for $w \ll \ell$, it would be unreasonable to expect that E_S would share the behavior of E and E_W on passing to the limit $w \rightarrow 2\ell \sin \theta_0$. Indeed, the value of E_S as $w \rightarrow 2\ell \sin \theta_0$ is finite.

For illustrative purposes, we hereafter confine attention to the particular situation in which the generators of the portion of the right circular conical surface $\mathcal{K}$ that lie above the plane spanned by $\mathbf{t}_i$ and $\mathbf{t}_2$ and the rectangular material strip $\mathcal{D}$ are of equal length, so that

$$\Lambda = \ell. \quad (5.51)$$

Granted that (5.51) holds and recalling again from (4.17) that $0 < \theta_0 < \pi/2$, we notice that the restrictions (4.18) involving ℓ , w , and Λ simplify to a single requirement:

$$\frac{w}{2\ell} < \sin \theta_0. \quad (5.52)$$

Using (5.51) in the expressions (5.7) and (5.16) for E and E_W , we thus have that

$$\begin{aligned} \frac{2(E - E_W)}{\mu \cot^2 \varphi} &= \mathrm{Ti}_2 \left(\frac{\cos \theta_0}{\sin \theta_0 - w/2\ell} \right) + \mathrm{Ti}_2 \left(\frac{1 - \cos \theta_0}{\sin \theta_0 - w/2\ell} \right) - \mathrm{Ti}_2 \left(\frac{\cos \theta_0}{\sin \theta_0 + w/2\ell} \right) \\ &\quad - \mathrm{Ti}_2 \left(\frac{1 - \cos \theta_0}{\sin \theta_0 + w/2\ell} \right) - \frac{\pi - \theta_0}{2} \log \left(\frac{\sin \theta_0 + w/2\ell}{\sin \theta_0 - w/2\ell} \right). \end{aligned} \quad (5.53)$$

Similarly, using (5.51) in the expressions (5.16) and (5.18) for E_W and E_S while bearing in mind that $\arctan(\cot \theta_0) = \theta_0/2$ for $0 < \theta_0 < \pi$, we have that

$$\frac{E_W - E_S}{\mu \cot^2 \varphi} = \frac{\pi - \theta_0}{2} \left(\log \left(\frac{\sin \theta_0 + w/2\ell}{\sin \theta_0 - w/2\ell} \right) - \frac{w}{\ell} \right). \quad (5.54)$$

Taking note of the asymptotic expression (5.47) for the small $w/2\ell$ behavior of $2(E - E_W)/\mu \cot^2 \varphi$ and its precursor (5.44) for $2(E_W - E_S)/\mu \cot^2 \varphi$, we consider the dimensionless measures

$$I = \frac{2(E - E_W)}{\mu f(1, \theta_0) \cot^2 \varphi} \quad \text{and} \quad J = \frac{E_W - E_S}{\mu g(1, \theta_0) \cot^2 \varphi} \quad (5.55)$$

Fig. 10 Plots of the scaled absolute error $I = 2(E - E_W)/\mu f(\theta_0, 1) \cot^2 \varphi$ for a rectangular material strip $\mathcal{D}$, of length ℓ and width w , isometrically deformed to coincide with a sector $\mathcal{S}$ of a right circular conical surface $\mathcal{K}$, with tip angle 2φ which opens downward from $H\mathbf{t}_3$ and intersects the (x_1, x_2) -plane to form a circle of radius $R = \ell \csc \varphi$, for $0 < w/2\ell < \sin \theta_0$, $\theta_0 = \pi/33, \pi/27, \pi/21, \pi/15, \pi/9$, and $\pi/3$

of the differences (5.53) and (5.54), where, by (5.45) and (5.46),

$$f(1, \theta_0) = \frac{6 \sin \theta_0 + 7 \sin 2\theta_0 - \sin 4\theta_0}{\sin^3 \theta_0} \quad \text{and} \quad g(1, \theta_0) = \frac{\pi - \theta_0}{6 \sin^3 \theta_0}. \quad (5.56)$$

Since $E > E_W > E_S$, I and J respectively represent scaled measures of the absolute errors incurred in approximating E by E_W and E_W by E_S .

In Fig. 10, we provide plots of I versus $w/2\ell$ for six representative values of θ_0 . For each choice of θ_0 , we take $10^{-2} \leq w/2\ell < \sin \theta_0$. In accord with the analytical bounds established in Sect. 5.1.5, we see that I increases monotonically with $w/2\ell$ for each choice of θ_0 and clearly exhibits the asymptotic characteristics dictated by (5.47) and (5.49) for $w \ll \ell$ and $w/2\ell \rightarrow \sin \theta_0$. For the values of θ_0 considered, the growth rate of I deviates from the cubic scaling that applies for $w \ll \ell$ at a value of $w/2\ell$ no greater than 3.0×10^{-2} and above that threshold it grows very rapidly with $w/2\ell$. It is also evident that, for each fixed value of $w/2\ell$, the value of I increases monotonically as the angle θ_0 decreases. This property of I is entirely consistent with intuitive expectations: For fixed values of ℓ and θ_0 , consider increasing w from a value small relative to ℓ toward its supremum $2\ell \sin \theta_0$. This should be accompanied by the emergence of a zone in which the splay between the referential rulings increases continuously as w increases. Any such increase would be accompanied by a concomitant concentration of curvature and, thus, bending energy. Alternatively, if ℓ and w are fixed, rotating the left-hand edge of $\mathcal{S}$ about the left-hand endpoint of the midline of $\mathcal{S}$

Fig. 11 Plots of the scaled absolute error $J = 2(E_W - E_S)/\mu g(\theta_0, 1) \cot^2 \varphi$ for a rectangular material strip $\mathcal{D}$ of length ℓ and width w isometrically deformed to coincide with a sector $\mathcal{S}$ of a right circular conical surface $\mathcal{K}$ for $0 < w/2\ell < \sin \theta_0$, $\theta_0 = \pi/33, \pi/27, \pi/21, \pi/15, \pi/9$, and $\pi/3$

would increase θ_0 and the opposite effect, a decrease of the bending energy, would thereby occur.

The developments leading to the expression (5.2) for the energy E stored in bending the rectangular material strip $\mathcal{D}$ into the conical ribbon $\mathcal{S}$ coincident with a portion of the right circular conical surface $\mathcal{K}$ do not rest on assuming that the half-width-to-length aspect ratio $w/2\ell$ of $\mathcal{D}$ is small. Indeed, even granted (5.51), the conditions (4.18) require only that $w < 2\ell \tan \theta_0$ for each $0 < \theta_0 < \pi/2$, from which we see that $w/2\ell$ may take any positive value. Since Sadowsky's [5, 6] derivation is based on the presumption that $w \ll \ell$, it would in contrast be decidedly unreasonable to expect that the expression (5.18) for E_S should accurately estimate E for finite values of $w/2\ell$. It is, however, interesting to explore how accurately E_S approximates Wunderlich's [3, 4] functional E_W . In Fig. 11, we provide plots of J versus $w/2\ell$. To facilitate comparisons, these plots are for the same values of θ_0 used in Fig. 10. Although the qualitative behavior of J closely resembles that of I for each choice of θ_0 considered, important quantitative differences are evident. On the qualitative side, since the limit of E_S is finite at $w/2\ell \rightarrow \sin \theta_0$ for all θ_0 satisfying (4.17), the transition between the cubic scaling that prevails for $w/2\ell \ll 1$ and the asymptotic divergence that occurs for $w/2\ell \rightarrow \sin \theta_0$ is more rapid than that exhibited by I . On the quantitative side, for values of $w/2\ell$ small enough to ensure that both I and J scale with the cube of $w/2\ell$ the curves for J are shifted downward relative to their counterparts for I . From the values of I and J at $w/2\ell = 1.0 \times 10^{-2}$ listed in Table 1, this shift amounts to a factor of approximately $2/3$. For sufficiently small half-width-to-length aspect ratios, the absolute error incurred by us-

Table 1 Values of the scaled relative errors I and J defined in (5.55) at $w/2\ell = 1.0 \times 10^{-2}$ and $w/2\ell = 3.5 \times 10^{-2}$ and for $\theta_0 = \pi/33, \theta_0 = \pi/27, \pi/21, \pi/15, \pi/9$, and $\pi/3$

θ_0	$w/2\ell = 1.0 \times 10^{-2}$		$w/2\ell = 3.5 \times 10^{-2}$	
	$I (\times 10^6)$	$J (\times 10^7)$	$I (\times 10^3)$	$J (\times 10^5)$
$\pi/33$	1.01003	6.71129	6.09847	3.11603
$\pi/27$	1.00669	6.69650	4.89521	3.02511
$\pi/21$	1.00403	6.68473	3.78157	2.95680
$\pi/15$	1.00205	6.67594	2.73486	2.90794
$\pi/9$	1.00074	6.67009	1.74625	2.87643
$\pi/3$	1.00012	6.66720	0.78801	2.86114

ing E_S to approximate E_W is therefore slightly smaller than that incurred by using E_W to approximate E .

A more significant quantitative distinction between I and J is apparent from the insets of Figs. 10 and Fig. 11, from which we see that, for each value of θ_0 considered, I begins to deviate from the cubic scaling (5.47) at a lower value of $w/2\ell$ than that at which J begins to deviate from (5.44). In particular, whereas the values of I for the given choices of θ_0 at the left-hand endpoint of the inset interval span nearly an order of magnitude, the corresponding values of J are graphically indistinguishable. The values of I and J at $w/2\ell = 3.5 \times 10^{-2}$ appear in Table 1. For each choice of θ_0 , the threshold of $w/2\ell$ above which E_S fails to afford an accurate approximation to E_W thus exceeds the threshold of $w/2\ell$ which fails to afford an accurate approximation to E .

Although the foregoing observations are specific to the particular problem of isometrically bending a rectangular material strip $\mathcal{D}$ of length ℓ and width w to coincide with a conical ribbon, they demonstrate that within the context of that problem Wunderlich's [3, 4] functional E_W does not provide an accurate approximation to the bending energy E unless the half-width-to-length ratio $w/2\ell$ is surprisingly small. Moreover, if $w/2\ell$ is sufficiently small to ensure that E_W provides an accurate approximation to E , our findings show that Sadowsky's [5, 6] functional E_S provides an accurate approximation to E_W and, thus, a reasonably accurate approximation to E . With reference to the discussion in Sects. 4.3.3 and 4.3.4, this difficulty stems from a failure to ensure a complete covering of the reference region. Only for rectangular material strips of sufficiently small half-width-to-length aspects ratios is it possible to neglect the presence of unruléd corners of the strip without compromising accuracy.

5.2 Example 2: Bending Energy of a Rectangular Strip Isometrically Deformed to a Helical Ribbon Coincident with a Portion of a Cylindrical Surface

For perspective, we now return to the example of a rectangular material strip $\mathcal{D}$ isometrically deformed into a helical ribbon $\mathcal{S}$ coincident with a portion $\mathcal{S}$ of a right circular cylindrical surface $\mathcal{Y}$ of radius r_0 discussed in Sect. 4.2. For that example, it is again most convenient to parametrize the bending energy E using the rectilinear coordinates (x_1, x_2) introduced in (4.1). Since the mean curvature of $\mathcal{Y}$, and, thus, $\mathcal{S}$, is uniform and equal to $r_0/2$, we see with reference to (4.1), that

$$E = \frac{\mu}{2r_0^2} \int_{-w/2}^{w/2} \int_0^\ell dx_1 dx_2 = \frac{\mu \ell w}{2r_0^2}. \quad (5.57)$$

Fig. 12 Covering of the coordinate pairs (α, β) which represent the collection $\tilde{\mathcal{P}}$ defined in (4.8) for two representative values of θ_0 . Triangular-like regions of $\mathcal{S}$ above the midline of $\mathcal{S}$ are not covered and triangular-like regions on $\mathcal{Y}$ below the midline of $\mathcal{S}$, not part of $\mathcal{S}$, are covered. The areas of these regions and their associated bending energies are equal

Using the transformation (4.4) between rectilinear coordinates (x_1, x_2) and curvilinear coordinate pairs (α, β) , recognizing that the Jacobian of that transformation is $\sin \theta_0$ for all α and β , and bearing in mind that the collection of (α, β) pairs needed to describe each material point $\mathbf{x}$ in $\mathcal{D}$ is given by $\mathcal{P}$ defined in (4.7), we may of course represent E in the alternative form

$$E = \frac{\mu \sin \theta_0}{2r_0^2} \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_{-\beta \cos \theta_0}^{\ell - \beta \cos \theta_0} d\alpha d\beta. \quad (5.58)$$

Evaluating the integrals on the right-hand side of (5.58), we immediately recover (5.57).

If, instead of integrating over all (α, β) pairs in $\mathcal{P}$, we integrate over the pairs in $\tilde{\mathcal{P}}$ defined in (4.8), we arrive at Wunderlich's [3, 4] functional

$$E_W = \frac{\mu \sin \theta_0}{2r_0^2} \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_0^\ell d\alpha d\beta. \quad (5.59)$$

As with (5.58), we may evaluate (5.59) to once again recover (5.57). Thus, in contrast to what occurs for the isometric deformation of a rectangular material strip to a conical ribbon, Wunderlich's [3, 4] functional determines the correct bending energy. This occurs for the simple reason that the nonvanishing principal curvature of the helical ribbon is uniform on each generator of the helical ribbon $\mathcal{S}$. In Fig. 12 we show the portions of $\mathcal{Y}$ that are determined by applying the mapping $\tilde{\mathbf{y}}_0(\alpha) + \beta \mathbf{t}_3$, with $\mathbf{y}_0$ defined in (4.5), to $\tilde{\mathcal{P}}$ for two representative values of θ_0 . Although doing so leaves two corners of $\mathcal{S}$ uncovered, these are compensated by two spurious nonmaterial regions each of which has the same area as the missing corners. Since the nonvanishing principal curvature of $\mathcal{Y}$ is uniform, we therefore see that the missing contributions to the bending energy that arise from using $\tilde{\mathcal{P}}$ instead of $\mathcal{P}$ are compensated for perfectly by the spurious regions. If the nonvanishing principal curvature is not uniform along the spatial generators, we conjecture that Wunderlich's [3, 4] functional will always underestimate the corresponding bending energy.

As independent analytical support of the foregoing geometrical argument, we observe that the expression (5.58) for the bending energy E can be written as

$$\begin{aligned}
 E &= \frac{\mu \sin \theta_0}{2r_0^2} \left(\int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_{-\beta \cos \theta_0}^{w \cot \theta_0} d\alpha d\beta + \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} d\alpha d\beta \right. \\
 &\quad \left. + \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_{\ell - w \cot \theta_0}^{\ell - \beta \cos \theta_0/2} d\alpha d\beta \right) \\
 &= \frac{\mu \sin \theta_0}{2r_0^2} \left(\int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_{-\beta \cos \theta_0}^0 d\alpha d\beta + \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_0^\ell d\alpha d\beta \right. \\
 &\quad \left. + \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_\ell^{\ell - \beta \cos \theta_0/2} d\alpha d\beta \right) \\
 &= \frac{\mu \sin \theta_0}{2r_0^2} \int_{-w \csc \theta_0/2}^{w \csc \theta_0/2} \int_0^\ell d\alpha d\beta \\
 &= E_W.
 \end{aligned} \tag{5.60}$$

5.3 Example 3: Bending Energy Resulting from Isometrically Deforming a Triangular Corner of a Rectangular Material Strip

It is also possible to construct isometric deformations for which the bending energy can be made to be as large as desired but Wunderlich's [3, 4] functional vanishes. As a particular example of this kind, consider a rectangular material strip $\mathcal{D}$ of length ℓ and width w with parallel rulings that are inclined at an angle $0 < \theta_0 < \pi/2$ measured counterclockwise from the midline of $\mathcal{D}$, as illustrated in Fig. 2. As we have observed previously, unless the referential directrix $\mathcal{C}_0$ is defined properly, then two triangular corners of $\mathcal{D}$ remain unrulid and no parametrization based on coordinates along the midline (i.e., that portion of the referential directrix $\mathcal{C}_0$ that consists only of material points of $\mathcal{D}$) and the rulings covers $\mathcal{D}$ completely. At the corners of each unrulid triangle, place a cylinder of radius r with its axis parallel to the rulings, as illustrated in the top left of Fig. 13. Next, roll each cylinder toward the hypotenuse of the unrulid triangle, ensuring that its axis remains parallel to the rulings, as illustrated in the sequence in Fig. 13, stopping once its axis lies directly above the hypotenuse of the triangle, as illustrated in the bottom right of Fig. 13. The bending energy of the material strip deformed in this way is proportional to $1/r^2$ and, thus, can be made increasingly large by decreasing the radius r of the cylinders about which the triangles are wrapped. At odds with this, however, Wunderlich's [3, 4] functional vanishes because it does not take into account the material points in the unrulid triangles and, hence, the bending of those triangles. While an analogous statement applies to Sadowsky's [5, 6] functional, that functional is predicated on the assumption that the width-to-length aspect ratio of $\mathcal{D}$ is infinitesimal.

The deformation required to roll up the unrulid corners of $\mathcal{D}$ as described above is not one-to-one and, thus, does not satisfy all conditions set forth in Sect. 2.1. Alternatively, it is possible to define a deformation in which the unrulid corners of $\mathcal{D}$ are rolled into spirals that can be tightened to drive up the bending energy in a manner analogous to that achieved by decreasing the radius r of the rods involved in the deformation described above.

Fig. 13 A flat rectangular material strip shown in perspective from above with midline identified as the intersection $C_* = C_0 \cap \mathcal{D}$ of its referential directrix C_0 and its reference configuration $\mathcal{D}$, and rulings at the angle θ_0 . Also shown are two small cylinders, parallel to the rulings, about which the triangular corners of the strip in their reference state (*top left*) are wrapped as shown in the sequence and, thus, isometrically deformed. The central portion of the strip which contains all of the rulings remains flat and undistorted

6 Applicability of Kirchhoff Rod Theory to the Modeling of Strip-Like Material Surfaces

It is well known that the shape of a Kirchhoff rod endowed with a material frame $\{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\}$, as introduced in Sect. 3, is completely determined by its midline and the orientations of its cross-sections, which are necessarily perpendicular to the directrix with tangent $\mathbf{d}_3$, and that this geometrical information is completely encoded in the Darboux vector $\boldsymbol{\omega}$ defined such that $\mathbf{d}'_i = \boldsymbol{\omega} \times \mathbf{d}_i$, $i = 1, 2, 3$. Although the directrix of a Kirchhoff rod may bend and the cross sections of the rod may rotate about $\mathbf{d}_3$, those cross sections are by hypothesis rigid and thus cannot change shape.¹⁹ In the present context, the cross sections of a sheet-like object with midsurface $\mathcal{S}$ modeled as a rod with spatial material frame $\{\mathbf{d}_1, \mathbf{d}_2, \mathbf{d}_3\}$ should be rectangular with the sides parallel to $\mathbf{d}_1$ being long (i.e., wide) in comparison to those parallel to $\mathbf{d}_2$ (i.e., through-thickness). Granted the interpretations of the components of the Darboux vector of such a rod stated and supplied immediately after (2.23), it follows from (2.30)₂ that the cross sections of a rod remain rigid only in the presence of the highly restrictive requirement $\kappa_g = 0$, which, by (2.32) is met only if the referential directrix $\mathcal{C}_0$ of $\mathcal{D}$ is straight. Otherwise, the rectangular cross sections bend about $\mathbf{d}_1$ and consequently do not remain rigid. This effect is evident in the sequence of deformations shown in Fig. 13. There, the short ends of the material surface, which are at the ends of the material portion of the referential directrix, are material lines and would represent the terminal cross sections of a rod-based description. To describe the referential material surface depicted in the top left of Fig. 13 as a Kirchhoff rod, it would be necessary to ensure that those terminal cross sections be rigid and, thus, deform only to the extent that they would only be allowed to rotate about the spatial directrix. This would be overly restrictive for any strip-like material surface and in particular would not allow for a deformation of the kind depicted in, say, the sequence shown in Fig. 13. This would drastically undermine any basis for modeling a sheet-like object as a Kirchhoff rod unless, of course, the conventional slenderness hypothesis underlying all rod theories known to the present authors were also in place. In the current setting, that hypothesis amounts to restricting attention to situations with rulings of characteristic length comparable to the through-thickness dimension of the sheet-like object. Although such a hypothesis is met by certain polymer and biopolymer filaments, it is only relevant to a small and not particularly significant class of problems involving thin unstretchable media like paper.

Dias and Audoly [2] place considerable emphasis on a purported connection between their approach to the description of strip-like material surfaces and the Kirchhoff theory of rods. In view of the foregoing discussion, such a connection is possible only if the width and the through-thickness of the ribbon are comparable. Since the through thickness of a ribbon or sheet is assumed to be negligible, the approach of Dias and Audoly [2] is more aptly connected to the theory of framed curves, which, as Giusteri and Fried [14] explain, can be viewed as the specialization of Kirchhoff rod theory that arises on restricting attention to rods of infinitesimally vanishing cross-sectional thickness.

¹⁹A geometrically exact rod model that carefully incorporates in-plane cross sectional deformation was recently developed by Kumar and Mukherjee [11]. Previously, Hodges [12] proposed a constitutively based strategy to account for cross sectional deformation and Gould and Burton [13] developed a rod theory in which each cross section is itself viewed as a rod.

7 Discussion and Summary

We have emphasized that in characterizing the representation of all isometric deformations of a flat two-dimensional material region into a curved surface in three dimensional Euclidean space it is important that the referential directrix and rulings, and the corresponding deformed directrix and generatrices, provide for the establishment of material curvilinear coordinates which completely cover the reference and distorted surfaces. Otherwise, the deformation of parts of the material surface is undefined. Also, we have discussed, by example, the importance of distinguishing between the differential geometric notion of an isometry between two or more two-dimensional surfaces embedded in three dimensional Euclidean space and the continuum mechanical notion of the isometric deformation of a single material surface.

The covering requirement noted above has not been fully appreciated in publications which deal with the isometric deformation of ribbons and, as a result, the oversight has led to misunderstandings and questionable claims. We have tracked the details of this issue in two examples of a rectangular flat material strip that is deformed into (i) a portion of a circular cylindrical surface, and (ii) a portion of a circular conical surface. We have observed that this oversight also is prevalent in the important one-dimensional reductions of Sadowsky [5, 6] and Wunderlich [3, 4] for representing the bending energy of a flat undistorted material surface that is isometrically deformed into a bent shape.

We proceeded to determine, in complete detail, the bending energy E of a rectangular material strip $\mathcal{D}$ of length ℓ and width w that is isometrically deformed to coincide with a portion $\mathcal{S}$ of a right circular conical surface $\mathcal{K}$. Further, we compared the results of that calculation to energies obtained from the corresponding one-dimensional energy functionals E_W and E_S of Wunderlich [3, 4] and Sadowsky [5, 6], finding that $E_S < E_W < E$ for all relevant choices of ℓ , w , cone apex angle 2φ , and certain parameters associated with $\mathcal{K}$ and the placement of the bent strip on $\mathcal{K}$. The discrepancy between E_S and E_W arises simply because Sadowsky's [5, 6] functional is supposed to apply only to situations in which the half-width-to-length ratio $w/2\ell$ of $\mathcal{D}$ is vanishingly small whereas Wunderlich's [3, 4] functional is designed to apply to situations in which that ratio is finite. The discrepancy between E_W and E arises instead from failing to ensure a surjective correspondence between material points and curvilinear coordinate parameter pairs and is, thus, far more relevant to our concerns regarding the covering requirement.

In some cases the Wunderlich functional underestimates the exact bending energy by a sizable and unacceptable amount. As an extreme example, we considered a rectangular material strip whose midline is identified as the referential directrix, and we marked the strip with parallel rulings at an angle θ_0 satisfying $0 < \theta_0 < \pi/2$ with the midline. In this case there are two triangular regions, one in the upper right corner and one in the lower left corner of the rectangle, corners cut off by the far right and far left rulings which terminate at the ends of the midline that are not covered by the coordinate system generated by the midline and the set of rulings. We isometrically deformed the strip by rolling up the corners on cylinders of equal and sufficiently small radii whose generators are parallel to the rulings, as shown in Fig. 13. The remaining central portion of the strip was untouched and remained flat in its undistorted state. For this deformation, the Wunderlich functional E_W records zero, while the bending energy E of the deformed strip is as large as one pleases depending on the radii of the cylinders over which the corners are wrapped. We concluded that the Wunderlich functional does not account for the covering requirement and, therefore, contains a fundamental and potentially significant deficiency; for many situations it does not well-represent the bending energy. As it turns out, for cases in which the half-width-to-length ratio $w/2\ell$ of $\mathcal{D}$ is sufficiently small, it is only incrementally better than the Sadowsky functional.

In the literature, Kirchhoff rod theory has been suggested as a possible theory for modeling the isometric deformation of material ribbons. In the last short section of this work, we gave reasons why this approach is viable only if the width and the through-thickness dimensions of the ribbon are comparable. An association of Kirchhoff rod theory with the study of the deformation of strip-like surfaces is more aptly connected to the theory of framed curves wherein a Kirchhoff rod is assumed to have infinitesimally vanishing cross-sectional thickness.

Acknowledgements The work of Eliot Fried was supported by the Okinawa Institute of Science and Technology Graduate University with subsidy funding from the Cabinet Office, Government of Japan. Yi-chao Chen thanks the Okinawa Institute of Science and Technology for hospitality and generous support during a sabbatical and subsequent visits. The authors express their gratitude to Michael Grunwald for his help in preparing preliminary versions of the figures.

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